

Pole placement technique for PSS and TCSC-based stabilizer design using simulated annealing

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Abstract

A pole placement technique for power system stabilizer (PSS) and thyristor controlled series capacitor (TCSC) based stabilizer using simulated annealing (SA) algorithm is presented in this paper. The proposed approach employs SA optimization technique to PSS (SAPSS) and TCSC-based stabilizer (SACSC) design. The design problem is formulated as an optimization problem where SA is applied to search for the optimal setting of the proposed SAPSS and SACSC parameters. A pole placement-based objective function to shift the dominant eigenvalues to the left in the s -plane is considered. The proposed SAPSS and SACSC have been examined on a weakly connected power system with different disturbances, loading conditions, and system parameter variations. Eigenvalue analysis and nonlinear simulation results show the effectiveness and the robustness of the proposed stabilizers and their ability to provide efficient damping of low frequency oscillations. In addition, the performance of the proposed stabilizers outperforms that of the conventional power system stabilizer (CPSS). It is also observed that the proposed SACSC improves greatly the voltage profile of the system under severe disturbances. © 2000 Elsevier Science Ltd. All rights reserved.

Keywords: Thyristor controlled series capacitor; Power system stabilizer; Simulated annealing; Pole placement

1. Introduction

Power systems are experiencing low frequency oscillations due to disturbances. The oscillations may sustain and grow to cause system separation if no adequate damping is available. To enhance system damping, the generators are equipped with power system stabilizers (PSSs) that provide supplementary feedback stabilizing signals in the excitation systems. PSSs extend the power system stability limit by enhancing the system damping of low frequency oscillations associated with the electromechanical modes [1].

Eigenvalue assignment techniques have been reported in the literature pertaining to design problems of conventional power system stabilizers (CPSSs) [2,3]. Unfortunately, the proposed techniques are iterative and require heavy computation burden due to system reduction procedure. This gives rise to time consuming computer codes. In addition, the initialization step of these algorithms is crucial and affects the final dynamic response of the controlled system. Hence, different designs assigning the same set of eigenvalues were simply obtained by using different initializations. Therefore,

a final selection criterion is required to avoid long runs of validation tests on the nonlinear model. Other techniques such as mathematical programming [4] have been applied to the problem of tuning of PSSs. The problem has been formulated as both a quadratic and a linear programming problem. However, this formulation is carried out at the expense of some conservativeness and the number of constraints becomes unduly large. A gradient procedure for optimization of PSS parameters is presented in [5]. The optimization process requires computations of sensitivity factors and eigenvectors at each iteration. This give rise to heavy computational burdens and slow convergence. In addition, the search process is susceptible to be trapped in local minima and the solution obtained will not be optimal. Several approaches based on modern control theory have been applied to PSS design problem. These include optimal control, adaptive control, variable structure control, and intelligent control [6–9].

The recent advances in power electronics have led to the development of reliable and high-speed flexible AC transmission system (FACTS) devices [10–12] such as static VAR compensator (SVC), thyristor-controlled phase shifter (TCPS), and thyristor controlled series capacitor (TCSC). FACTS are designed to enhance power system stability by increasing the system damping in addition to their primary

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Nomenclature

P_m	Mechanical input power of the generator
P_e	Electrical output power of the generator
Z	Transmission line impedance, $Z = R + jX$
Y_L	Local load admittance, $Y_L = g + jb$
D	Damping constant of the generator
δ	Rotor angle of the generator
ω	Speed of the generator
ρ	Derivative operator d/dt
E_{fd}	Field voltage
T'_{do}	Open circuit field time constant
x_d	d -axis reactance of the generator
x'_d	d -axis transient reactance of the generator
x_q	q -axis reactance of the generator
K_A	Gain of the excitation system
T_A	Time constant of the excitation system
V_{ref}	Reference voltage
v	Terminal voltage of the generator
v_d	d -axis component of the terminal voltage
v_q	q -axis component of the terminal voltage
i_d	d -axis component of the armature current
i_q	q -axis component of the armature current
E'_q	Transient EMF in q -axis of the generator
u_{PSS}	Output signal of the PSS
u_{CSC}	Output signal of the TCSC-based stabilizer
X_{CSC}^{ref}	TCSC reference reactance
X_{CSC}	TCSC reactance
K_C	Gain of the TCSC
T_C	Time constant of the TCSC

functions such as voltage and power flow control. In this paper, the TCSC as one of the promising FACTS devices is considered.

A considerable attention has been directed for investigating the effect of TCSC on power system stability [13–20]. Several approaches based on modern control theory have been applied to TCSC controller design. Chen et al. [13] presented a state feedback controller for TCSC by using a pole placement technique. However, the controller requires all system states. This reduces its applicability. Chang and Chow [14] developed a time optimal control strategy for the TCSC where a performance index of time was minimized. A fuzzy logic controller for a TCSC was proposed in [15]. The impedance of the TCSC was adjusted based on machine rotor angle and the magnitude of the speed deviation. In addition, different control schemes for a TCSC were proposed such as variable structure controller [16,17], bilinear generalized predictive controller [18], H_∞ -based controller [19], and neural network controller [20].

Despite the potential of modern control techniques with different structures, power system utilities still prefer a conventional lead-lag controller structure [21–23]. The reasons behind that might be the ease of on-line tuning

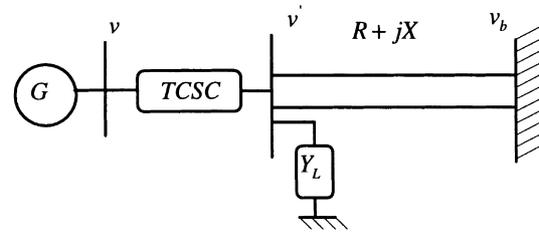


Fig. 1. Single machine infinite bus system with a TCSC.

and the lack of assurance of the stability related to some adaptive or variable structure techniques. It is shown that the appropriate selection of conventional lead-lag stabilizer parameters results in effective damping to low frequency oscillations [23].

In the last few years, simulated annealing (SA) algorithm [24–26] appeared as a promising heuristic algorithm for handling the combinatorial optimization problems. It has been theoretically proved that the SA algorithm converges to the optimum solution. The SA algorithm is robust, i.e. the final solution quality does not strongly depend on the choice of the initial solution. Therefore, the algorithm can be used to improve the solution of other methods. Another strong feature of SA algorithm is that a complicated mathematical model is not required and the constraints can be easily incorporated [24]. Unlike the gradient-descent techniques, SA is a derivative-free optimization algorithm and no sensitivity analysis is required to evaluate the objective function. This feature simplifies the constraints imposed on the objective function considered.

In this paper, a pole placement technique for simulated annealing-based PSS (SAPSS) and simulated annealing-based TCSC stabilizer (SACSC) design is presented. The design objective is to shift the unstable and poorly damped modes to the left in the s -plane. The proposed SAPSS and SACSC have been applied and tested on a weakly connected power system under several loading conditions and severe disturbances. The effectiveness of the proposed stabilizers to extend the system stability limit and to improve the system voltage profile is demonstrated. In addition, the robustness of the proposed stabilizers to system parameter variations is examined.

2. Thyristor controlled series capacitor

In this study, a single machine infinite bus system with a TCSC shown in Fig. 1 is considered. A TCSC has been placed in series with the transmission line to change the line flow. Therefore, a TCSC can extend the power transfer capability and provide additional damping for low frequency oscillations. The configuration of a TCSC is shown in Fig. 2. It comprises a fixed capacitor in parallel with a thyristor-controlled reactor. Controlling the firing angle of the thyristors can regulate the TCSC reactance and its degree of compensation.

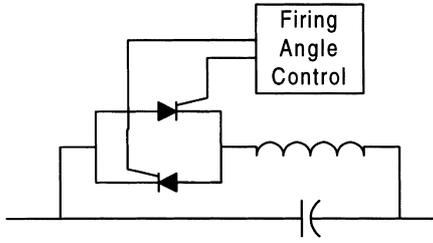


Fig. 2. The configuration of TCSC.

3. Linearized power system model

The generator is represented by the third-order model comprising of the electromechanical swing equation and the generator internal voltage equation. The swing equation is divided into the following equations:

$$\rho\delta = \omega_b(\omega - 1) \quad (1)$$

$$\rho\omega = \frac{(P_m - P_e - D)(\omega - 1)}{M} \quad (2)$$

P_m is assumed to be constant and P_e can be expressed as

$$P_e = v_d i_d + v_q i_q \quad (3)$$

The internal voltage, E'_q , equation is

$$\rho E'_q = \frac{(E_{fd} - (x_d - x'_d) i_d - E'_q)}{T'_{do}} \quad (4)$$

The IEEE Type-ST1 excitation system shown in Fig. 3 is considered in this study. It can be described as

$$\rho E_{fd} = \frac{(K_A(V_{ref} - v + u_{PSS}) - E_{fd})}{T_A} \quad (5)$$

where

$$v = (v_d^2 + v_q^2)^{1/2} \quad (6)$$

$$v_d = x_q i_q \quad (7)$$

$$v_q = E'_q - x'_d i_d \quad (8)$$

Fig. 4 illustrates the block diagram of a TCSC with conventional lead-lag stabilizer. The TCSC dynamics can be

expressed as

$$\rho X_{CSC} = \frac{(K_c(X_{CSC}^{ref} + u_{CSC}) - X_{CSC})}{T_c} \quad (9)$$

where X_{CSC}^{ref} represents the desired degree of compensation.

In the design of damping controller, the linearized incremental model around a nominal operating point is usually employed [1]. The linearized power system model can be written as, see Appendix A,

$$\begin{bmatrix} \rho\Delta\delta \\ \rho\Delta\omega \\ \rho\Delta E'_q \\ \rho\Delta E_{fd} \end{bmatrix} = \begin{bmatrix} 0 & 377 & 0 & 0 \\ -\frac{K_1}{M} & -\frac{D}{M} & -\frac{K_2}{M} & 0 \\ -\frac{K_4}{T'_{do}} & 0 & -\frac{K_3}{T'_{do}} & \frac{1}{T'_{do}} \\ -\frac{K_A K_5}{T_A} & 0 & -\frac{K_A K_6}{T_A} & -\frac{1}{T_A} \end{bmatrix} \times \begin{bmatrix} \Delta\delta \\ \Delta\omega \\ \Delta E'_q \\ \Delta E_{fd} \end{bmatrix} + \begin{bmatrix} 0 & 0 \\ 0 & -\frac{K_p}{M} \\ 0 & -\frac{K_q}{T'_{do}} \\ \frac{K_A}{T_A} & -\frac{K_A K_v}{T_A} \end{bmatrix} \begin{bmatrix} u_{PSS} \\ \Delta X_{CSC} \end{bmatrix} \quad (10)$$

In short, the linearized system model can be written as

$$\rho X = AX + BU \quad (11)$$

Fig. 5 illustrates the block diagram of the linearized power system model. The expressions of constants $K_1 - K_6$, K_p , K_q , and K_v are given in Appendix A.

4. Simulated annealing algorithm

4.1. Overview

Simulated annealing is an optimization technique that simulates the physical annealing process in the field of combinatorial optimization. Annealing is the physical

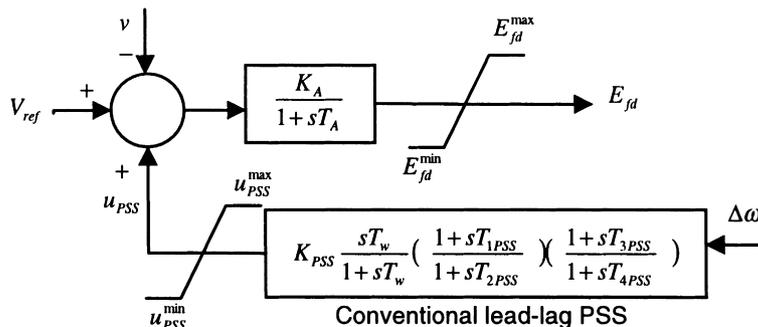


Fig. 3. IEEE Type-ST1 excitation system with conventional PSS.

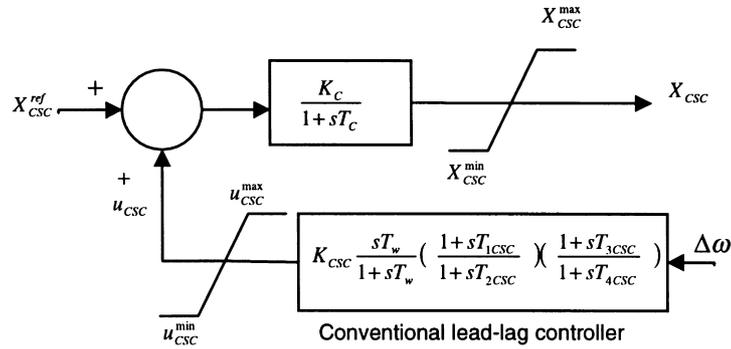


Fig. 4. TCSC with conventional lead-lag stabilizer.

process of heating up a solid until it melts, followed by slow cooling it down by decreasing the temperature of the environment in steps. At each step, the temperature is maintained constant for a period of time sufficient for the solid to reach thermal equilibrium. At any temperature T , the thermal equilibrium state is characterized by the *Boltzmann distribution*. This distribution gives the probability of the solid being in a state i with energy E_i at temperature T as

$$P_i = k \exp(-E_i/T) \tag{12}$$

where k is a constant.

Metropolis et al. [25] proposed a Monte Carlo method to simulate the process of reaching thermal equilibrium at a fixed value of the temperature T . In this method, a randomly generated perturbation of the current configuration of the solid is applied so that a trial configuration is obtained. Let E_c and E_t denote the energy level of the current and trial configurations, respectively. If $E_t < E_c$, then a lower energy level has been reached, and the trial configuration is accepted and becomes the current configuration. On the

other hand, if $E_t \geq E_c$ the trial configuration is accepted as current configuration with probability proportional to $\exp(-\Delta E/T)$, $\Delta E = E_t - E_c$. The process continues until the thermal equilibrium is achieved after a large number of perturbations, where the probability of a configuration approaches the *Boltzmann distribution*.

By gradually decreasing the temperature T and repeating Metropolis simulation, new lower energy levels become achievable. As T approaches zero least energy configurations will have a positive probability of occurring.

4.2. SA algorithm

At first, the analogy between a physical annealing process and a combinatorial optimization problem is based on the following [24]:

- Solutions in an optimization problem are equivalent to configurations of a physical system.
- The cost of a solution is equivalent to the energy of a configuration.

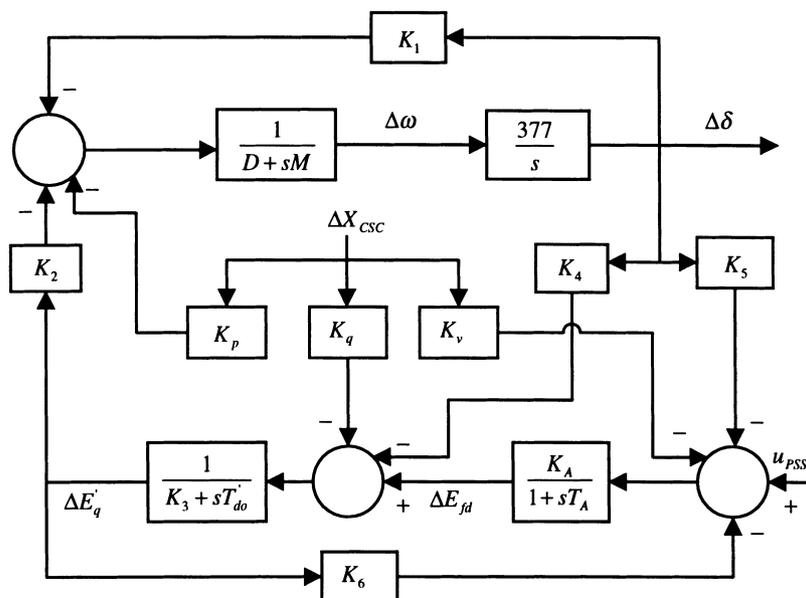


Fig. 5. Block diagram of the linearized power system model.

In addition, a control parameter C_p is introduced to play the role of the temperature T .

The basic elements of SA are briefly stated and defined as follows.

- *Current, trial, and best solutions*, x_{current} , x_{trial} , and x_{best} : these solutions are sets of the optimized parameter values at any iteration.
- *Acceptance criterion*: at any iteration, the trial solution can be accepted as the current solution if it meets one of the following criteria—(a) $J(x_{\text{trial}}) < J(x_{\text{current}})$; (b) $J(x_{\text{trial}}) > J(x_{\text{current}})$ and $\exp(-(J(x_{\text{trial}}) - J(x_{\text{current}}))/C_p) \geq \text{rand}(0, 1)$. Here, $\text{rand}(0, 1)$ is a random number with domain $[0, 1]$ and $J(x_{\text{trial}})$ and $J(x_{\text{current}})$ are the objective function values associated with x_{trial} and x_{current} , respectively. Criterion (b) indicates that the trial solution is not necessarily rejected if its objective function is not as good as that of the current solution with hoping that a much better solution becomes reachable.
- *Acceptance ratio*: at a given value of C_p , an n_1 trial solutions can be randomly generated. Based on the acceptance criterion, an n_2 of these solutions can be accepted. The acceptance ratio is defined as n_2/n_1 .
- *Markov chain*: it is defined as a sequence of trial solutions where the probability of the outcome of a given trial solution depends only on the outcome of the previous trial solution. In the SA algorithm, the set of the outcomes is given by the finite set of solutions. The acceptance criteria described above show clearly that the outcome of a trial solution depends only on the outcome of the previous one. Hence, the concept of the Markov chain can be used [24]. The allowable number of transitions at each value of the control parameter C_p represents the length of each chain.
- *Cooling schedule*: it specifies a set of parameters that governs the convergence of the algorithm. This set includes an initial value of control parameter C_{p0} , a decrement function for decreasing the value of C_p , and a finite number of iterations or transitions at each value of C_p , i.e. the length of each homogeneous Markov chain. The initial value of C_p should be large enough to allow virtually all transitions to be accepted. However, this can be achieved by starting off at a small value of C_{p0} and multiplying it with a constant larger than 1, α , i.e. $C_{p0} = \alpha C_{p0}$. This process continues until the acceptance ratio is close to 1. This is equivalent to heating up process in physical systems. The decrement function for decreasing the value of C_p is given by $C_p = \mu C_p$ where μ is a constant smaller than but close to 1. Typical values lie between 0.8 and 0.99 [24]. The acceptance criteria show that at large values of C_p , large deteriorations will be accepted; as C_p decreases, only smaller deteriorations will be accepted and finally, as C_p approaches zero, no deteriorations will be accepted at all. This feature, in contrast to gradient-descent and other local search algorithms, avoids trapping in local minima.

- *Equilibrium condition*: it occurs when the current solution does not change for a certain number of iterations at a given value of C_p . It can be achieved by generating a large number of transitions at that value of C_p .
- *Stopping criteria*: these are the conditions under which the search process will terminate. In this study, the search will terminate if one of the following criteria is satisfied—(a) the number of Markov chains since the last change of the best solution is greater than a prespecified number; or (b) the number of Markov chains reaches the maximum allowable number.

The general algorithm of SA can be described in steps as follows.

Step 1: Set the initial value of C_{p0} and randomly generate an initial solution x_{initial} and calculate its objective function. Set this solution as the current solution as well as the best solution, i.e. $x_{\text{initial}} = x_{\text{current}} = x_{\text{best}}$.

Step 2: Randomly generate an n_1 of trial solutions in the neighborhood of the current solution.

Step 3: Check the acceptance criterion of these trial solutions and calculate the acceptance ratio. If acceptance ratio is close to 1 go to step 4; else set $C_{p0} = \alpha C_{p0}$, $\alpha > 1$, and go back to Step 2.

Step 4: Set the chain counter $k_{ch} = 0$.

Step 5: Generate a trial solution x_{trial} . If x_{trial} satisfies the acceptance criterion set $x_{\text{current}} = x_{\text{trial}}$, $J(x_{\text{current}}) = J(x_{\text{trial}})$, and go to Step 6; else go to Step 6.

Step 6: Check the equilibrium condition. If it is satisfied go to Step 7; else go to Step 5.

Step 7: Check the stopping criteria. If one of them is satisfied then stop; else set $k_{ch} = k_{ch} + 1$ and $C_p = \mu C_p$, $\mu < 1$, and go back to Step 5.

5. Proposed design approach

5.1. Structure of the proposed stabilizers

A conventional lead-lag controller structure for both SAPSS and SACSC as shown in Figs. 2 and 3 is considered in this study. The stabilizing signals of the proposed SAPSS and SACSC can be expressed as

$$u_{\text{PSS}} = K_{\text{PSS}} \frac{sT_w}{1 + sT_w} \left(\frac{1 + sT_1 \text{PSS}}{1 + sT_2 \text{PSS}} \right) \left(\frac{1 + sT_3 \text{PSS}}{1 + sT_4 \text{PSS}} \right) \Delta\omega \quad (13)$$

$$u_{\text{CSC}} = K_{\text{CSC}} \frac{sT_w}{1 + sT_w} \left(\frac{1 + sT_1 \text{CSC}}{1 + sT_2 \text{CSC}} \right) \left(\frac{1 + sT_3 \text{CSC}}{1 + sT_4 \text{CSC}} \right) \Delta\omega \quad (14)$$

In this structure, the washout time constant T_w and the time constants $T_2 \text{PSS}$, $T_4 \text{PSS}$, $T_2 \text{CSC}$, and $T_4 \text{CSC}$ are usually prespecified. In this study, $T_w = 5$ s and $T_2 \text{PSS} = T_4 \text{PSS} = T_2 \text{CSC} = T_4 \text{CSC} = 0.1$ s. The controller gains, K_{PSS} and

K_{CSC} , and time constants, $T_{1\text{ PSS}}$, $T_{3\text{ PSS}}$, $T_{1\text{ CSC}}$, and $T_{3\text{ CSC}}$ remain to be determined.

5.2. Problem formulation

To increase the system damping to the electromechanical modes, the objective function J defined below is proposed.

$$J = \sum_{\sigma_i \geq \sigma_0} (\sigma_0 - \sigma_i)^2 \tag{15}$$

where σ_i is the real part of the i^{th} eigenvalue and σ_0 is a chosen threshold. The value of σ_0 represents the desirable level of system damping. This level can be achieved by shifting the dominant eigenvalues to the left of $s = \sigma_0$ line in the s -plane. This insures also some degree of relative stability. The condition $\sigma_i \geq \sigma_0$ is imposed on J evaluation to consider only the unstable or poorly damped modes.

The problem constraints are the parameter bounds. Therefore, the design problem can be formulated as the following optimization problem.

Minimize J (16)

Subject to

$$K_{PSS}^{\min} \leq K_{PSS} \leq K_{PSS}^{\max} \tag{17}$$

$$T_{1\text{ PSS}}^{\min} \leq T_{1\text{ PSS}} \leq T_{1\text{ PSS}}^{\max} \tag{18}$$

$$T_{3\text{ PSS}}^{\min} \leq T_{3\text{ PSS}} \leq T_{3\text{ PSS}}^{\max} \tag{19}$$

$$K_{CSC}^{\min} \leq K_{CSC} \leq K_{CSC}^{\max} \tag{20}$$

$$T_{1\text{ CSC}}^{\min} \leq T_{1\text{ CSC}} \leq T_{1\text{ CSC}}^{\max} \tag{21}$$

$$T_{3\text{ CSC}}^{\min} \leq T_{3\text{ CSC}} \leq T_{3\text{ CSC}}^{\max} \tag{22}$$

The minimum and maximum values of the controller gains are set as 0.1 and 100, respectively [26]. The minimum

Table 1
The optimal settings of the controller parameters of the proposed schemes

Proposed SAPSS			Proposed SACSC		
K_{PSS}	$T_{1\text{ PSS}}$	$T_{3\text{ PSS}}$	K_{CSC}	$T_{1\text{ CSC}}$	$T_{3\text{ CSC}}$
22.523	0.1212	0.2156	74.726	0.0115	0.1249

values of $T_{1\text{ PSS}}$ and $T_{3\text{ PSS}}$ are set slightly above the value of $T_{2\text{ PSS}}$ and $T_{4\text{ PSS}}$, respectively, to compensate the phase lag between u_{PSS} and E'_q [27]. The maximum values of $T_{1\text{ PSS}}$, $T_{3\text{ PSS}}$, $T_{1\text{ CSC}}$, and $T_{3\text{ CSC}}$ are set to 1.0 s.

5.3. Application of SA algorithm

The SA algorithm has been applied to the above optimization problem to search for optimal settings of the proposed stabilizers. In our implementation, the threshold value σ_0 was chosen to be -3.0 . The search will terminate if one of the following stopping criteria is satisfied: (1) best solution does not change for more than 20 chains; (2) number of chains reaches 100; or (3) value of the objective function reaches zero, i.e. all the dominant eigenvalues have been shifted to the left of $s = \sigma_0$ line.

The final settings of the optimized parameters for the proposed stabilizers are given in Table 1. The convergence rate of the objective function J with the number of iterations is shown in Fig. 6. It is worth mentioning that the optimization process has been carried out with the system operating at nominal loading condition given in Table 2.

6. Simulation results

To assess the effectiveness and robustness of the proposed stabilizers, three different loading conditions given in Table 2 were considered with different

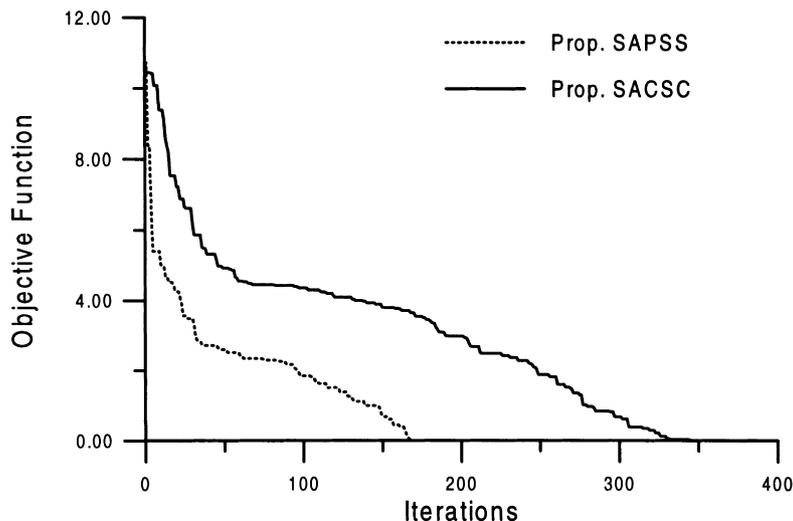


Fig. 6. Objective function variations of the proposed schemes.

Table 2
Loading conditions

Loading	P (pu)	Q (pu)	X_{csc} (pu)
Nominal	1.00	0.015	0.0
Light	0.70	0.300	0.2
Heavy	1.10	0.400	-0.2

disturbances. The performance of the proposed stabilizers is compared to that of CPSS given in Refs. [27,28].

It is worth mentioning that all the time domain simulations were carried out using the nonlinear power system model. The system data is given in Appendix A.

6.1. Nominal loading

At this loading condition, the system eigenvalues with and without the proposed stabilizers are given in Table 3. It is shown that the open loop system is unstable because of the negative damping of electromechanical mode. It is

Table 3
System eigenvalues with and without control in 1/s

No Control	CPSS	Prop. SAPSS	Prop. SACSC
$+0.30 \pm j4.96$	$-1.16 \pm j4.40$	$-3.00 \pm j6.74$	$-3.32 \pm j5.57$
$-10.39 \pm j3.28$	$-4.60 \pm j7.41$	$-3.29 \pm j4.67$	$-4.31 \pm j5.77$
-	-0.20, -18.68	-19.24, -8.37	-13.47, -8.37
-	-	-0.21	-23.09, -0.21

quite clear that the proposed stabilizers outperform the CPSS and shift substantially the electromechanical mode eigenvalue to the left of the line $s = -3.0$ in the s -plane. This enhances greatly the system stability and improves the damping characteristics of electromechanical mode.

The behavior of the proposed stabilizer under transient conditions was verified by applying a 6-cycle three-phase fault at the infinite bus at the end of one transmission line. The fault was cleared without line tripping. The system response is shown in Fig. 7. It can be seen that the response

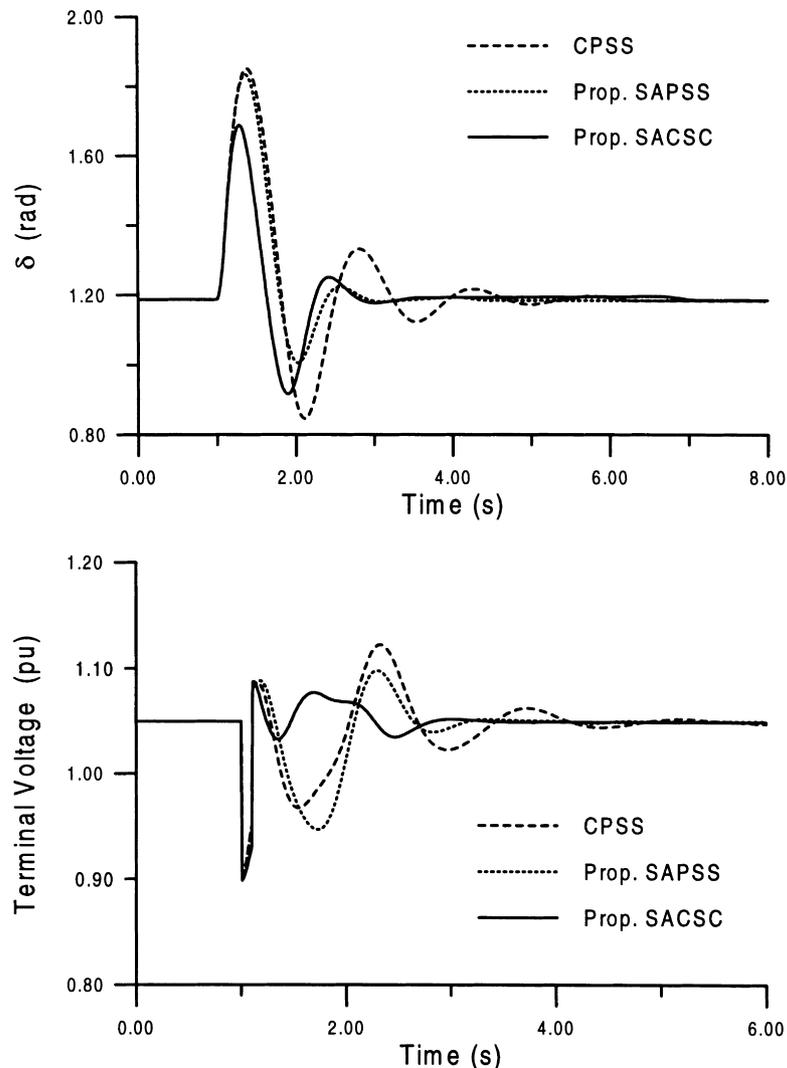


Fig. 7. System response to the fault test with nominal loading without line tripping.

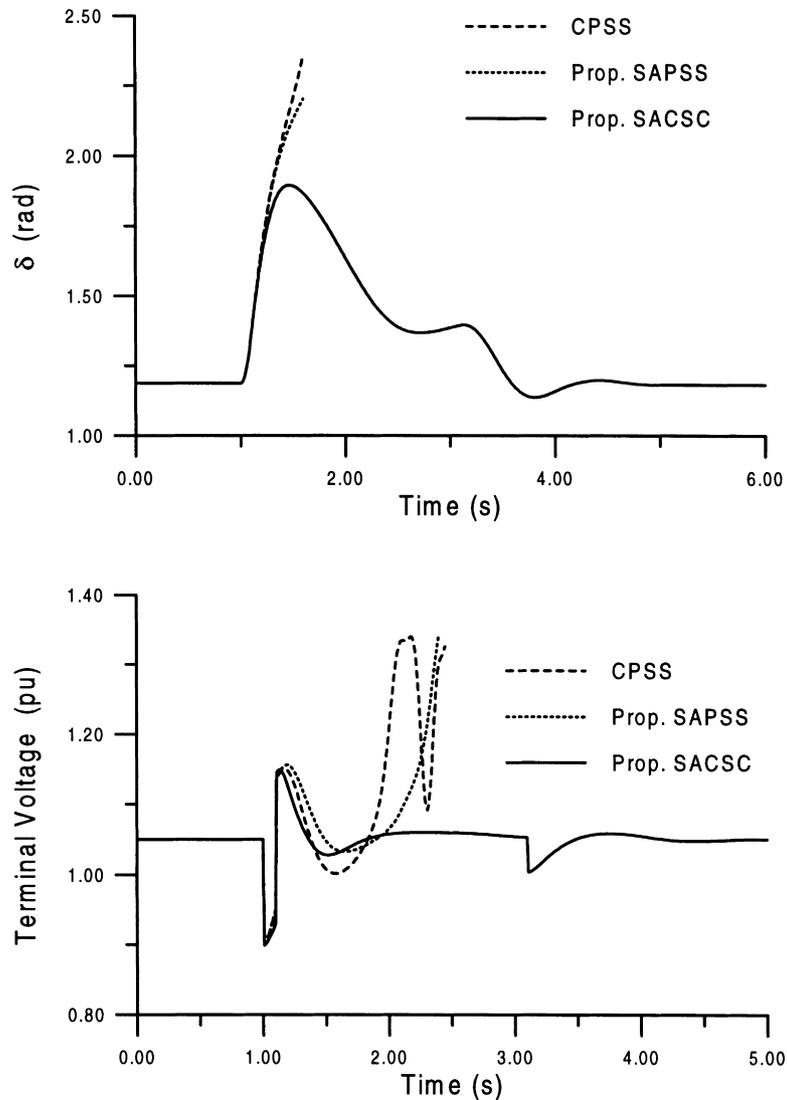


Fig. 8. System response to the fault test with nominal loading with line tripping for 2 s.

of the proposed stabilizers is much faster than that of CPSS. In addition, the first swing in the torque angle is significantly suppressed and the voltage profile is greatly improved with the proposed SACSC.

Another severe disturbance was considered at this loading condition; that is, a 6-cycle fault was applied as above. However, the faulty line was tripped for 2 s. The system response to this disturbance is shown in Fig. 8. It is clear that the proposed SACSC has good damping characteristics to low frequency oscillations and stabilizes the system under this severe disturbance. This extends the power system stability limit and the power transfer capability. These positive results of the proposed SACSC can be attributed to its faster response compared to that of power system stabilizer. It can be concluded that although PSSs extend the power system stability limit by enhancing the system damping, they suffer a drawback of being liable to cause great variations in the voltage profile and they

may even result in losing system stability under severe disturbances.

6.2. Light loading

A 6-cycle three-phase fault disturbance at the infinite bus was applied. The faulty line was tripped for 4 s. The results are shown in Fig. 9. It can be seen that the proposed stabilizers suppress the first swing in torque angle and extend the system stability limit. In addition, the voltage profile is greatly improved with the proposed SACSC stabilizer in terms of overshoots and settling time.

A parameter variation test was also applied to assess the robustness of the proposed stabilizers. At $t = 1$ s, one of the transmission lines was tripped for maintenance purposes, i.e. the line impedance was doubled. The system response is shown in Fig. 10. It is clear that the proposed stabilizers outperform the CPSS. In addition, the system

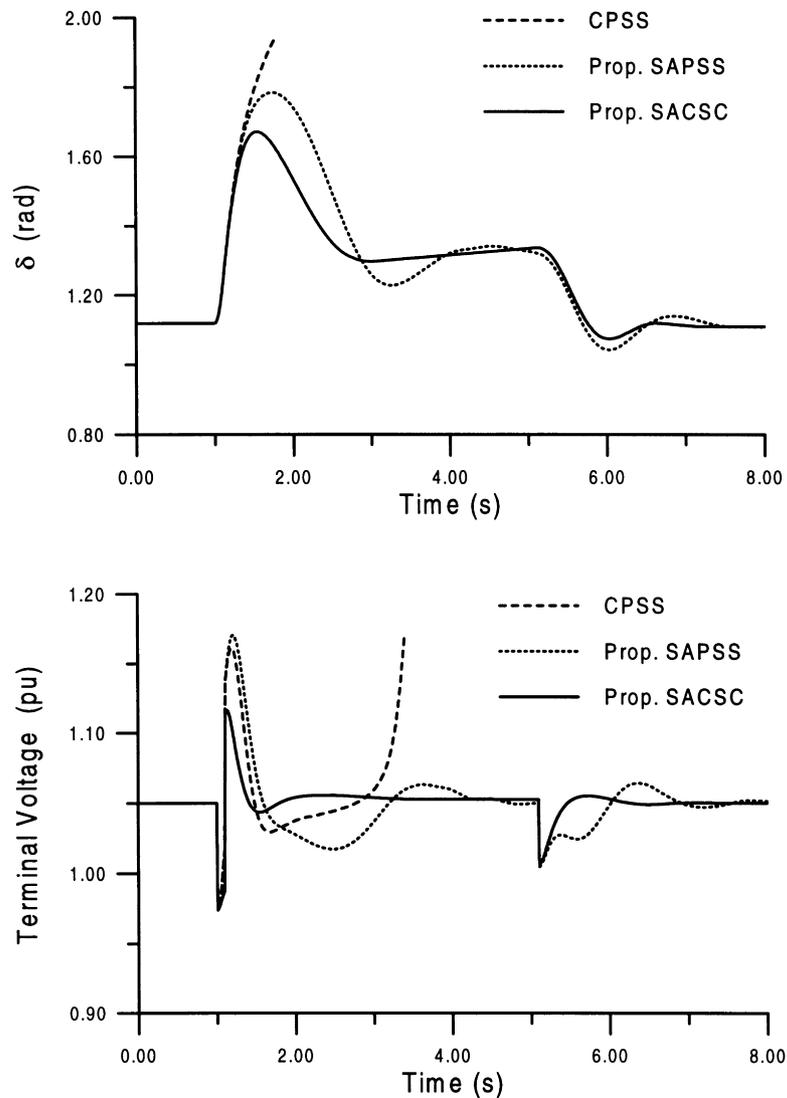


Fig. 9. System response to the fault test with light loading with line tripping for 4 s.

voltage profile is greatly improved with the proposed SACSC.

6.3. Heavy loading

A 5-cycle three-phase fault disturbance at the infinite bus was applied. The results are shown in Fig. 11. It can be seen that the first swing is substantially suppressed and the voltage profile is greatly improved with the proposed SACSC. It can be concluded that the proposed SACSC has better damping characteristics to electromechanical modes of oscillations and improves greatly the power system stability and system voltage profile.

7. Conclusions

In this study, a pole placement technique for PSS and TCSC-based stabilizer is presented. The tuning parameters

of the proposed SAPSS and SACSC were optimized using simulated annealing algorithm as the optimization engine. The proposed stabilizers have been applied and tested on a weakly connected power system under different disturbances, loading conditions, and system parameter variations. The eigenvalue analysis and the nonlinear simulation results show the effectiveness and the robustness of the proposed stabilizers and their ability to provide good damping of low frequency oscillations and improve greatly the system voltage profile. In addition, the TCSC-based stabilizer outperforms the power system stabilizers in terms of the first swing stability and the voltage profile. It can be concluded that although PSSs extend the power system stability limit by enhancing the system damping, they suffer a drawback of being liable to cause great variations in the voltage profile and they may even result in losing system stability under severe disturbances.

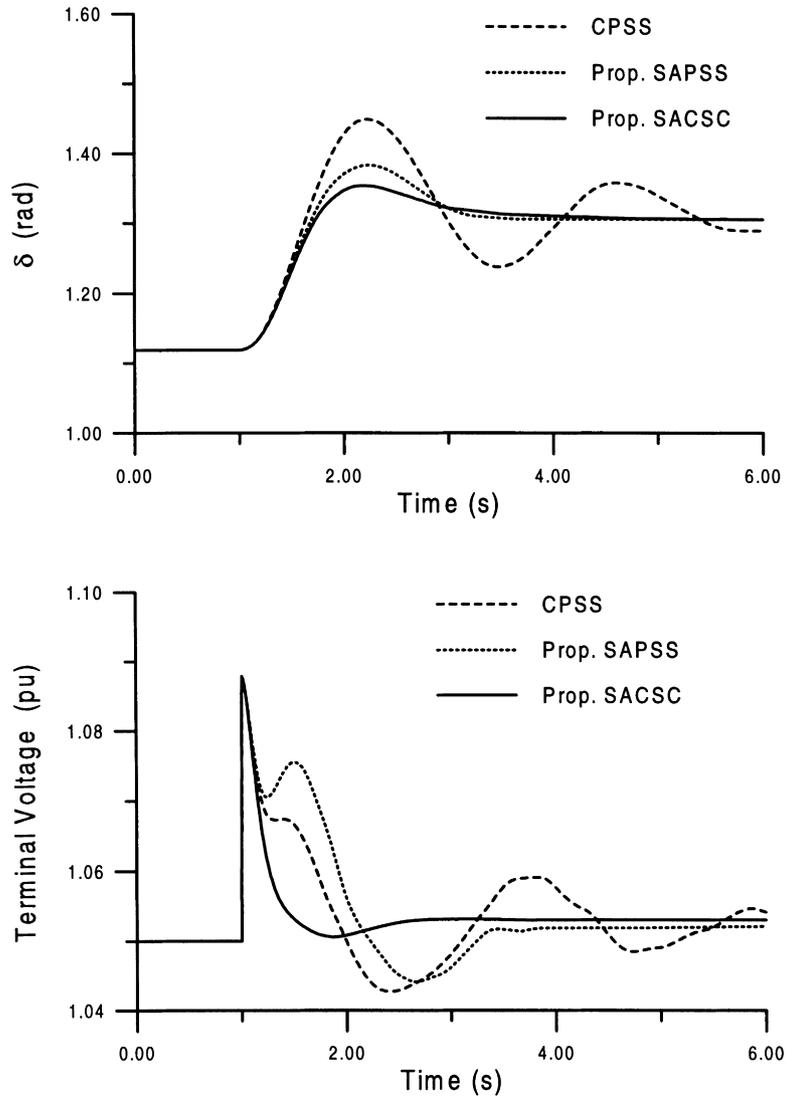


Fig. 10. System response to the line impedance variation test with light loading.

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Appendix A

Referring to Fig. 1, the voltage $v' = v - jX_{CSC}i$, i is the generator armature current. The d and q components of v' can be written as

$$v'_d = (x_q + X_{CSC})i_q \quad (A1)$$

$$v'_q = E'_q - (x'_d + X_{CSC})i_d \quad (A2)$$

The load current $i_L = v'Y_L$, and the line current $i_{line} = i - i_L$. The infinite bus voltage $v_b = v' - i_{line}Z$. The

components of v_b can be written as

$$v_b \sin \delta = c_1 v'_d - c_2 v'_q - R i_d + X i_q \quad (A3)$$

$$v_b \cos \delta = c_2 v'_d + c_1 v'_q - X i_d - R i_q \quad (A4)$$

Substituting from Eqs. (A1) and (A2) into Eqs. (A3) and (A4), the following two equations can be obtained:

$$c_3 i_d + c_4 i_q = v_b \sin \delta + c_2 E'_q \quad (A5)$$

$$c_5 i_d + c_6 i_q = v_b \cos \delta - c_1 E'_q \quad (A6)$$

Solving Eqs. (A5) and (A6) simultaneously, i_d and i_q expressions can be obtained. Linearizing Eqs. (A5) and (A6) at the nominal loading condition, Δi_d and Δi_q can be expressed in

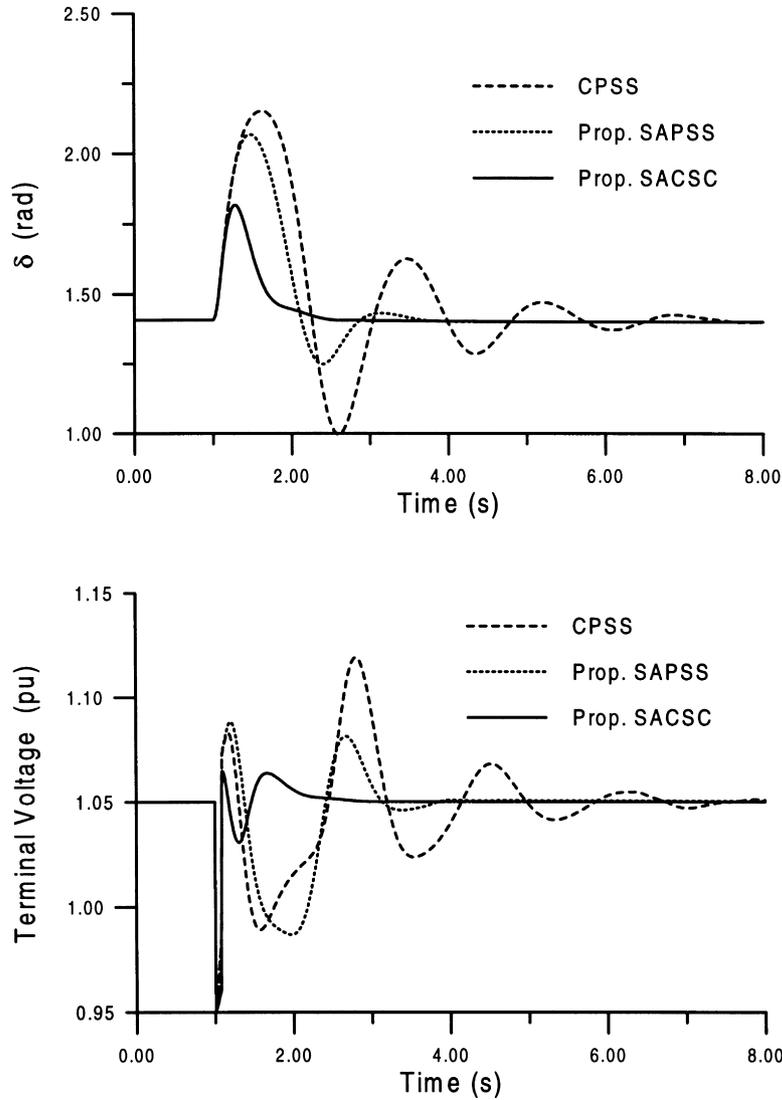


Fig. 11. System response to the fault test with heavy loading.

terms of $\Delta\delta$, $\Delta E'_q$, and ΔX_{CSC} as follows:

$$c_3\Delta i_d + c_4\Delta i_q = v_b \cos \delta \Delta\delta + c_2\Delta E'_q + c_7\Delta X_{CSC} \quad (A7)$$

$$c_5\Delta i_d + c_6\Delta i_q = -v_b \sin \delta \Delta\delta - c_1\Delta E'_q + c_8\Delta X_{CSC} \quad (A8)$$

Solving Eqs. (A7) and (A8) simultaneously, Δi_d and Δi_q can be expressed as

$$\Delta i_d = c_9\Delta\delta + c_{10}\Delta E'_q + c_{11}\Delta X_{CSC} \quad (A9)$$

$$\Delta i_q = c_{12}\Delta\delta + c_{13}\Delta E'_q + c_{14}\Delta X_{CSC} \quad (A10)$$

The constants c_1 – c_{14} are expressions of Z , Y_L , x'_d , x_q , i_{d0} , i_{q0} , E'_{q0} , and X_{CSC}

The linearized form of v_d and v_q can be written as

$$\Delta v_d = x_q\Delta i_q \quad (A11)$$

$$\Delta v_q = \Delta E'_q - x'_d\Delta i_d \quad (A12)$$

Using Eqs. (A1)–(A12), the following expressions can be easily obtained

$$\Delta P_e = K_1\Delta\delta + K_2\Delta E'_q + K_p\Delta X_{CSC} \quad (A13)$$

$$(K_3 + sT'_{do})\Delta E'_q = \Delta E_{fd} - K_4\Delta\delta - K_q\Delta X_{CSC} \quad (A14)$$

$$\Delta v = K_5\Delta\delta + K_6\Delta E'_q + K_v\Delta X_{CSC} \quad (A15)$$

where the constants K_1 – K_6 , K_p , K_q , and K_v are expressions of c_9 – c_{14} .

Appendix B

The system data are as follows.

$$M = 9.26 \text{ s}; \quad T'_{do} = 7.76; \quad D = 0.0;$$

$$x_d = 0.973; \quad x'_d = 0.19; \quad x_q = 0.55;$$

$$R = -0.034;^1 \quad X = 0.997; \quad g = 0.249;$$

$$b = 0.262; \quad K_A = 50; \quad T_A = 0.05;$$

$$K_s = 1.0; \quad T_s = 0.05;$$

$$|u_{PSS}| \leq 0.2 \text{ pu}; \quad |u_{CSC}| \leq 0.2 \text{ pu};$$

$$|X_{CSC}| = 0.5X|E_{fd}| \leq 7.3 \text{ pu}.$$

All resistances and reactances are in pu and time constants are in seconds.

With the nominal loading condition given in Table 2, the system matrices are

$$A = \begin{bmatrix} 0.0 & 3.77 & 0.0 & 0.0 \\ -0.0588 & 0.0 & -0.1303 & 0.0 \\ -0.0900 & 0.0 & -0.1957 & 0.1289 \\ 95.5320 & 0.0 & -815.93 & -20.00 \end{bmatrix}$$

$$B = \begin{bmatrix} 0.0 & 0.0 \\ 0.0 & 0.0704 \\ 0.0 & 0.0177 \\ 1000 & 93.846 \end{bmatrix}$$

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¹ The negative value of R stems from deriving of the one-machine-infinite-bus model by dynamic equivalencing of a multimachine system where smaller generators are replaced by equivalent impedances with negative resistances [27].