

A simple model for turbulent boundary layer mass transfer on flat plate in parallel flow

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Abstract

A simple model for turbulent mass transfer from a flat plate valid for wider range of Schmidt numbers is proposed. Based on $1/n$ power law velocity/concentration profiles and integral momentum/mass equations, the model reveals that λ_c , a parameter in universal concentration profile, is a function of Sc . Through an empirical approach, it is given as $\lambda_c = 8.55 Sc^{0.37}$. With this function, the model matches with experimental data up to $Sc = 108$. © 2002 Elsevier Science B.V. All rights reserved.

Keywords: Turbulent mass transfer; Schmidt number; Momentum/mass equations

1. Introduction

The objective of this communication is to present a simplified semi-empirical approach to account for the mass transfer coefficient for the case of turbulent boundary layer formed over a soluble flat plate held parallel to the flow valid for a wide range of Sc . In laminar range, experimental data have been found to be in agreement with theoretical predictions based on integral boundary layer equation [1]. For turbulent flows, the analysis and the predictions are yet not conclusive because of the absence of a universal turbulent model. Correlations were proposed based on Chilton–Colburn analogy [2] and later corrected by Rubesin's starting length correction factor [3]. Several workers correlated turbulent heat transfer data using velocity and temperature profiles based on $1/n$ power law. Reynolds [4] showed that at $n = 7$, heat transfer data had better agreement. Interest in $1/n$ power law has been revived with extended range up to Reynolds numbers of 10 000 000 [5–7]. Mass transfer coefficient can be obtained through $1/n$ power law and the integral equation with the assumption of identical hydrodynamic and concentration boundary layer thickness, limiting the solution to $Sc = 1$ [8]. Recently Nassif et al. [9] used $1/n$ power law to correlate average mass transfer data for a naphthalene plate subliming in parallel flow of air. With the introduction of exponent on Schmidt number and relating the universal

velocity constants to Reynolds number, they could extend the validity of their correlation up to $Sc = 2.5$.

In the present work, a more general correlation valid for a wider range of Schmidt numbers is proposed based on the mass transfer integral equation. The constant λ_c , in $1/n$ concentration profile, is a function of λ and Sc where λ as defined in Eq. (1), is taken, as discussed elsewhere, to be a constant. A relation between λ_c and Sc has been obtained by correlating available heat and mass transfer data with the model.

2. Analysis and discussion

Consider a horizontal flat plate oriented in the direction of an incompressible, Newtonian and steady-state flow as shown in Fig. 1. Convective solid–fluid mass transfer is taking place at the plate surface after an initial inert length x_0 . The universal velocity profile in the hydrodynamic boundary layer can be written as

$$u^+ = \lambda[y^+]^{1/n} \quad (1)$$

where

$$u^+ = \frac{u_x}{u^*}, \quad u^* = \sqrt{\frac{\tau_w}{\rho}}, \quad y^+ = \frac{u^* y}{\nu} \quad (2)$$

At $y = \delta$, the velocity equals the free stream velocity U . With this boundary condition λ is eliminated from Eq. (1) to give

$$u_x = U \left(\frac{y}{\delta} \right)^{1/n} \quad (3)$$

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Nomenclature

B, D and E	terms defined in Eqs. (14), (17) and (18)
C_A	concentration of component A
$C_{A\infty}$	bulk concentration of the free stream
C_{A0}	surface concentration of the plate
C'_A	$(C_A - C_{A0})$
C'^+_A	dimensionless concentration given by Eq. (11)
C_p	heat capacity
D_{AB}	binary diffusion coefficient
h_x	local heat transfer coefficient
k	thermal conductivity
k_x, k_L	local and average mass transfer coefficients
L	length of the plate
n	exponent in the power law
N_{A0}	mass transfer flux
Nu_x	local Nusselt number, $h_x x/k$
Pr	Prandtl number, $C_p \mu/k$
Re_x, Re_L	Reynolds number based on x and L , Ux/ν and UL/ν
Sc	Schmidt number, ν/D_{AB}
Sc_t	turbulent Schmidt number
Sh_L	average Sherwood number, $k_L L/D_{AB}$
St_x, St_L	local and average Stanton numbers, k_x/U , k_L/U
u_x	local velocity in the boundary layer
u^+	dimensionless velocity given by Eq. (2)
U	free stream velocity
x	axial distance
x_0	starting inert length
y	axial distance
y^+	dimensionless distance for hydrodynamic analysis, yu^*/ν
y^+_c	dimensionless distance for mass transfer analysis, yu^*/D_{AB}
Greek letters	
α	parameter defined in Eq. (24)
δ	hydrodynamic boundary layer thickness
δ_c	concentration boundary layer thickness
ε_c	eddy concentration diffusivity
ε_m	eddy momentum diffusivity
λ	constant in universal velocity profile
λ_c	constant in universal concentration profile
μ	fluid viscosity
ν	kinematic viscosity, μ/ρ
ρ	fluid density
τ_w	shear stress at wall

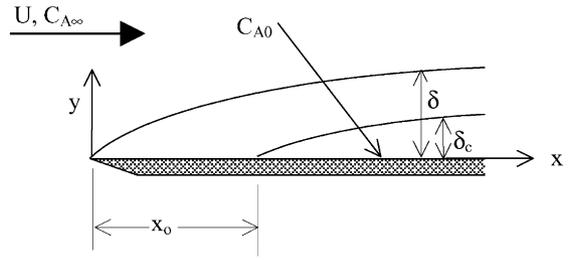


Fig. 1. Schematic diagram of momentum and concentration boundary layers.

The boundary layer thickness δ can be calculated by solving the momentum integral equation

$$\frac{d}{dx} \int_0^\delta (U - u_x) u_x dy = \frac{\tau_w}{\rho} \quad (6)$$

together with Eq. (3) and the boundary condition at $x = 0$, $\delta = 0$. Thus, δ can be expressed as

$$\delta = \lambda^{(-2n/(n+3))} \left[\frac{(n+3)(n+2)x}{n} \right]^{((n+1)/(n+3))} \times \left[\frac{\nu}{U} \right]^{(2/(n+3))} \quad (7)$$

It should be noted that for the widely used values for $n = 7$ and $\lambda = 8.74$, Eq. (7) yields,

$$\delta = 0.371x Re_x^{-0.2} \quad (8)$$

which is in agreement with

$$\delta = 0.376x Re_x^{-0.2} \quad (9)$$

Eq. (9) was obtained from experimental data of Schultz-Grunow [10] (as quoted in Skelland [8]). Similar to velocity, a concentration profile is assumed

$$C'^+_A = \lambda_c [y^+_c]^{1/n} \quad (10)$$

where the dimensionless concentration C'^+_A is defined as

$$C'^+_A = \frac{C'_A}{C'^*_A} = \frac{(C_A - C_{A0})u^*}{N_{A0}} \quad \text{using } C'^*_A = \frac{N_{A0}}{u^*} \quad (11)$$

A corresponding dimensionless distance y^+_c is defined as $y^+_c = (yu^*/D_{AB})$. At the edge of the boundary layer (i.e., $y = \delta_c$), concentration is $C_{A\infty}$. This condition with Eq. (10) gives $1/n$ power law profile for concentration.

$$\frac{C'_A}{C'_{A\infty}} = \left(\frac{y}{\delta_c} \right)^{1/n} \quad (12)$$

Local Stanton number, St_{AB_x} can be obtained through Eqs. (4), (7), (10)–(12):

$$\frac{N_{A0}}{UC'_{A\infty}} = \frac{k_x}{U} = St_{AB_x} = Bx^{((1-n)/n(n+3))} \delta_c^{-1/n} \quad (13)$$

$$u^* = U \lambda^{(-n/(n+1))} \left[\frac{\nu}{\delta U} \right]^{(1/(n+1))} \quad (4)$$

$$\tau_w = \rho U^2 \lambda^{(-2n/(n+1))} \left[\frac{\nu}{\delta U} \right]^{(2/(n+1))} \quad (5)$$

where

$$B = \lambda^{((1-n)/(n+3))} \lambda_c^{-1} \left[\frac{(n+3)(n+2)}{n} \right]^{((1-n)/n(n+3))} \times \left[\frac{\nu}{U} \right]^{((n-1)/n(n+3))} \left[\frac{D_{AB}}{U} \right]^{1/n} \quad (14)$$

The integral equation for mass transfer is

$$\frac{d}{dx} \int_0^{\delta_c} (C'_{A\infty} - C'_A) u_x dy = N_{A0} \quad (15)$$

Eqs. (3), (12) and (15) with boundary condition, at $x = x_0$, $\delta_c = 0$, yield

$$\delta_c = x^{((n+1)/(n+3))} \left[\frac{E(3+n)}{1-3D+n-nD} \right]^{(n/(n+2))} \times \left[1 - \left(\frac{x_0}{x} \right)^{((2+n)(1-3D+n-nD)/n(n+3))} \right]^{(n/(n+2))} \quad (16)$$

Here

$$D = \frac{n}{3n+n^2} \quad (17)$$

and

$$E = (n+2)\lambda^{(-(n+1)/(n+3))} \lambda_c^{-1} \left(\frac{\nu}{U} \right)^{((n+1)/n(n+3))} \times \left(\frac{D_{AB}}{U} \right)^{1/n} \left(\frac{(n+2)(n+3)}{n} \right)^{(2/n(n+3))} \quad (18)$$

Final expression for St_{AB_x} can be obtained by substituting δ_c from Eq. (16) into Eq. (13).

$$St_{AB_x} = \lambda^{((3-n^2)/(n+2)(n+3))} \lambda_c^{(-(n+1)/(n+2))} Sc^{(-(n+1)/n(n+2))} \times Re_x^{(-2/(n+3))} \left[\frac{(n+3)(n+2)}{1-3D+n-nD} \right]^{(-1/(n+2))} \times \left[\frac{(n+3)(n+2)}{n} \right]^{(-(n+1)/(n+2)(n+3))} \times \left[1 - \left(\frac{x_0}{x} \right)^{((2+n)(1-3D+n-nD)/n(n+3))} \right]^{(-1/(n+2))} \quad (19)$$

In the present analysis, the values of n and λ were taken as 7 and 8.74, respectively, which are the most widely used values for fully developed turbulent flows [11]. Therefore, Eq. (19) becomes

$$St_{AB_x} = 0.1982 \lambda_c^{-0.889} Sc^{-0.12698} Re_x^{-0.2} \left[1 - \left(\frac{x_0}{x} \right)^{0.9} \right]^{-1/9} \quad (20)$$

The average Sh is obtained by following integration using IMSL subroutine QDAGS

$$Sh_L = \frac{Sc Re_L}{(L-x_0)} \int_{x_0}^L St_{AB_x} dx. \quad (21)$$

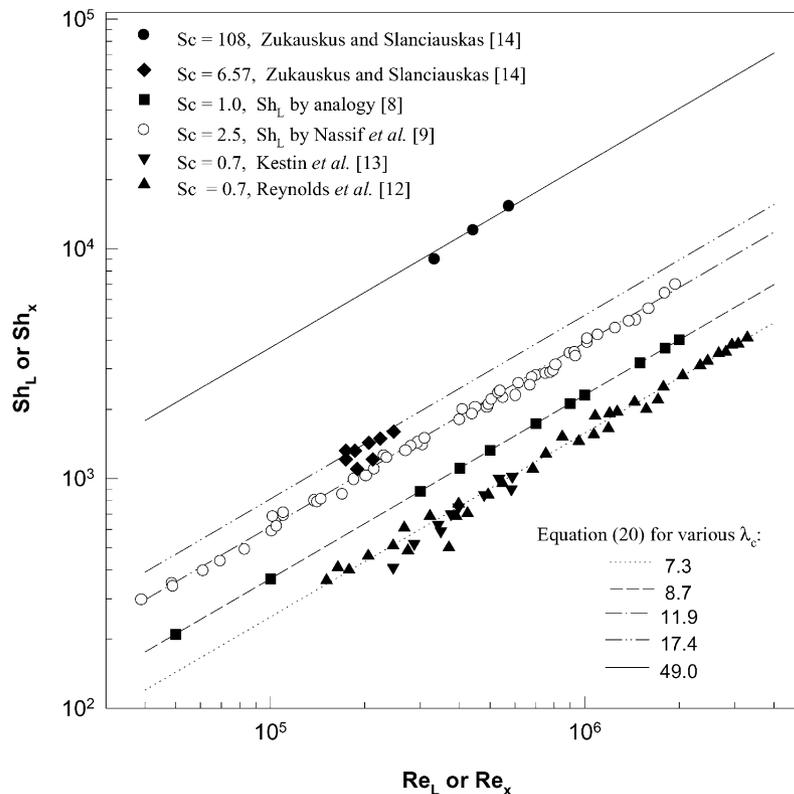


Fig. 2. Comparison of experimental data with Eq. (20). See Reynolds et al. [12], Kestin et al. [13], Zukauskus and Slanciauskas [14].

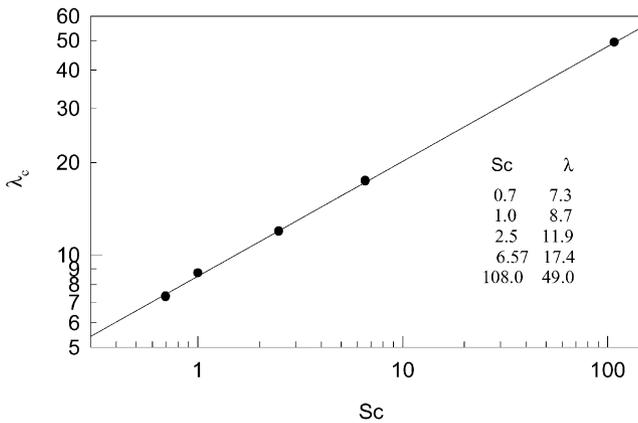


Fig. 3. Plot of parameter λ_c versus Sc .

At this point, an explicit expression for λ_c is required, which if obtained would render the whole analysis simple. Relevant data on flat plate (including those for heat transfer) at different values of Sc (or Pr) were matched with the present analysis using Eq. (20) or its averaged form. The matched plots are shown in Fig. 2. It should be noted that value of λ_c was adjusted till a good agreement with the experimental data was established. Thus, different adjusted values of λ_c were obtained for corresponding Sc . The desired expression for λ_c is obtained by linear regression of adjusted λ_c and Sc . The data as shown in Fig. 3, displays linearity and, therefore, can be easily extrapolated. The resulting equation is

$$\lambda_c = 8.55Sc^{0.37} \quad (22)$$

Dividing Eq. (22) by $\lambda = 8.74$ yields,

$$\frac{\lambda_c}{\lambda} = 0.98Sc^{0.37} \quad (23)$$

Eq. (23) is the equivalent of Eq. (22), but is more meaningful in its present format. If the constant in Eq. (23) is approximated to unity, the ratio λ_c/λ becomes purely a function of Sc , a ratio of momentum and mass diffusivities. In principle, this ratio should be a function of turbulent Schmidt number ($Sc_t = \varepsilon_m/\varepsilon_c$). It should be noted that Eq. (23) qualitatively matches the equation quoted by Nassif et al. [9] who have postulated that

$$Sc^\alpha = \frac{\nu + \varepsilon_m}{D_{AB} + \varepsilon_c} \quad (24)$$

where α in the above equation covers the contribution due to eddy diffusivities. The authors obtained the values of α and λ for $n = 7$ by comparing their analysis with an empirical heat transfer equation [15] valid for $0.5 < Pr < 1.0$. Later, α and λ were generalized, through a very empirical approach, to be functions of n which itself is taken as function of Re_L . Therefore, the α and λ depend on hydrodynamics only which is at variance with the assumption (implicit in Eq. (23)) that α depends also on ε_c . However, their analysis holds only for Sc between 1 and 2.5. At any other Sc widely off the limits of unity, a mismatch is expected. The predictions from

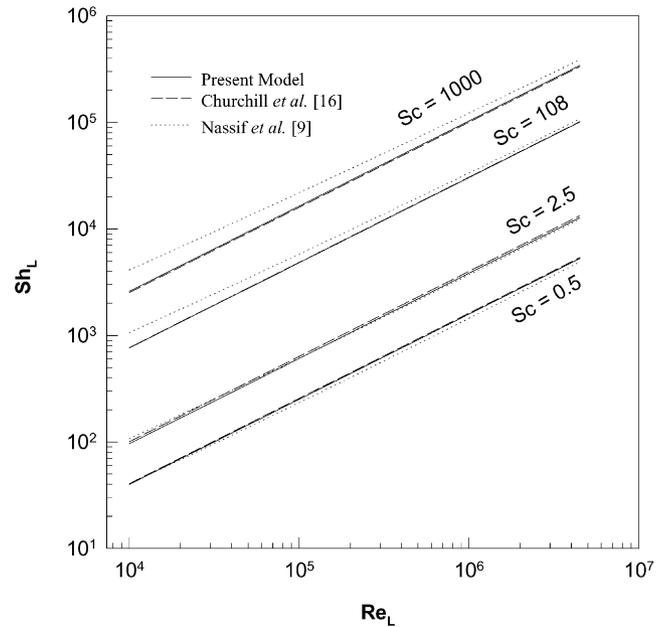


Fig. 4. Comparison of models at various values of Sc .

the present model are compared with the models of Nassif et al. [9] and Churchill [16] in Fig. 4. The model of Nassif et al. expectedly overpredicts at higher values of Sc , while the present model is in coherence with Churchill's model. In addition, it is proposed that the present model would cover the predictions for a much wider range of Sc number. This is based on intuitive arguments that the transfer mechanism remains invariant even at high Sc . This implies that linear dependence of λ with $Sc^{0.37}$ is sustained to give an extended validity of the present model.

3. Conclusion

A simple semi-empirical model to predict mass transfer coefficient for fully developed boundary layer turbulent flow over a flat surface is proposed. For the widely used values of $n = 7$ and $\lambda = 8.74$, the model incorporates an empirical expression for λ_c , which together with integral equations and $1/n$ power law velocity and concentration profiles, calculates Sherwood numbers. Expression for λ_c was obtained by matching experimental data for a range of Sc from 0.7 to 108. The validity of this function can be extended by comparing with data at higher Sc . Experiments are in progress using electrochemical limiting diffusion current technique which will provide data for Sc of the order of 2000.

Acknowledgements

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