

Structural Mechanics I

KEY OF HOME WORK : 4

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KEY TO HOMEWORK #4.

PROBLEM # ① (4-77)!

SOLUTION -

GIVEN -

$$T_1 = -20^\circ \text{F}$$

$$T_2 = 90^\circ \text{F}$$

$$A = 5.10 \text{ in}^2$$

$$L = 40 \text{ ft}$$

$$\alpha_{st} = 6.6 \times 10^{-6} / ^\circ \text{F}$$

$$E_{st} = 29 \times 10^3 \text{ ksi}$$

Required -

$$\delta = ?$$

$$\delta = \alpha_{st} \times \Delta T \times L_{st}$$

$$\Rightarrow \delta = 6.6 \times 10^{-6} \times [90 - (-20)] \times 40 \times 12$$

$$\Rightarrow \delta = \underline{\underline{0.348 \text{ in}}}$$

now finding

$$F_{st} = ? \text{ at } T_3 = 110^\circ \text{F}$$

Compatibility -

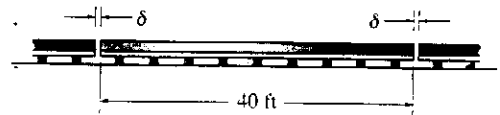
$$\delta = \delta_T - \delta_F$$

$$\Rightarrow \alpha_{st} \cdot \Delta T \cdot L_{st} - \frac{F_{st} \cdot L_{st}}{A_{st} \cdot E_{st}} = 0.348$$

$$\Rightarrow 6.6 \times 10^{-6} \times [110 - (-20)] \times 40 \times 12 - \frac{F_{st} \times 40 \times 12}{5.1 \times 29 \times 10^3} = 0.348$$

$$\Rightarrow F_{st} = \underline{\underline{19.6 \text{ kip}}}$$

4-77. The 40-ft-long A-36 steel rails on a train track are laid with a small gap between them to allow for thermal expansion. Determine the required gap δ so that the rails just touch one another when the temperature is increased from $T_1 = -20^\circ \text{F}$ to $T_2 = 90^\circ \text{F}$. Using this gap, what would be the axial force in the rails if the temperature were to rise to $T_3 = 110^\circ \text{F}$? The cross-sectional area of each rail is 5.10 in^2 .



Prob. 4-77

PROBLEM #2 (4-80)!

Solution!

Given:-

$$T_1 = 25^\circ\text{C}.$$

$$A = 10 \times 15 = 150 \text{ mm}^2$$

$$\alpha_{al} = 24 \times 10^{-6} / ^\circ\text{C}$$

$$E_{al} = 68.9 \times 10^3 \text{ MPa}$$

$$\alpha_{am} = 26 \times 10^{-6} / ^\circ\text{C}$$

$$E_{am} = 44.7 \times 10^3 \text{ MPa}.$$

Compatibility!

$$\delta_{al} + \delta_{am} = 1.5 \text{ mm}.$$

$$\Rightarrow \alpha_{al} \cdot \Delta T \cdot L_{al} + \alpha_{am} \cdot \Delta T \cdot L_{am} = 1.5$$

$$\Rightarrow \Delta T = \frac{1.5}{(\alpha_{al} \cdot L_{al} + \alpha_{am} \cdot L_{am})}$$

$$\Rightarrow \Delta T = \frac{1.5}{(24 \times 10^{-6} \times 600 + 26 \times 10^{-6} \times 400)} = 60.5^\circ\text{C}$$

$$\Rightarrow \Delta T = 60.5^\circ\text{C}$$

$$\Rightarrow T_2 = T_1 + \Delta T = 25 + 60.5$$

$$\Rightarrow \underline{T_2 = 85.5^\circ\text{C}}$$

Now finding.

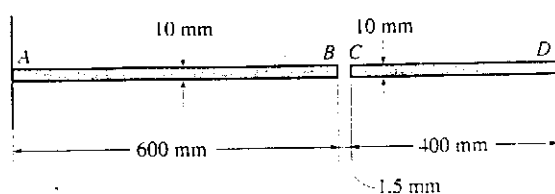
$$F_{al} = ? + F_{am} = ? \text{ at } T_2 = 100^\circ\text{C}.$$

Equilibrium!

$$\rightarrow \sum F_x = 0$$

$$F_{AL} = F_{AM} = \dots \text{--- (1)}$$

*4-80. A thermo gate consists of a 6061-T6-aluminum plate AB and an Am-1004-T61-magnesium plate CD , each having a width of 15 mm and fixed supported at their ends. If the gap between them is 1.5 mm when the temperature is $T_1 = 25^\circ\text{C}$, determine the temperature required to just close the gap. Also, what is the axial force in each plate if the temperature becomes $T_2 = 100^\circ\text{C}$? Assume bending or buckling will not occur.



Prob. 4-80

Compatibility! -

$$(\delta_{al})_T - (\delta_{al})_F = -(\delta_{am})_T + (\delta_{am})_F$$

$$\Rightarrow \alpha_{al} \cdot \Delta T \cdot L_{al} - \frac{F_{al} \times L_{al}}{A_{al} \times E_{al}} = -\alpha_{am} \cdot \Delta T \cdot L_{am} + \frac{F_{am} \cdot L_{am}}{A_{am} \cdot E_{am}}$$

$$\Rightarrow 24 \times 10^{-6} \times (100 - 85.5) \times 600 - \frac{F_{al} \times 600}{150 \times 68.9 \times 10^3} =$$

$$-26 \times 10^{-6} (100 - 85.5) \times 400 + \frac{F_{am} \times 400}{150 \times 44.7 \times 10^3}$$

$$0.209 - 5.806 \times 10^{-5} F_{al} = -0.1508 + 5.966 \times 10^{-5} F_{am} \quad \text{--- (2)}$$

Solving (1) & (2) we get.

$$5.966 \times 10^{-5} F_{am} + 5.806 \times 10^{-5} F_{am} = 0.209 + 0.1508$$

$$\Rightarrow F_{am} = 3056 \text{ N.}$$

$$\therefore F_{am} = 3.05 \text{ kN.}$$

$$\Rightarrow \underline{\underline{F_{am} = F_{al} = 3.05 \text{ kN.}}}$$

PROBLEM #3 (4-85):-

Solution!-

Given!-

$$T_1 = 60^\circ \text{F}$$

$$A = \frac{\pi}{4} (1.25)^2 = 1.227 \text{ in}^2$$

$$\alpha_{al} = 13 \times 10^{-6} / ^\circ \text{F}; E_{al} = 10 \times 10^3 \text{ ksi}$$

$$\alpha_{cu} = 9.4 \times 10^{-6} / ^\circ \text{F}; E_{cu} = 18 \times 10^3 \text{ ksi}$$

compatibility!-

$$\delta_{al} + \delta_{cu} = 0.008$$

$$\Rightarrow \alpha_{al} \cdot \Delta T \cdot L_{al} + \alpha_{cu} \cdot \Delta T \cdot L_{cu} = 0.008$$

$$\Delta T = \frac{0.008}{13 \times 10^{-6} \times 8 + 9.4 \times 10^{-6} \times 4}$$

$$\Rightarrow \Delta T = 56.5^\circ \text{F}$$

$$\therefore T_2 = T_1 + \Delta T = 60 + 56.5 = 116.5^\circ \text{F}$$

$$\therefore T_2 = 116.5^\circ \text{F}$$

Now finding

$$\sigma_{al} = ? \text{ \& } \sigma_{cu} = ?$$

Equilibrium!-

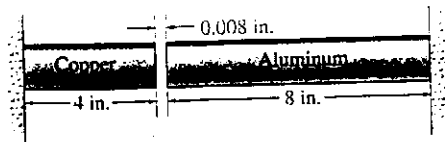
$$\begin{aligned} \uparrow \sum F_x &= 0 \\ F_{al} &= F_{cu} \quad \text{--- (1)} \end{aligned}$$

compatibility!-

$$(\delta_{cu})_T - (\delta_{cu})_F = -(\delta_{al})_T + (\delta_{al})_F$$

$$\Rightarrow \alpha_{cu} \cdot \Delta T \cdot L_{cu} - \frac{F_{cu} \cdot L_{cu}}{A_{cu} \cdot E_{cu}} = -\alpha_{al} \cdot \Delta T \cdot L_{al} + \frac{F_{al} \cdot L_{al}}{A_{al} \cdot E_{al}}$$

4-85. The two circular rod segments, one of aluminum and the other of copper, are fixed to the rigid walls such that there is a gap of 0.008 in. between them when $T_1 = 60^\circ \text{F}$. What larger temperature T_2 is required in order to just close the gap? Each rod has a diameter of 1.25 in., $\alpha_{al} = 13(10^{-6})/^\circ \text{F}$, $E_{al} = 10(10^3) \text{ ksi}$, $\alpha_{cu} = 9.4(10^{-6})/^\circ \text{F}$, $E_{cu} = 18(10^3) \text{ ksi}$. Determine the average normal stress in each rod if $T_2 = 200^\circ \text{F}$.



Probs. 4-85/4-86

$$9.4 \times 10^6 \times (200 - 116.5) \times 4 - \frac{F_{C4} \times 4}{1.227 \times 19 \times 10^3} =$$

$$- 13 \times 10^6 \times (200 - 116.5) \times 8 + \frac{F_{AL} \times 8}{1.227 \times 10 \times 10^3}$$

$$3.1396 \times 10^3 - 1.812 \times 10^{-4} F_{C4} = -8.684 \times 10^{-3} + 6.52 \times 10^{-4} F_{AL}$$

$$6.52 \times 10^{-4} F_{AL} + 1.812 \times 10^{-4} F_{C4} - 0.012 = 0 \quad \text{--- (2)}$$

By solving (1) & (2) we get .

$$F_{AL} = F_{C4} = 14.2 \text{ kip.}$$

$$\therefore \overline{\sigma}_{AL} = \overline{\sigma}_{C4} = \frac{F_{AL} \text{ (or) } F_{C4}}{A_{AL} \text{ (or) } A_{C4}} = \frac{14.2}{1.227}$$

$$\Rightarrow \overline{\sigma}_{AL} = \overline{\sigma}_{C4} = 11.6 \text{ ksi}$$

PROBLEM # 4 (4-87)!

SOLUTION! -

GIVEN! -

A-36 Steel.

$$E = 29 \times 10^3 \text{ ksi.}$$

$$\alpha = 6.6 \times 10^{-6} / ^\circ\text{F.}$$

$$\Delta T = (40 + 15x) ^\circ\text{F.}$$

$x \rightarrow$ in feet.

$$\therefore \text{change in length } \delta L = \delta T = \int_0^L \alpha \Delta T dx$$

$$\therefore \delta T = \alpha \int_0^8 (40 + 15x) dx$$

$$= 6.6 \times 10^{-6} \left[40x + \frac{15x^2}{2} \right]_0^8$$

$$= 6.6 \times 10^{-6} \left[40 \times 8 + \frac{15 \times 8^2}{2} \right]$$

$$\Rightarrow \delta T = 5.28 \times 10^{-3} \text{ in.}$$

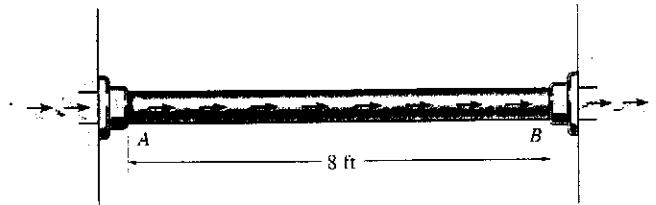
$$\therefore \text{Average Normal Stress} = \sigma_{\text{avg}} = \frac{\delta T \cdot E}{L}$$

$$\Rightarrow \sigma_{\text{avg}} = \frac{5.28 \times 10^{-3} \times 29 \times 10^3 \times 12}{8 \times 12}$$

$$= 19.14 \text{ ksi}$$

$$\Rightarrow \underline{\underline{\sigma_{\text{avg}} = 19.14 \text{ ksi.}}}$$

4-87. The pipe is made of A-36 steel and is connected to the collars at A and B. When the temperature is 60°F , there is no axial load in the pipe. If hot gas traveling through the pipe causes its temperature to rise by $\Delta T = (40 + 15x)^\circ\text{F}$, where x is in feet, determine the average normal stress in the pipe. The inner diameter is 2 in., the wall thickness is 0.15 in.



Probs. 4-87/4-88

PROBLEM # 5 (4-95)! -

4-95. If the allowable normal stress for the bar is $\sigma_{\text{allow}} = 120 \text{ MPa}$, determine the maximum axial force P that can be applied to the bar.

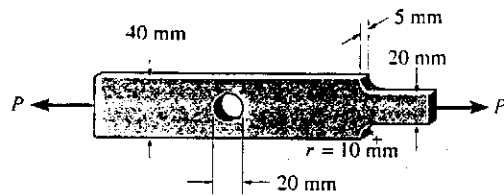
SOLUTION! -

Given! -

$$\sigma_{\text{all}} = 120 \text{ MPa}$$

Required! -

$$P_{\text{max}} = ?$$



Prob. 4-95

First considering the circular hole! -

Finding stress concentration factor k' from Figure 4-25 (text; page 159).

$$\therefore r = \frac{20}{2} = 10 \text{ mm}; \quad w = 40 \text{ mm}$$

$$\therefore \frac{r}{w} = \frac{10}{40} = 0.25$$

$$\Rightarrow k = 2.375 \quad (\text{from Fig. 4-25})$$

$$k = \frac{\sigma_{\text{max}}}{\sigma_{\text{avg}}} \Rightarrow \sigma_{\text{avg}} = \frac{\sigma_{\text{max}}}{k} = \frac{120}{2.375} = 50.5$$

$$\Rightarrow \sigma_{\text{avg}} = \frac{P}{(w-2r)t} = 50.5 \text{ MPa}$$

$$\Rightarrow P = (w-2r)t \times 50.5 = (40-20) \times 5 \times 50.5$$

$$\Rightarrow P = \underline{\underline{5.05 \text{ kN}}}$$

Now considering the fillet! -

Finding k' from Figure 4-24 (text; page 159)

$$\therefore r = 10 \text{ mm}; \quad h = 20 \text{ mm}; \quad w = 40 \text{ mm}$$

$$\therefore \frac{r}{h} = \frac{10}{20} = 0.5; \quad \frac{w}{h} = \frac{40}{20} = 2.0$$

$\therefore K = 1.4$ [from Fig. 4.24).

$$K = \frac{\sigma_{\max}}{\sigma_{\text{avg}}}$$

$$\Rightarrow \sigma_{\text{avg}} = \frac{\sigma_{\max}}{K} = \frac{120}{1.4} = 85.71 \text{ MPa}$$

$$K \sigma_{\text{avg}} = \frac{P}{ht} = 85.71 \text{ MPa}$$

$$\Rightarrow P = ht \times 85.71$$

$$\Rightarrow P = 20 \times 5 \times 85.71$$

$$\Rightarrow P = 8.57 \text{ kN}$$

$\therefore P_{\max}$ is lowest value of above two P's.

$$\therefore \underline{\underline{P_{\max} = 5.05 \text{ kN}}}$$

PROBLEM # 6 (4-100)!-

Solution!-

$$P = \int \sigma dA$$

$\Rightarrow P =$ volume of the stress distribution diagram
Assuming distribution to be triangular.

$\Rightarrow P =$ [whole rectangular volume - triangular volume]

$$\Rightarrow P = [30 \times 40 \times 20 - \frac{1}{2} \times 40 \times 10 \times 20]$$

$$\Rightarrow \underline{P = 20 \text{ kN}}$$

$$\therefore K = \text{stress concentration factor} = \frac{\sigma_{\max}}{\sigma_{\text{avg}}}$$

$$\therefore \sigma_{\text{avg}} = \frac{P}{kt} = \frac{20 \times 10^3}{40 \times 20}$$

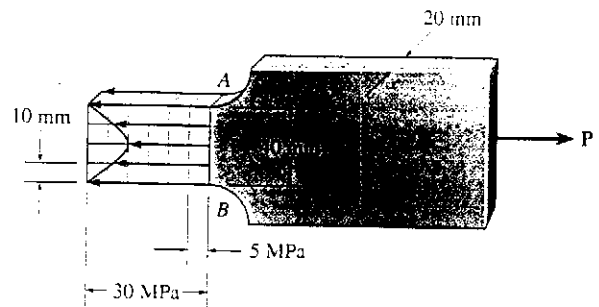
$$\Rightarrow \sigma_{\text{avg}} = 25 \text{ MPa}$$

$$\& \sigma_{\max} = 30 \text{ MPa}$$

$$\Rightarrow K = \frac{\sigma_{\max}}{\sigma_{\text{avg}}} = \frac{30}{25} = 1.2$$

$$\Rightarrow \underline{K = 1.2}$$

*4-100. The resulting stress distribution along the section AB of the bar is shown in the figure. From this distribution, determine the approximate resultant axial force P applied to the bar. Also, what is the stress-concentration factor for this geometry?



Prob. 4-100

PROBLEM # 7(4-101)!

4-101. The A-36 steel plate has a thickness of 12 mm. It has shoulder fillets at B and C, and $\sigma_{\text{allow}} = 150 \text{ MPa}$. Determine the maximum axial load P that it can support. Compute its elongation neglecting the effect of the fillets.

Solution!

Given:-

A-36 steel

$$E = 200 \times 10^3 \text{ MPa}$$

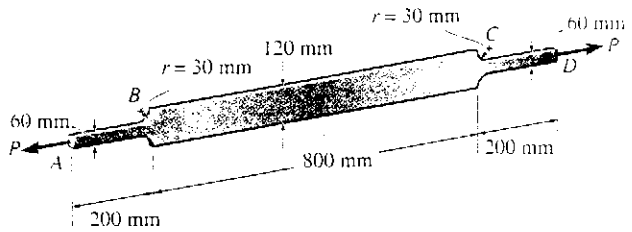
$$t = 12 \text{ mm}$$

$$w = 120 \text{ mm}$$

$$h = 60 \text{ mm}$$

$$r = 30 \text{ mm}$$

$$\sigma_{\text{all}} = 150 \text{ MPa}$$



Prob. 4-101

Required:-

$$P = ?$$

$$\delta_{AD} = ?$$

Computing Maximum axial Load P!

First determining stress concentration factor 'k' from Figure 4-24 (text; Page # 159).

$$\frac{w}{h} = \frac{120}{60} = 2 \quad \& \quad \frac{r}{h} = \frac{30}{60} = 0.5$$

$$\therefore k = 1.4$$

$$\therefore \sigma_{\text{avg}} = \frac{P}{ht} = \frac{P}{60 \times 12} = \frac{P}{720}$$

$$\text{and } k = \frac{\sigma_{\text{max}}}{\sigma_{\text{avg}}} \Rightarrow \sigma_{\text{avg}} = \frac{\sigma_{\text{max}}}{k}$$

$$\therefore \frac{P}{720} = \frac{150}{1.4}$$

$$\Rightarrow P = \underline{\underline{77.14 \text{ kN}}}$$

P.T.O.

Computing Elongation! -

$$\delta_{A/D} = \sum \frac{PL}{AE} = \left(\frac{PL}{AE}\right)_{A/B} + \left(\frac{PL}{AE}\right)_{B/C} + \left(\frac{PL}{AE}\right)_{C/D}$$

$$= \left\{ \frac{77.14 \times 10^3 \times 200}{(60 \times 12)(200 \times 10^3)} \right\} + \left\{ \frac{77.14 \times 10^3 \times 800}{(120 \times 12)(200 \times 10^3)} \right\} + \left\{ \frac{77.14 \times 10^3 \times 200}{(60 \times 12)(200 \times 10^3)} \right\}$$

$$\Rightarrow \delta_{A/D} = 0.4285 \text{ mm}$$

$$\Rightarrow \delta_{A/D} = \underline{\underline{0.429 \text{ mm}}}$$