

**KING FAHD UNIVERSITY OF PETROLEUM & MINERALS  
ELECTRICAL ENGINEERING DEPARTMENT**

EE380

CONTROL ENGINEERING

081

November 26, 2008

Time: 5:15-6:45 PM

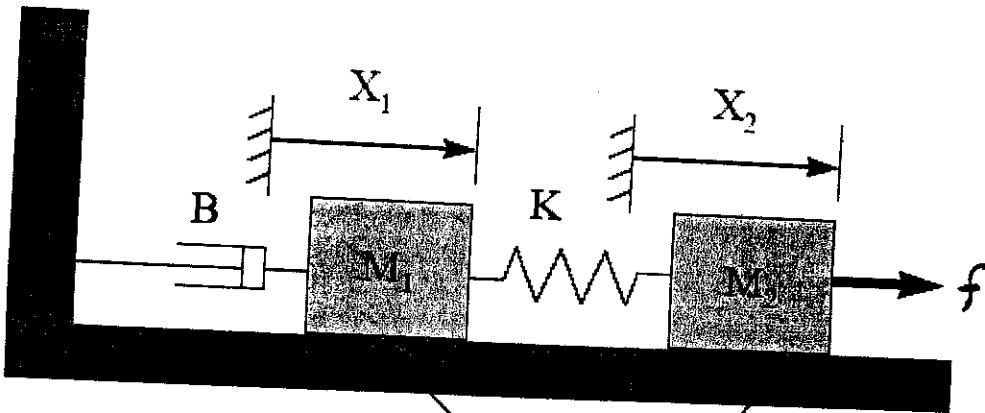
**[MAJOR EXAM # 1]**

Instructor: Dr. Jamil M. Bakhshwain

Name:	<u>Key Solution</u>
ID #:	
Section	

PROBLEM #	SCORE	MAXIMUM
1		25
2		25
3		20
4		30
<b>TOTAL</b>		<b>100</b>

Problem#1: Let  $Z = [x_1 \ x_2 \ \dot{x}_1 \ \dot{x}_2]$  be the state vector. Find the state variable equations and the transfer function  $X_1(s)/F(s)$  for the mechanical system.



$$\textcircled{1} \quad M_2 \ddot{x}_2 + K(x_2 - x_1) = f \quad \text{frictionless}$$

$$\textcircled{2} \quad M_1 \ddot{x}_1 + B \dot{x}_1 + K(x_1 - x_2) = 0$$

$$\dot{z}_1 = z_3$$

$$\dot{z}_2 = z_4$$

$$\dot{z}_3 = \ddot{x}_1 = -\frac{K}{M_1} z_1 + \frac{K}{M_1} z_2 - \frac{B}{M_1} z_3$$

$$\dot{z}_4 = \ddot{x}_2 = \frac{K}{M_2} z_1 - \frac{K}{M_2} z_2 + \frac{1}{M_2} f$$

$$Y = X_1 = z_1 = [1 \ 0 \ 0 \ 0] Z$$

$$\left. \begin{array}{l} \dot{z}_1 = z_3 \\ \dot{z}_2 = z_4 \\ \dot{z}_3 = \ddot{x}_1 = -\frac{K}{M_1} z_1 + \frac{K}{M_1} z_2 - \frac{B}{M_1} z_3 \\ \dot{z}_4 = \ddot{x}_2 = \frac{K}{M_2} z_1 - \frac{K}{M_2} z_2 + \frac{1}{M_2} f \end{array} \right\} \Rightarrow \dot{Z} = \left[ \begin{array}{cccc|cc} 0 & 0 & 0 & 0 & 1 & 0 \\ 0 & 0 & 0 & 0 & 0 & 1 \\ -\frac{K}{M_1} & \frac{K}{M_1} & 0 & 0 & -\frac{B}{M_1} & 0 \\ \frac{K}{M_2} & -\frac{K}{M_2} & 0 & 0 & 0 & 0 \end{array} \right] Z + \left[ \begin{array}{c} 0 \\ 0 \\ 0 \\ \frac{1}{M_2} \end{array} \right] U$$

$$Y = [1 \ 0 \ 0 \ 0] Z$$

$$\text{Taking L.T. of Eqn } \textcircled{1} \Rightarrow (M_2 s^2 + K) X_2(s) - K X_1(s) = F(s) \quad *$$

$$\text{ " " " " } \textcircled{2} \Rightarrow (M_1 s^2 + B s + K) X_1(s) = K X_2(s) \quad **$$

$$\text{From } (**) \quad X_2(s) = \frac{1}{K} (M_1 s^2 + B s + K) X_1(s)$$

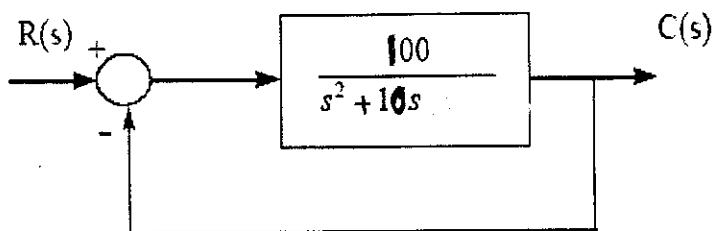
$$\text{Substitute into } (*) \Rightarrow [(M_2 s^2 + K)(M_1 s^2 + B s + K) \frac{1}{K} - K] X_1(s) = F(s)$$

$$\therefore [(M_2 s^2 + K)(M_1 s^2 + B s + K) - K^2] X_1(s) = K F(s)$$

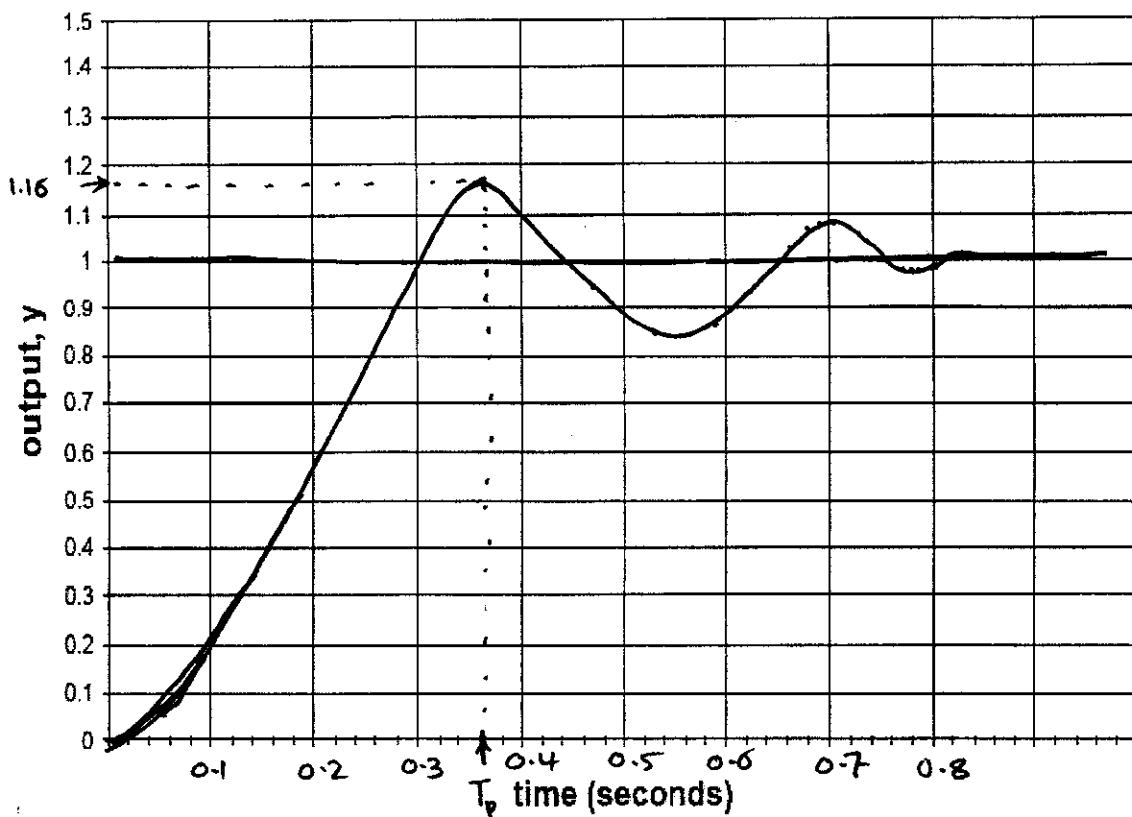
$$\therefore \frac{X_1(s)}{F(s)} = \frac{K}{s[M_1 M_2 s^3 + M_2 B s^2 + K(M_1 + M_2)s + BK]}$$

Problem #2:

A unity feedback control system is shown below:



- find the natural frequency, damping ratio, and damped natural frequency of the closed loop system
- determine the %OS,  $T_p$  and  $T_s$  for a step input to the closed loop system
- sketch the unit step response of the closed loop system on the graph below



$$\frac{C(s)}{R(s)} = \frac{100}{s^2 + 10s + 100} \Rightarrow \boxed{\omega_n = 10}, \quad 2\zeta\omega_n = 10 \Rightarrow \boxed{\zeta = 0.5}$$

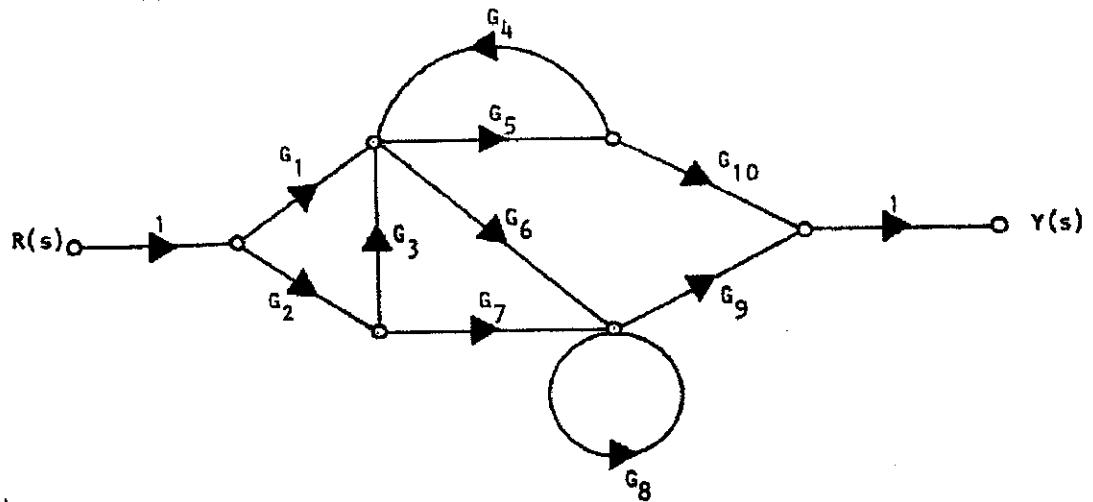
$$\omega_d = \omega_n \sqrt{1 - \zeta^2} = 5\sqrt{3} = B, \quad \boxed{\alpha = \zeta\omega_n = 5}$$

$$P.O. = e^{-\frac{\alpha\pi}{B}} \times 100 = e^{-\frac{\pi}{5\sqrt{3}}} \times 100 = 16.3\%, \quad T_s = \frac{4}{\alpha} = 0.8 \text{ sec}$$

$$T_p = \frac{\pi}{B} = \frac{\pi}{5\sqrt{3}} = 0.36 \text{ sec}$$

Problem #3

Find the transfer function  $Y(s)/R(s)$  using Mason's gain formula



Forward Paths:

$$P_1 = G_1 G_5 G_{10} \quad \Delta_1 = 1 - G_8$$

$$P_2 = G_1 G_6 G_9 \quad \Delta_2 = 1$$

$$P_3 = G_2 G_3 G_5 G_{10} \quad \Delta_3 = 1 - G_8$$

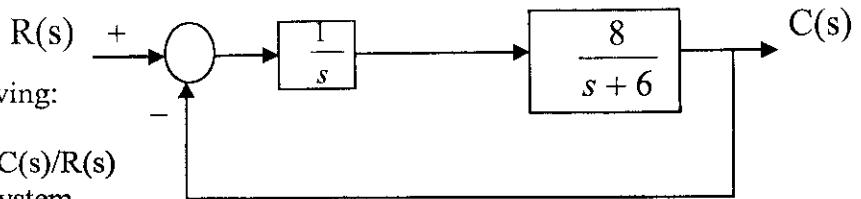
$$P_4 = G_2 G_7 G_9 \quad \Delta_4 = 1 - G_4 G_5$$

$$P_5 = G_2 G_3 G_6 G_9 \quad \Delta_5 = 1$$

$$\Delta = 1 - (G_8 + G_4 G_5) + \underbrace{G_8 G_4 G_5}_{\text{non touching loops}}$$

$$\therefore \frac{Y(s)}{R(s)} = \frac{\sum_{i=1}^5 P_i \Delta_i}{\Delta}$$

Problem#4:



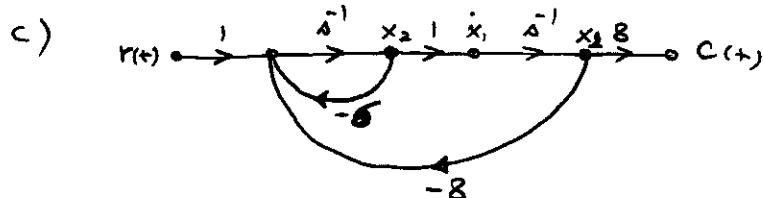
For the system shown find the following:

- The closed loop transfer function  $C(s)/R(s)$
- The differential equation for the system
- The state variables flow graph
- The phase variable state equations
- The state transition matrix  $\Phi(t)$
- $c(t)$  for a unit step input when I.C = zero

$$a) \frac{C(s)}{R(s)} = \frac{8}{s^2 + 6s + 8} \Rightarrow b) \ddot{C}(t) + 6\dot{C}(t) + 8C(t) = 8r(t)$$

$$d) \text{Phase-variable form } \dot{x} = \begin{bmatrix} 0 & 1 \\ -8 & -6 \end{bmatrix}x + \begin{bmatrix} 0 \\ 1 \end{bmatrix}r$$

$$C = Y = [8 \quad 0]x$$



$$e) \Phi(s) = (sI - A)^{-1} = \begin{bmatrix} s & -1 \\ 8 & s+6 \end{bmatrix}^{-1} = \frac{1}{s^2 + 6s + 8} \begin{bmatrix} s+6 & 1 \\ -8 & s \end{bmatrix}$$

$$\Phi(s) = \begin{bmatrix} \frac{2}{s+2} - \frac{1}{s+4} & \frac{1/2}{s+2} - \frac{1/2}{s+4} \\ \frac{-4}{s+2} + \frac{4}{s+4} & \frac{-1}{s+2} + \frac{2}{s+4} \end{bmatrix} \Rightarrow \Phi(t) = \begin{bmatrix} 2e^{-2t} - e^{-4t} & \frac{1}{2}e^{-2t} - \frac{1}{2}e^{-4t} \\ -4e^{-2t} + 4e^{-4t} & -e^{-2t} + 2e^{-4t} \end{bmatrix}$$

$$f) y(t) = C\Phi(s)B \quad U(s) = [8 \ 0] \underbrace{\begin{bmatrix} s+6 & 1 \\ -8 & s \end{bmatrix}}_{\Delta(s)} \begin{bmatrix} 0 \\ 1 \end{bmatrix} \frac{1}{s}$$

$$Y(s) = \frac{8}{s(s^2 + 6s + 8)} = \frac{A}{s} + \frac{B}{s+2} + \frac{C}{s+4} = \frac{1}{s} + \frac{-2}{s+2} + \frac{1}{s+4}$$

$$\therefore C(t) = 1 - 2e^{-2t} + e^{-4t} \quad t \geq 0$$

Alternative Solution: from T.F  $C(s) = \frac{8}{s(s^2 + 6s + 8)}$ , then proceed as above.