

$$\theta_{E/D} = \theta_E = \sum \frac{TL}{JG}$$

$$\therefore \theta_E = \frac{1}{\frac{\pi}{2}(0.015)^4 \times 75 \times 10^9} [-90 \times 0.75 + 30 \times 0.25] = -0.010 \text{ Rad} = 0.010 \text{ Rad.}$$

$$\theta_E \times 100 = \theta_F \times 150 \Rightarrow \theta_F = \frac{150}{100} \theta_E = \frac{150}{100} \times 0.010 = 0.015 \text{ Rad.}$$

$$\theta_{A/F} = \sum \frac{TL}{JG} = \frac{T_{GF} \times L_{GF}}{JG} = \frac{-60 \times 0.25}{\frac{\pi}{2}(0.015)^4 \times 75 \times 10^9} = -0.00252 = 0.00252 \text{ Rad.}$$

$$\theta_A = \theta_F + \theta_{A/F} = 0.015 + 0.00252 = 0.01752 \text{ Rad} = 1.002^\circ \quad \text{Ans.}$$

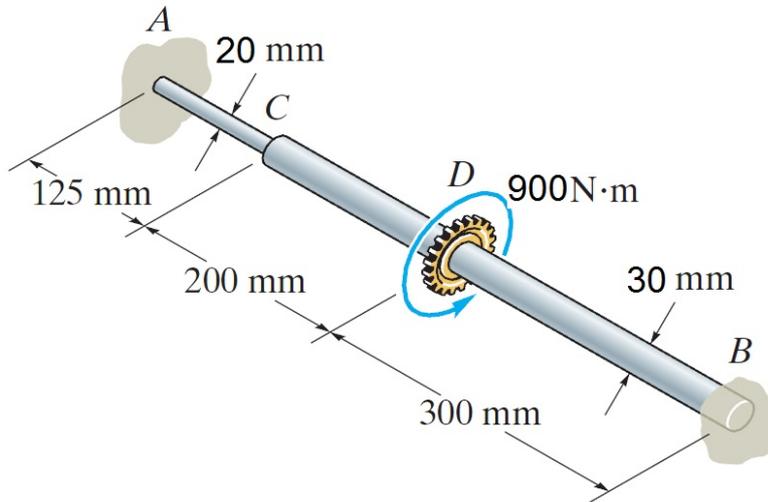
The magnitude of the maximum stress in the whole shaft is as follows:-

$$(\tau_{\max})_{\text{at E}} = \frac{90 \times 0.015}{\frac{\pi}{2}(0.015^4)} = 16.98 \text{ MPa.} \quad \text{Ans}$$

Problem # 2:-

Given Data:

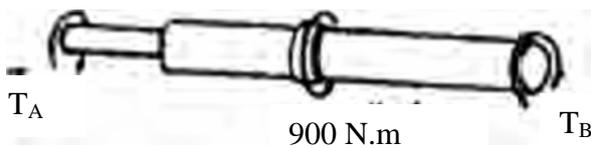
- The shown figure.
- Steel shaft (AC d=20mm and CB d =30mm)
- Fixed at ends A and B.
- G =100 GPa.



Required:

- ❖ The support reactions at A and B.
- ❖ The angle of twist of point D.

Solution:



Equilibrium : $\Rightarrow T_A + T_B - 900 = 0 \dots\dots\dots (1)$

Compatibility condition: $-\phi_{D/A} = \phi_{D/B}$

$$\frac{T_A * 0.125}{\frac{\pi}{2} * (0.01)^4 * G} + \frac{T_A * 0.2}{\frac{\pi}{2} * (0.015)^4 * G} = \frac{T_B * 0.3}{\frac{\pi}{2} * (0.015)^4 * G}$$

□ $2.776T_A = T_B \dots\dots\dots (2)$

From the Equations (1) and (2)

□ $T_A = 238.35 \text{ N.m}$ and $T_B = 661.65 \text{ N.m}$ **Ans**

The angle of twist of point D is as follows:-

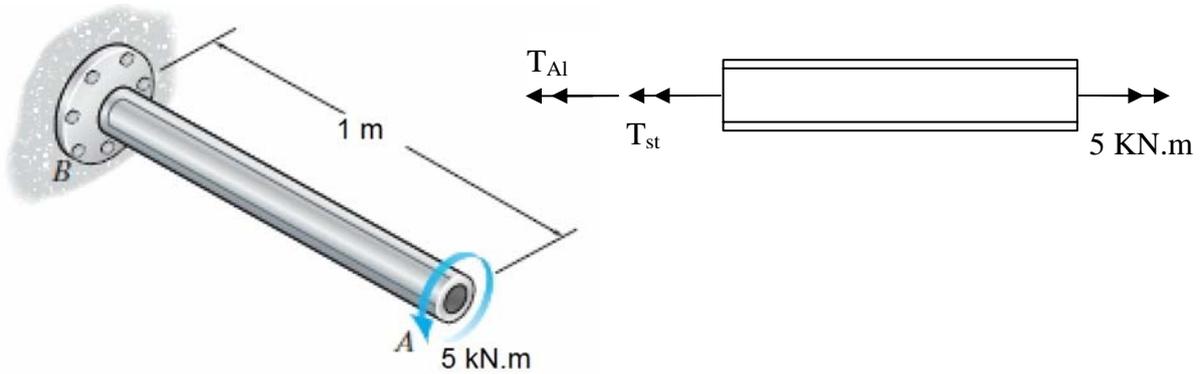
$$\phi_D = \sum \frac{TL}{JG} = \frac{T_A * 0.125}{\frac{\pi}{2} * (0.01)^4 * G} + \frac{T_A * 0.2}{\frac{\pi}{2} * (0.015)^4 * G} = \frac{16450617.3T_A}{\frac{\pi}{2} * G} = \frac{3.921 * 10^9}{\frac{\pi}{2} * 100 * 10^9}$$

$\phi_D = 0.02496 \text{ Rad.}$ **Ans**

Problem # 3:-

Given Data:

- The shown figure (composite shaft).
- Fixed at B.
- Solid circular steel part outer radius = 40mm and G = 100 GPa.
- Aluminum tube (inner radius =40mm and outer radius =70 mm and G=30GPa).



Required:

- ❖ The values of the shear stress along the radius of the shaft.
- ❖ Plotting of the distribution of the shear stress along the radius of the shaft.
- ❖ The angle of the twist of point A.

Solution:

Equilibrium: $-T_{Al} - T_{St} + 5000 = 0 \dots\dots (1)$

Compatibility: $\phi = \phi_{Al} = \phi_{St} \Rightarrow$

$$\frac{T_{Al} * L}{J_{Al} * G_{Al}} = \frac{T_{St} * L}{J_{St} * G_{St}} \Rightarrow \frac{T_{Al} * L}{\frac{\pi}{2} (0.07^4 - 0.04^4) * 30 * 10^9} = \frac{T_{St} * L}{\frac{\pi}{2} (0.04^4) * 100 * 10^9}$$

$$\rightarrow T_{Al} = 2.5137 T_{St} \dots\dots (2) \quad \rightarrow T_{Al} = 3577 \text{ N.m and } T_{St} = 1423 \text{ N.m}$$

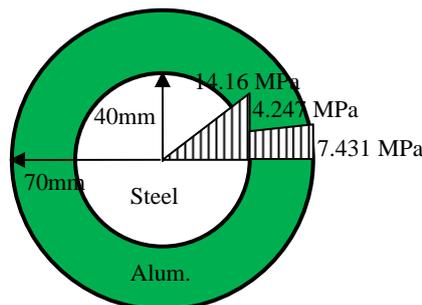
$$(\tau_{St})_{max} = \frac{1423 * 0.04}{\frac{\pi}{2} (0.04^4)} = 14.155 \text{ MPa} \quad \text{Ans.}$$

$$(\tau_{Al})_{min} = \frac{3577 * 0.04}{\frac{\pi}{2} (0.07^4 - 0.04^4)} = 4.247 \text{ MPa} \quad \text{Ans}$$

$$(\tau_{Al})_{max} = \frac{3577 * 0.07}{\frac{\pi}{2} (0.07^4 - 0.04^4)} = 7.431 \text{ MPa} \quad \text{Ans}$$

$$\phi_A = \frac{3577 * 1}{\frac{\pi}{2} (0.07^4 - 0.04^4) * 30 * 10^9} = 0.00354 \text{ Rad.} \quad \text{Ans}$$

The plot of the distribution of the shear stress along the radius of the shaft as follows:-

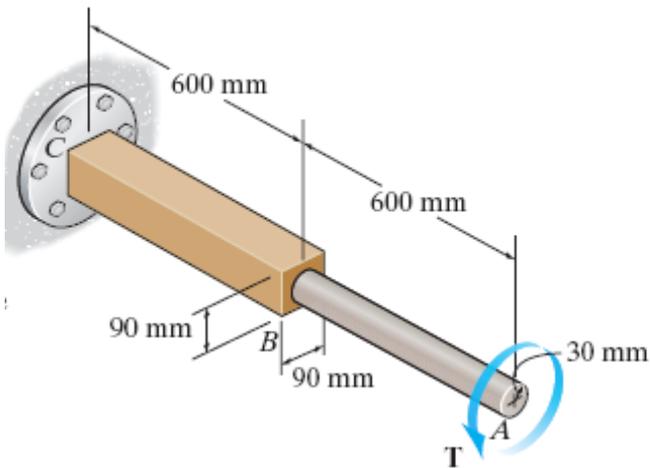


Shear stress distribution along the radius of the shaft.

Problem # 4 (Problem 5-101 Page 230 in the text book with the revised data):-

Given Data:

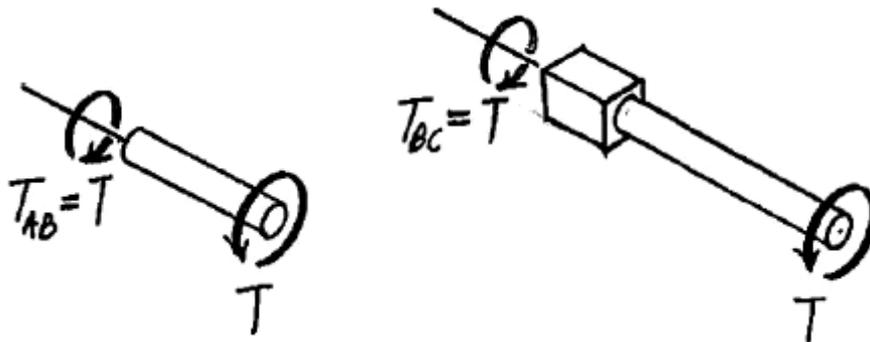
- The shown figure.
- Material of A-36.
- Allowable shear stress = 50 MPa.
- Allowable angle of twist at end A = 0.03 rad.
- The shaft is fixed at end C.



Required:

- ❖ The maximum allowable torque T that can be applied at end A.

Solution:



Allowable Shear Stress: For segment AB ,

$$\tau_{\text{allow}} = \frac{T_{ABC}}{J}, \quad 50 \times 10^6 = \frac{T(0.03)}{\frac{\pi}{2}(0.03^4)}$$

$$T = 2120.57 \text{ N.m} \quad \dots (1)$$

For segment BC ,

$$\tau_{\text{allow}} = \frac{4.81T_{BC}}{a^3}, \quad 50 \times 10^6 = \frac{4.81T}{(0.09)^3}$$

$$T = 7577.96 \text{ N.m} \quad \dots\dots(2)$$

Angle of Twist:

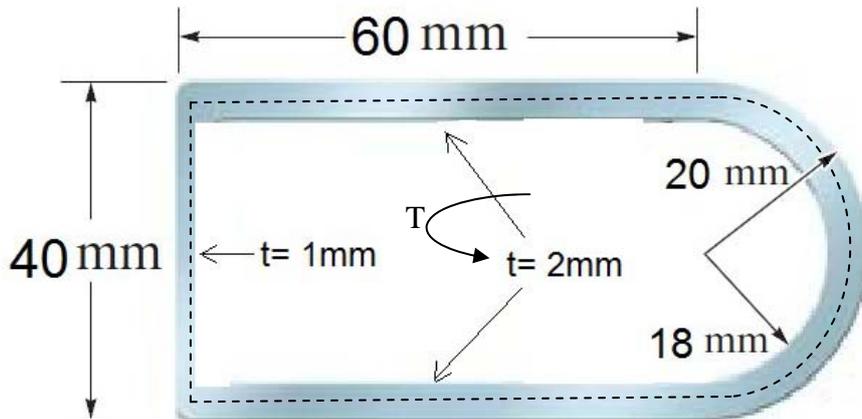
$$\phi_A = \frac{T_{AB}L_{AB}}{JG} + \frac{7.10T_{BC}L_{BC}}{a^4G}$$
$$0.03 = \frac{T(0.6)}{\frac{\pi}{2}(0.03^4)(75)(10^9)} + \frac{7.10T(0.6)}{(0.09)^4(75)(10^9)}$$

$T = 4193.86 \text{ N.m} \dots\dots(3)$, *from the three values* \square **the minimum value is the Allowable $T = 2120.57 \text{ N.m}$** **Ans.**

Problem # 5:-

Given Data:

- The shown figure.
- Subjected torque T is 10 N.m
- G=100 GPa.
- Shaft Length =800 mm.



Required:

- ❖ The magnitude and location of the maximum shear stress in the whole shaft.
- ❖ The angle of twist of the shaft end.

Solution:

For semicircle part, $r = \frac{20+18}{2} = 19 \text{ mm} = 0.019 \text{ m}$.

$$\Rightarrow A_m = \frac{\pi}{2} * 0.019^2 + (0.038 * 0.0595) = 0.00283 \text{ m}^2$$

$$\left(\oint ds \right)_A = 0.038 \text{ m}$$

$$\left(\oint ds \right)_B = \pi * 0.019 + 2 * 0.0595 = 0.1787 \text{ m}$$

$$\Rightarrow \tau_{max} = \tau_A = \frac{T}{2t_{min}A_m} = \frac{10}{2 * 0.001 * 0.00283} = 1.767 \text{ MPa} \quad \text{Ans}$$

$$\Rightarrow \tau_B = \frac{T}{2tA_m} = \frac{10}{2 * 0.002 * 0.00283} = 0.8834 \text{ MPa} \quad \text{Ans}$$

□ The maximum stress at point A in the shown figure and equal to 1.767 MPa. Ans

Angle of twist:-

$$\phi = \frac{TL}{4A_m^2 G} \oint \frac{ds}{t} = \frac{10 * 0.8}{4 * 0.00283^2 * 100 * 10^9} \left(\frac{0.038}{0.001} + \frac{0.1787}{0.002} \right)$$

$$= 0.00318 \text{ Rad} \quad \text{Ans}$$

