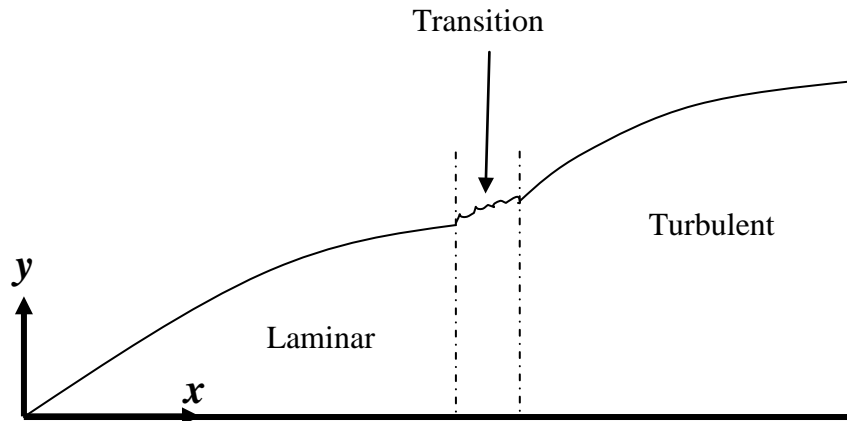


Boundary-Layer Flow over a Flat Plate

“Approximate Method”



The momentum balance on a control volume of the boundary layer leads to the following equation:

$$v_{\infty} \frac{d}{dx} \int_0^{\delta} v_x dy = \frac{\tau_w}{\rho} + \frac{d}{dx} \int_0^{\delta} v_x^2 dy \quad (1)$$

The approximate method of boundary layer analysis can be summarized as follows:

(a) Assume the velocity profile, v_x , as a function of x and y .

(b) Evaluate the integrals: $\int_0^{\delta} v_x dy$ and $\int_0^{\delta} v_x^2 dy$.

(c) Evaluate the wall-shear stress: $\tau_w \equiv \mu \left. \frac{\partial v_x}{\partial y} \right|_{y=0}$.

(d) Substitute in the above momentum balance and get δ and τ_w as functions of the distance, x , the velocity far from the plate, $v_{x\infty}$ and the fluid properties (ρ, μ).

The question now, how to assume v_x ? v_x can be assume as follows:

$$\frac{v_x}{v_{x\infty}} = f(\zeta) \text{ where } \zeta \equiv \frac{y}{\delta(x)}$$

The function $f(\zeta)$ must satisfy the following properties:

1. $f(0) = 0$ $(v_x = 0 \text{ @ } y = 0)$
2. $f(1) = 1$ $(v_x = v_{x\infty} \text{ @ } y = \delta)$
3. $\left. \frac{df}{d\zeta} \right|_{\zeta=1} = 0$ $\left(\left. \tau_{yx} \right|_{y=\delta} = \mu \left. \frac{\partial v_x}{\partial y} \right|_{y=\delta} = 0 \right)$; 4. $\left. \frac{d^2 f}{d\zeta^2} \right|_{\zeta=0} = 0$, 5. $\left. \frac{d^2 f}{d\zeta^2} \right|_{\zeta=1} = 0$

Laminar Boundary Layer

Now let us follow the summary outlined previously for the approximate method of boundary layer analysis:

- (a) The following velocity profile $\frac{v_x}{v_{x\infty}} = \text{Sin}\left(\frac{\pi}{2}\zeta\right)$ satisfies the properties of the function $f(\zeta)$ listed before, hence it can be assumed.

(b) Evaluate:
$$\int_0^{\delta} v_x dy = \int_0^{\delta} v_{x\infty} \sin\left(\frac{\pi}{2}\frac{y}{\delta}\right) dy = -v_{x\infty} \frac{2}{\pi} \delta \left[\cos\left(\frac{\pi}{2}\frac{y}{\delta}\right) \right]_0^{\delta} = \frac{2}{\pi} v_{x\infty} \delta$$

Also,
$$\int_0^{\delta} v_x^2 dy = \int_0^{\delta} v_{x\infty}^2 \text{Sin}^2\left(\frac{\pi}{2}\frac{y}{\delta}\right) dy = v_{x\infty}^2 \int_0^{\delta} \left[1 - \text{Cos}^2\left(\frac{\pi}{2}\frac{y}{\delta}\right) \right] dy$$

$= v_{x\infty}^2 \left[\int_0^{\delta} dy - \int_0^{\delta} \text{Cos}^2\left(\frac{\pi}{2}\frac{y}{\delta}\right) dy \right]$ (using integration formula in appendix

A)

$$= v_{x\infty}^2 \left[[y]_0^{\delta} - \frac{2\delta}{\pi} \frac{1}{2} \left[\frac{\pi}{2}\frac{y}{\delta} + \frac{1}{2} \text{Sin}\left(2\frac{\pi}{2}\frac{y}{\delta}\right) \right]_0^{\delta} \right] = v_{x\infty}^2 \frac{\delta}{2}$$

(c) Evaluate:
$$\tau_w = \mu \left. \frac{\partial v_x}{\partial y} \right|_{y=0}$$

the derivative: $\frac{\partial v_x}{\partial y} = \frac{\partial}{\partial y} \left(v_{x\infty} \sin \left(\frac{\pi y}{2 \delta} \right) \right) = v_{x\infty} \frac{\pi}{2 \delta} \cos \left(\frac{\pi y}{2 \delta} \right)$

and therefore: $\tau_w = \left[\mu v_{x\infty} \frac{\pi}{2 \delta} \cos \left(\frac{\pi y}{2 \delta} \right) \right]_{y=0} = \frac{\mu v_{x\infty} \pi}{2 \delta}$

(d) Substitute in momentum balance, equation (1):

$$v_{\infty} \frac{d}{dx} \left(\frac{2}{\pi} v_{\infty} \delta \right) = \frac{\mu v_{\infty} \pi}{2 \delta \rho} + \frac{d}{dx} \left(v_{\infty}^2 \frac{\delta}{2} \right) \quad \Rightarrow \quad \frac{2}{\pi} v_{\infty}^2 \frac{d\delta}{dx} = \frac{\mu v_{\infty} \pi}{2 \delta \rho} + \frac{v_{\infty}^2}{2} \frac{d\delta}{dx}$$

$$v_{\infty} \left(\frac{2}{\pi} - \frac{1}{2} \right) \frac{d\delta}{dx} = \frac{\mu \pi}{2 \rho \delta} \quad \Rightarrow \quad 0.137 v_{\infty} \frac{d\delta}{dx} = \frac{\mu \pi}{2 \rho \delta}$$

Integrate: $\int_0^{\delta} \delta \, d\delta = \frac{\mu \pi}{2 \rho (0.137 v_{\infty})} \int_0^x dx$

$$\frac{\delta^2}{2} = \frac{\mu \pi}{2 \rho (0.137 v_{\infty})} x \quad (\div x^2)$$

$$\frac{\delta^2}{x^2} = \frac{\mu \pi}{\rho (0.137 v_{\infty})} \frac{1}{x} = \frac{\pi}{0.137} \frac{\mu}{\rho v_{\infty} x}$$

Define Reynolds number for boundary layer flow: $Re_x = \frac{\rho v_{\infty} x}{\mu}$

$$\frac{\delta^2}{x^2} = \frac{\pi}{0.137} \frac{1}{Re_x} \quad \Rightarrow \quad \frac{\delta}{x} = \frac{4.79}{\sqrt{Re_x}}$$

The above equation provides the thickness of the boundary layer as a function of x and Re_x .

Now we evaluate the shear stress at the flat plate:

Recall, $\tau_w = \mu \frac{dv_x}{dy} \Big|_{y=0} = \frac{\mu v_{\infty} \pi}{2 \delta}$

$$\Rightarrow \tau_w = \frac{\mu v_\infty \pi}{2} \frac{\sqrt{\text{Re}_x}}{4.79 x} = \frac{\mu v_\infty \pi}{2} \frac{\sqrt{\frac{\rho v_\infty x}{\mu}}}{4.79 x} = \frac{\pi}{(2)(4.79)} \overbrace{\mu^{0.328}}^{0.328} \frac{\mu^{1/2} v_\infty^{3/2} \rho^{1/2}}{x^{1/2}}$$

Recall the definition of Drag Coefficient:

$$C_f \equiv \frac{\tau_w}{\frac{1}{2} \rho v_\infty^2}$$

$$C_f = 0.328 \frac{\mu^{1/2} v_\infty^{3/2} \rho^{1/2}}{x^{1/2}} \frac{1}{\frac{1}{2} \rho v_\infty^2} = (2)(0.328) \frac{\mu^{1/2}}{v_\infty^{1/2} \rho^{1/2} x^{1/2}} \quad \Rightarrow C_f = \frac{0.656}{\sqrt{\text{Re}_x}}$$

Turbulent Boundary Layer

The preceding analysis was for laminar B.L. However, it is found experimentally that when $\text{Re}_x > 3.2 \times 10^5$, the boundary layer becomes turbulent over a flat plate:

$$\text{Re}_x < 3.2 \times 10^5$$

Laminar boundary layer

$$\text{Re}_x > 3.2 \times 10^5$$

Turbulent boundary layer

What is the value of x at which the boundary layer undergoes transition to turbulent flow?

$$\frac{\rho v_\infty x}{\mu} = 3.2 \times 10^5 \quad \Rightarrow \quad x = 3.2 \times 10^5 \frac{\mu}{\rho v_\infty}$$

The velocity profile for turbulent boundary layer is based on experience from experimental data:

From experience: $\frac{v_x}{v_\infty} = \left(\frac{y}{\delta}\right)^{1/7}$ for turbulent B.L.

$$\int_0^\delta v_x dy = \int_0^\delta v_\infty \left(\frac{y}{\delta}\right)^{1/7} dy = \frac{7}{8} v_\infty \delta$$

$$\int_0^{\delta} v_x^2 dy = \int_0^{\delta} v_{\infty}^2 \left(\frac{y}{\delta}\right)^{2/7} dy = \frac{7}{9} v_{\infty}^2 \delta$$

$$\tau_w = \mu \left. \frac{dv_x}{dy} \right|_{y=0} = \mu \left[\frac{d}{dy} \left(v_{\infty} \left(\frac{y}{\delta}\right)^{1/7} \right) \right]_{y=0} = \mu \delta^{-1/7} \frac{1}{7} \left[\frac{1}{y^{6/7}} \right]_{y=0} = \mu \delta^{-1/7} \frac{1}{7} \left[\frac{1}{0} \right] = \infty!$$

Therefore, we have a problem that the turbulent velocity profile that is based on experience

$\frac{v_x}{v_{\infty}} = \left(\frac{y}{\delta}\right)^{1/7}$, predicts infinite shear stress τ_w . However, from experiment:

$$C_f = \frac{\tau_w}{\frac{1}{2} \rho v_{\infty}^2} = 0.045 \left(\frac{\mu}{\rho v_{\infty} \delta} \right)^{1/4}$$

$$\Rightarrow \tau_w = 0.045 \frac{1}{2} \rho v_{\infty}^2 \left(\frac{\mu}{\rho v_{\infty} \delta} \right)^{1/4}$$

Recall, the momentum balance on a control volume of the boundary layer:

$$v_{\infty} \frac{d}{dx} \int_0^{\delta} v_x dy = \frac{\tau_w}{\rho} + \frac{d}{dx} \int_0^{\delta} v_x^2 dy$$

$$\Rightarrow v_{\infty} \frac{d}{dx} \left(\frac{7}{8} v_{\infty} \delta \right) = \frac{1}{\rho} \left(0.045 \frac{1}{2} \rho v_{\infty}^2 \left(\frac{\mu}{\rho v_{\infty} \delta} \right)^{1/4} \right) + \frac{d}{dx} \left(\frac{7}{9} v_{\infty}^2 \delta \right)$$

$$\Rightarrow \left(\frac{7}{8} - \frac{7}{9} \right) v_{\infty}^2 \frac{d\delta}{dx} = \frac{0.045}{2} v_{\infty}^2 \left(\frac{\mu}{\rho v_{\infty} \delta} \right)^{1/4}$$

Integrating and simplifying:

$$\Rightarrow \frac{\delta}{x} = \frac{0.376}{(\text{Re}_x)^{1/5}}$$

and

$$C_f = 0.045 \left(\frac{\mu}{\rho v_{\infty} \delta} \right)^{1/4} = \frac{0.0576}{(\text{Re}_x)^{1/5}}$$

Summary

Laminar B.L.

$$\text{Re}_{cr} = 3.2 \times 10^5 \quad \Rightarrow \quad x_{cr} = 3.2 \times 10^5 \frac{\mu}{\rho v_\infty} \quad \text{Re}_x \leq \text{Re}_{cr} \quad \Rightarrow \quad x \leq x_{cr}$$

$$\frac{v_x}{v_\infty} = f(\zeta), \quad \zeta = \frac{y}{\delta(x)} \quad C_f = \frac{\tau_w}{\frac{1}{2} \rho v_\infty^2}$$

$f(\zeta)$	$\frac{\delta}{x}$	C_f
Exact Solution	Numerical See Fig 8.5	$\frac{0.664}{\sqrt{\text{Re}_x}}$
$\sin\left(\frac{\pi}{2} \zeta\right)$	$\frac{4.79}{\sqrt{\text{Re}_x}}$	$\frac{0.656}{\sqrt{\text{Re}_x}}$
ζ	Do it yourself !	$\frac{0.578}{\sqrt{\text{Re}_x}}$
$2\zeta - \zeta^2$	Do it yourself !	$\frac{0.730}{\sqrt{\text{Re}_x}}$
$\frac{3}{2}\zeta - \frac{1}{2}\zeta^3$	Do it yourself !	$\frac{0.646}{\sqrt{\text{Re}_x}}$
$2\zeta - 2\zeta^3 + \zeta^4$	Do it yourself !	$\frac{0.686}{\sqrt{\text{Re}_x}}$

Turbulent B.L.

$$\text{Re}_L > \text{Re}_{cr} \quad C_f = \frac{\tau_w}{\frac{1}{2} \rho v_\infty^2} = 0.045 \left(\frac{\mu}{\rho v_\infty \delta} \right)^{\frac{1}{4}}$$

$$\frac{\delta}{x} = \frac{0.376}{(\text{Re}_x)^{1/5}} \quad \text{equation (8.61 of textbook)}$$

$$C_f \equiv \frac{\tau_w}{\frac{1}{2} \rho v_\infty^2} = \frac{0.0576}{(\text{Re}_x)^{1/5}} \quad \text{equation (8.62 of textbook)}$$

Calculation of Drag Force Over a Flat Plate

Drag Force: $F_D = A \tau_w$

However, $A = f(x)$ and $\tau_w = f(x)$

$$\Rightarrow F_D = \int \tau_w dA; \quad dA = dx \text{ (Area per unit width)}$$

$$\Rightarrow F_D = \int_0^L \tau_w dx \quad \text{(Drag Force per unit width)}$$

$$x_{cr} = 3.2 \times 10^5 \frac{\mu}{\rho v_\infty} \quad \Rightarrow F_D = \int_0^{x_{cr}} (\tau_w)_{\text{Laminar}} dx + \int_{x_{cr}}^L (\tau_w)_{\text{Turbulent}} dx$$

$$F_D = \left\{ \int_0^{x_{cr}} (\tau_w)_{\text{Laminar}} dx + \int_{x_{cr}}^L (\tau_w)_{\text{Turbulent}} dx \right\} \times \frac{\frac{1}{2} \rho v_\infty^2}{\frac{1}{2} \rho v_\infty^2} = \frac{1}{2} \rho v_\infty^2 \left[\int_0^{x_{cr}} (C_f)_{\text{Laminar}} dx + \int_{x_{cr}}^L (C_f)_{\text{Turbulent}} dx \right]$$

$$= \frac{1}{2} \rho v_\infty^2 \left[\int_0^{x_{cr}} \frac{0.656}{\sqrt{\text{Re}_x}} dx + \int_{x_{cr}}^L \frac{0.0576}{(\text{Re}_x)^{1/5}} dx \right] = \frac{1}{2} \rho v_\infty^2 \left[\frac{0.656}{\sqrt{\frac{\rho v_\infty}{\mu}}} \int_0^{x_{cr}} x^{-\frac{1}{2}} dx + \frac{0.0576}{\left(\frac{\rho v_\infty}{\mu}\right)^{1/5}} \int_{x_{cr}}^L x^{-\frac{1}{5}} dx \right]$$

$$F_D = \frac{1}{2} \rho v_\infty^2 \left[\frac{0.656}{\sqrt{\frac{\rho v_\infty}{\mu}}} (2) \left(x_{cr}^{\frac{1}{2}} \right) + \frac{0.0576}{\left(\frac{\rho v_\infty}{\mu}\right)^{1/5}} \left(\frac{5}{4} \right) \left(L^{\frac{4}{5}} - x_{cr}^{\frac{4}{5}} \right) \right]$$

$$F_D = \frac{1}{2} \rho v_\infty^2 \left[0.656(2) \frac{x_{cr}}{\sqrt{\text{Re}_{cr}}} + 0.0576 \left(\frac{5}{4} \right) \left(\frac{L}{(\text{Re}_L)^{1/5}} - \frac{x_{cr}}{(\text{Re}_{cr})^{1/5}} \right) \right]$$

Note: the above analysis for calculation of the drag force per unit width is done for the case when $L > x_{cr}$ ($\text{Re}_L > \text{Re}_{cr}$). However, for the case when $L < x_{cr}$ ($\text{Re}_L < \text{Re}_{cr}$) the flow over the entire length of the plate is laminar and the drag force per unit width:

$$F_D = \int_0^L (\tau_w)_{\text{Laminar}} dx = \frac{1}{2} \rho v_\infty^2 \int_0^L (C_f)_{\text{Laminar}} dx = \frac{1}{2} \rho v_\infty^2 \left[\int_0^L \frac{0.656}{\sqrt{\text{Re}_x}} dx \right] = \frac{1}{2} \rho v_\infty^2 \left[0.656(2) \frac{L}{\sqrt{\text{Re}_L}} \right]$$

Example

Air ($\rho_{\text{air}} = 0.075 \text{ lb}_m/\text{ft}^3$, $\mu_{\text{air}} = 1.35 \times 10^{-5} \text{ lb}_m/\text{ft s}$) flows over a flat plate with a velocity 50 ft/s. Calculate the drag force over the flat plate if:

- (a) The plate is 1 ft long along the direction of flow.
- (b) The plate is 5 ft long along the direction of flow.

Solution

$$(a) \quad \text{Re}_L = \frac{\rho v_\infty L}{\mu} = \frac{(0.075)(50)(1)}{1.35 \times 10^{-5}} = 2.8 \times 10^5 < 3.2 \times 10^5$$

In this case, the flow over the entire plate is laminar:

$$\begin{aligned} F_D &= \frac{1}{2} \rho v_\infty^2 \left[0.656(2) \frac{L}{\sqrt{\text{Re}_L}} \right] = \frac{1}{2} \times 0.075 \times 50^2 \left[0.656(2) \frac{1}{\sqrt{2.8 \times 10^5}} \right] \\ &= 0.23 \frac{\text{lb}_m \text{ ft}}{\text{s}^2} \frac{1}{\text{ft}} = 7.2 \times 10^{-3} \frac{\text{lb}_f}{\text{ft}} \end{aligned}$$

$$(b) \quad \text{Re}_L = \frac{\rho v_\infty L}{\mu} = \frac{(0.075)(50)(5)}{1.8 \times 10^{-4}(0.075)} = 1.39 \times 10^6 > 3.2 \times 10^5$$

In this case, the flow undergoes transition from laminar to turbulent. The transition takes place at:

$$x_{\text{cr}} = 3.2 \times 10^5 \frac{\mu}{\rho v_\infty} = 3.2 \times 10^5 \frac{1.18 \times 10^{-4}}{50} = 1.152 \text{ ft} < 5 \text{ ft}$$

$$F_D = \frac{1}{2} \rho v_\infty^2 \left[0.656(2) \frac{x_{\text{cr}}}{\sqrt{\text{Re}_{\text{cr}}}} + 0.0576 \left(\frac{5}{4} \right) \left(\frac{L}{(\text{Re}_L)^{1/5}} - \frac{x_{\text{cr}}}{\sqrt{(\text{Re}_{\text{cr}})^{1/5}}} \right) \right]$$

$$F_D = \frac{1}{2} (1.8 \times 10^{-4} \times 0.075) 50^2 \left[0.656(2) \frac{1.152}{\sqrt{3.2 \times 10^5}} + 0.0576 \left(\frac{5}{4} \right) \left(\frac{5}{(1.39 \times 10^6)^{1/5}} - \frac{1.152}{(3.2 \times 10^5)^{1/5}} \right) \right]$$

$$F_D = 2.53 \times 10^{-4} \frac{\text{lb}_m \text{ ft}}{\text{s}^2} \frac{1}{\text{ft}} = 7.86 \times 10^{-6} \frac{\text{lb}_f}{\text{ft}}$$