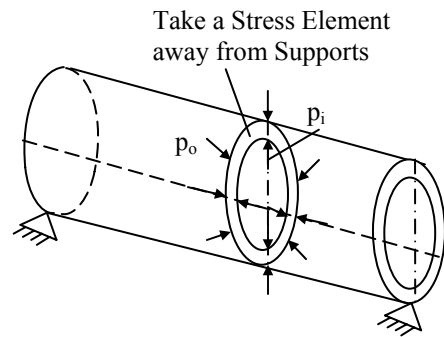
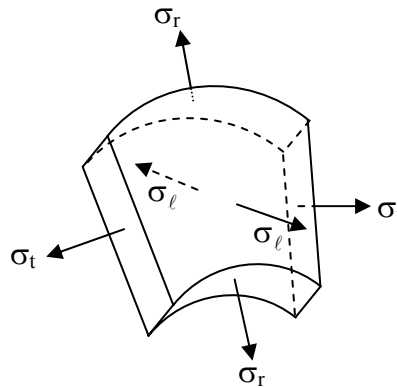


## 4-15 Stresses in Pressurized Cylinders

Cylindrical pressure vessels, hydraulic cylinders, air tanks and pipes carrying pressurized fluids develop tangential, radial and longitudinal (axial) stresses ( $\sigma_t$ ,  $\sigma_r$  and  $\sigma_\ell$ ). A typical pressurized cylinder with inside and outside pressures of  $p_i$  and  $p_o$  is shown below:



The 3-D stress element taken from the wall of the cylinder shown above is enlarged as:



These stresses are given as:

$$\sigma_t = \frac{p_i r_i^2 - p_o r_o^2 - r_i^2 r_o^2 (p_o - p_i) / r^2}{r_o^2 - r_i^2}$$

$$\sigma_r = \frac{p_i r_i^2 - p_o r_o^2 + r_i^2 r_o^2 (p_o - p_i) / r^2}{r_o^2 - r_i^2}$$

$$\sigma_\ell = \frac{p_i r_i^2 - p_o r_o^2}{r_o^2 - r_i^2}$$

In these equations,  $r_i$  and  $r_o$  are the inside and outside radii for the cross-section and  $r$  is the radial distance of the stress element on the cross-section from its center. *When the outside pressure is zero, i.e.  $p_o=0$ ,*

$$\sigma_t = \frac{p_i r_i^2}{r_o^2 - r_i^2} \left( 1 + \frac{r_o^2}{r^2} \right)$$

$$\sigma_r = \frac{p_i r_i^2}{r_o^2 - r_i^2} \left( 1 - \frac{r_o^2}{r^2} \right)$$

$$\sigma_\ell = \frac{p_i r_i^2}{r_o^2 - r_i^2}$$

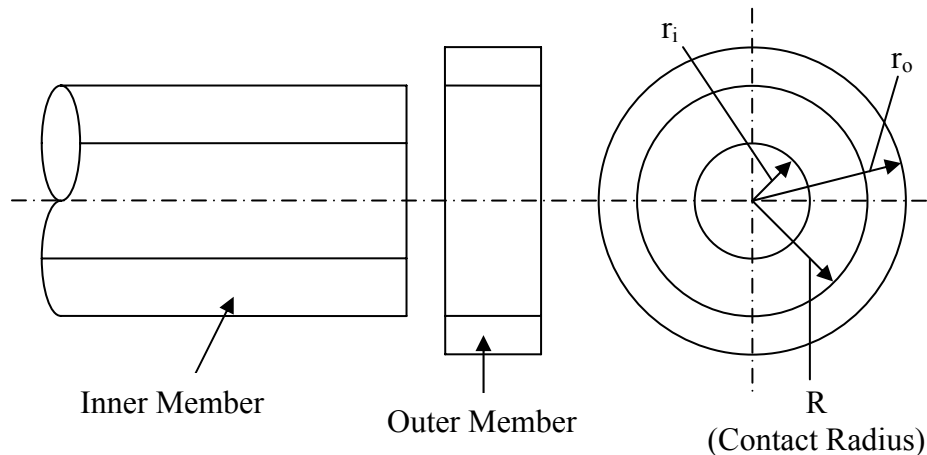
*Thin-Walled Cylinders:*

In the special case when the thickness of the cylinder  $t$  is much less than its average radius  $r$ , i.e.  $r \geq 20t$ , and assuming again that  $p_o=0$ , then the above equations greatly simplify to:

$$(\sigma_t)_{\max} = \frac{p_i (d_i + t)}{2t}, \quad \sigma_r \approx 0 \quad \text{and} \quad \sigma_\ell = \frac{p_i d_i}{4t}.$$

#### 4-17 Press and Shrink Fits

When two cylindrical parts are assembled by shrinking or press fitting one part upon another, a contact pressure  $p$  is developed between the two parts along the contact surface at the radius of  $R$ . An example is the press fitting of a gear onto a shaft. The below figure illustrates the process:



After the assemblage, the inner member shrinks an amount of  $\delta_i$  and the outer member expands  $\delta_o$  in the radial direction. Given the total interference  $\delta = |\delta_o| + |\delta_i|$ , we can find the contact pressure  $p$  from the following relation:

$$\delta = \frac{pR}{E_o} \left( \frac{r_o^2 + R^2}{r_o^2 - R^2} + \nu_o \right) + \frac{pR}{E_i} \left( \frac{R^2 + r_i^2}{R^2 - r_i^2} - \nu_i \right)$$

where  $E_o$  and  $E_i$  are the modules of elasticity;  $\nu_o$  and  $\nu_i$  are the Poisson's ratios for the outer and inner members. If the materials of the outer and inner members are same, i.e.  $E_o = E_i$  and  $\nu_o = \nu_i$ , then the pressure  $p$  is given directly as

$$p = \frac{E\delta}{R} \left[ \frac{(r_o^2 - R^2)(R^2 - r_i^2)}{2R^2(r_o^2 - r_i^2)} \right].$$