

7

Examples on Multi-Dimensional States of Stress and Strain (Generalized Hooke's Law)

Example 1:

Given:

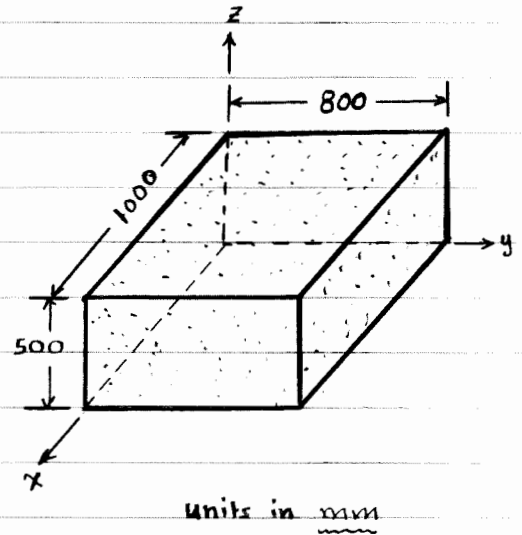
The block shown,
After applying forces
in the x, y, and z directions,
the changes in dimensions are

$$\Delta x = 0.3 \text{ mm}$$

$$\Delta y = -0.16 \text{ mm}$$

$$\Delta z = 0.2 \text{ mm}$$

$$E = 70 \text{ GPa} \quad ; \quad \nu = 0.3$$



Req'd.:

The normal strains and stresses

Soln.:

$$\epsilon_x = \frac{\Delta x}{l_x} = \frac{0.3}{1000} = 300 \underbrace{(10)^{-6}}_{\mu} \text{ mm/mm} \Rightarrow \boxed{\epsilon_x = 300 \mu \text{ mm/mm}}$$

$$\epsilon_y = \frac{\Delta y}{l_y} = \frac{-0.16}{800} \Rightarrow \boxed{\epsilon_y = -200 \mu \text{ mm/mm}}$$

$$\epsilon_z = \frac{\Delta z}{l_z} = \frac{0.2}{500} \Rightarrow \boxed{\epsilon_z = 400 \mu \text{ mm/mm}}$$

$$\sigma_x = \frac{E}{(1+\nu)(1-2\nu)} [(1-\nu)\epsilon_x + \nu(\epsilon_y + \epsilon_z)] = \frac{70 (10)^9}{(1+0.3)(1-0.6)} [(1-0.3)300 + 0.3(-200 + 400)] (10)^{-6}$$

$$= 1.3462 (10)^{11} (270) (10)^{-6} \Rightarrow \boxed{\sigma_x = 36.3 (10)^6 \text{ Pa} = 36.3 \text{ MPa (T)}}$$

$$\sigma_y = \text{''} [(1-\nu)\epsilon_y + \nu(\epsilon_x + \epsilon_z)] \Rightarrow \boxed{\sigma_y = 9.42 \text{ MPa (T)!!}}$$

$$\sigma_z = \text{''} [(1-\nu)\epsilon_z + \nu(\epsilon_x + \epsilon_y)] \Rightarrow \boxed{\sigma_z = 41.7 \text{ MPa (T)}}$$

Example 2:

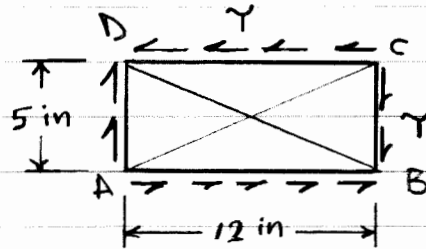
Given:

The plate shown

$\tau = 20 \text{ ksi}$

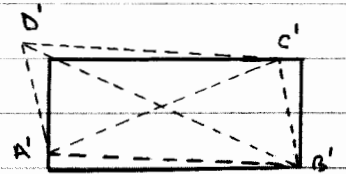
$E = 10000 \text{ ksi}$

$\nu = 0.3$



Req'd.:

- The shearing strains between AB & AD and between AB & BC
- The change in lengths of AC & BD
- The normal strains in the diagonals



Soln.:

$$G = \frac{E}{2(1+\nu)} = \frac{10000}{2(1+0.3)} = 3846.2 \text{ ksi}$$

$$a) \gamma_{DAB} = \frac{\tau}{G} = \frac{-20}{3846} \Rightarrow \gamma_{DAB} = -0.0052 \text{ rad (in/in)}$$

Note that - sign means the angle DAB became $> 90^\circ$ as shown.

$$\gamma_{ABC} = \frac{+20}{3846} \Rightarrow \gamma_{ABC} = 0.0052 \text{ rad (in/in)}$$

← You may drop the units

b) Note that the shearing strain is an angle measured in radian.

$$AC = BD = \sqrt{(5)^2 + (12)^2} = 13 \text{ in}$$

$$\gamma = 0.0052 \text{ rad} = 0.2979^\circ$$

$$\Rightarrow \angle D'A'B' = 90 + 0.2979 = 90.2979^\circ$$

$$\angle A'B'C' = 90 - 0.2979 = 89.7021^\circ$$

$$(\angle B'D')^2 = (\angle A'B')^2 + (\angle A'D')^2 - 2 \angle A'B'(\angle A'D') \cos 90.2979^\circ$$

$$= (12)^2 + (5)^2 - 2(12)(5) \cos 90.2979^\circ = 169.624 \Rightarrow B'D' = 13.0240 \text{ in}$$

Note that $A'B' = AB$, $B'C' = BC$, $C'D' = CD$, $D'A' = DA$ because shearing stress does not change the lengths of the sides of the rectangle.

$$\Delta l_{BD} = 13.024 - 13 \Rightarrow \Delta l_{BD} = 0.024 \text{ in} \Rightarrow \epsilon_{BD} = \frac{\Delta l}{l} = \frac{0.024}{13} \Rightarrow \epsilon_{BD} = 1846 \mu \text{ in/in}$$

$$(\angle A'C')^2 = (12)^2 + (5)^2 - 2(12)(5) \cos 89.7021^\circ \Rightarrow A'C' = 12.9760 \text{ in}$$

$$\Rightarrow \Delta l_{AC} = 12.976 - 13 \Rightarrow \Delta l_{AC} = -0.02402 \Rightarrow \epsilon_{AC} = \frac{-0.024}{13} \Rightarrow \epsilon_{AC} = -1846 \mu \text{ in/in}$$