5. THE PERFORMANCE OF FEEDBACK CONTROL SYSTEMS

EVALUATION OF T_P , P.O., T_R , T_s FOR **UNDERDAMPED SECOND-ORDER SYSTEMS**

Let us consider a generalized single loop second order system. The closed-loop output is

$$Y(s) = \frac{\omega_n^2}{s^2 + 2\zeta\omega_n s + \omega_n^2} R(s)$$

$$R(s) = \frac{1}{s}$$

$$\varphi_n^2 + 2\zeta\omega_n s + \omega_n^2 R(s)$$
With a unit step input, we obtain
$$Y(s) = \frac{\omega_n^2}{s(s^2 + 2\zeta\omega_n s + \omega_n^2)} = \frac{\omega_n^2}{s(s + \zeta\omega_n \pm j_n \sqrt{1 - \zeta^2})}$$

$$y(t) = 1 - \frac{1}{\sqrt{1 - \zeta^2}} e^{-\zeta\omega_n t} \sin(\omega_n \sqrt{1 - \zeta^2} t + \theta); \theta = \cos^{-1}\zeta$$

EVALUATION OF T_p

 T_p is found by differentiating y(t) and finding the first zero crossing after t=0. This task is simplified by differentiating in the frequency domain. Assuming zero initial conditions, we get

$$\mathcal{L}^{1}[\dot{\mathbf{y}}(t)] = \mathbf{s}\mathbf{Y}(\mathbf{s}) = \frac{\omega_{n}^{2}}{\mathbf{s}^{2} + 2\zeta\omega_{n}\mathbf{s} + \omega_{n}^{2}} = \frac{\omega_{n}^{2}}{(\mathbf{s} + \zeta\omega_{n})^{2} + \omega_{n}^{2}(1 - \zeta^{2})}$$
$$\mathcal{L}^{1}[\dot{\mathbf{y}}(t)] = \frac{\frac{\omega_{n}}{\sqrt{1 - \zeta^{2}}}\omega_{n}\sqrt{1 - \zeta^{2}}}{(\mathbf{s} + \zeta\omega_{n})^{2} + \omega_{n}^{2}(1 - \zeta^{2})}$$

Therefore,

$$\dot{y}(t) = \frac{\omega_n}{\sqrt{1-\zeta^2}} e^{-\zeta\omega_n t} \sin \omega_n \sqrt{1-\zeta^2} t$$

Setting the derivative equal to zero yields

$$\omega_n \sqrt{1-\zeta^2} t = n\pi$$
; $\rightarrow t = \frac{\pi n}{\omega_n \sqrt{1-\zeta^2}} n = 0, 1, 2, 3, ...$

Each value of *n* yields the time for local maxima or minima. The first peak, which occurs at the peak time, T_p , is found by Letting n = 1:

$$T_p = \frac{\pi}{\omega_n \sqrt{1 - \zeta^2}}$$

Notice that T_p is a function of ζ and ω_n

EVALUATION OF P.O.

The peak response M_{p_t} is found by evaluating y(t) at the peak time T_p .

$$M_{p_t} = 1 - \frac{1}{\sqrt{1 - \zeta^2}} e^{\frac{-\zeta \pi}{\sqrt{1 - \zeta^2}}} \sin(\pi + \theta) = 1 + e^{\frac{-\zeta \pi}{\sqrt{1 - \zeta^2}}}$$

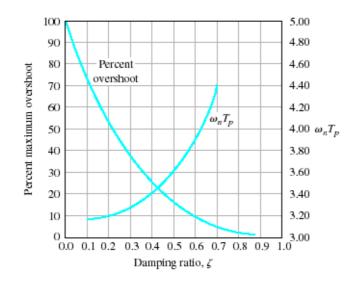
Therefore, The percent overshoot is

P.O. = $e^{\frac{-\zeta \pi}{\sqrt{1-\zeta^2}}}$ 100 Notice that *P.O.* is a function only of ζ

The inverse equation allows one to find ζ given *P*.*O*.

$$\zeta = \frac{-\ln(\frac{P.O.}{100})}{\sqrt{\pi^2 + \ln^2(\frac{P.O.}{100})}}$$

The percent overshoot and the normalized peak time, $\omega_n T_p$, versus the damping ration is shown in the figure.



EVALUATION OF T_s

The settling time is the time required for the response y(t) to reach and stay within a specified absolute percentage δ of the final value. Using a value of $\delta = 2\%$, the settling time occurs when

$$1 + \frac{1}{\sqrt{1 - \zeta^2}} e^{-\zeta \omega_n T_s} = 1.02 \quad \to \frac{1}{\sqrt{1 - \zeta^2}} e^{-\zeta \omega_n T_s} = 0.02 \quad \to T_s = -\frac{\ln(0.02\sqrt{1 - \zeta^2})}{\zeta \omega_n}$$

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the above equation can be approximated to

 $e^{-\zeta\omega_n T_s} < 0.02 \Rightarrow \zeta\omega_n T_s = 4$

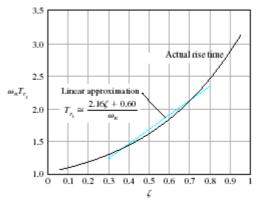
$$T_s = \frac{4}{\zeta \omega_n}$$

Notice that T_s is a function of ζ and ω_n

The settling time for the 2% criterion is approximately four time constants $(\tau = \frac{1}{\zeta \omega_n})$.

EVALUATION OF T_r

An analytical expression for the rise time is difficult to obtain. However, using a computer, we solve the equation of y(t) for the values of $\omega_n t$ that yield y(t) = 0.1 and y(t) = 0.9. Subtracting the two values of $\omega_n t$ yields the normalized rise time as shown in the figure. We can utilize the linear approximation:



Notice that T_r is a function of ζ and ω_n

 $T_r = \frac{2.16\zeta + 0.60}{\omega_n}$

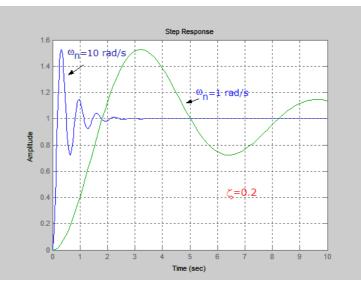
[Accurate for $0.3 \le \zeta \le 0.8$]

Effects of ω_n

For a given ζ , the response is faster for larger ω_n , as shown in the figure

Why is the overshoot the same?

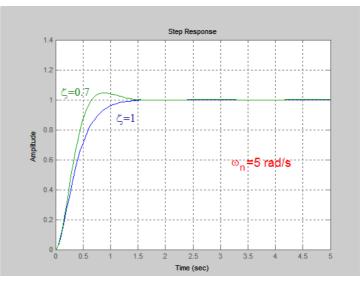
Find T_p , T_r , T_s and *P.O.* from graphs and from equations. Any comments?



Effects of ζ

For a given ω_n , the response is faster for lower ζ , as shown in the figure.

Find T_p , T_r , T_s and P. *O*. from graphs and from equations. Any comments?



SKILL-ASSESSMENT EXERCISE

Find ζ , ω_n , T_p , T_r , T_s , and *P*.O. for a system whose transfer function is

$$G(s) = \frac{361}{s^2 + 16s + 361}$$

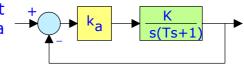
Answers: $\zeta = 0.421, \omega_n = 19 \text{ rad/s}, T_p = 0.182 \text{ s}, T_r = 0.079 \text{ s}, T_s = 0 = .5 \text{ s}, P.O. = 23\%$

Plot the step response using MATLAB and check the accuracy of the results.

Drill Problem [Due on Wednesday 30 October, 2002]

Consider the servo system shown.

1. Design the system to have the fastest response without overshoot, and a settling time of 0.25 s.



2. Simulate the system to verify the design.