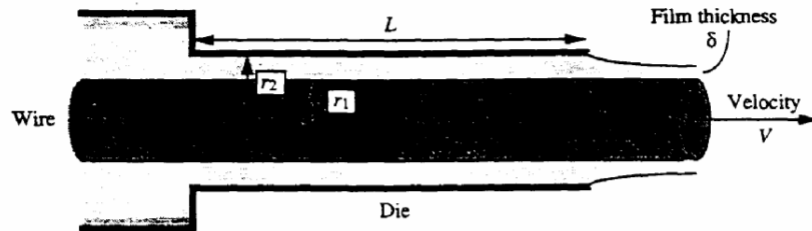


6.2-1

Wire Coating



Z-Momentum Balance (cylindrical coordinates)

$$\rho \frac{Dv_z}{Dt} = \rho \left(\frac{\partial v_z}{\partial t} + v_r \frac{\partial v_z}{\partial r} + \frac{v_\theta}{r} \frac{\partial v_z}{\partial \theta} + v_z \frac{\partial v_z}{\partial z} \right)$$

$$= -\frac{\partial p}{\partial z} + \mu \left[\frac{1}{r} \frac{\partial}{\partial r} \left(r \frac{\partial v_z}{\partial r} \right) + \frac{1}{r^2} \frac{\partial^2 v_z}{\partial \theta^2} + \frac{\partial^2 v_z}{\partial z^2} \right] + \rho g_z$$

Assumptions

1. Steady state
2. $v_z \neq 0$; $v_r = v_\theta = 0$; $v_z = v_z(r)$
3. $g_z = 0$ (g is unimportant)
4. p constant

Simplified Momentum Balance

$$\frac{d}{dt} + \frac{dv_z}{dt} = 0$$

Integrate Twice

$$r \frac{dv_z}{dt} = c_1, \quad v_z = c_1 \ln r + c_2$$

6.2-2

Boundary Conditions

$$\left. \begin{array}{l} r = r_1: v_z = V \\ r = r_2: v_z = 0 \end{array} \right\} \text{Hence}$$

$$C_1 = -\frac{V}{\ln(r_2/r_1)}$$

$$C_2 = \frac{V \ln r_2}{\ln(r_2/r_1)}$$

Velocity Profile

$$v_z = V \frac{\ln(r_2/r)}{\ln(r_2/r_1)} = V \frac{\ln(r/r_1)}{\ln(r_1/r_2)}$$

Volumetric Flow Rate

$$Q = \int_{r_1}^{r_2} 2\pi r dr \frac{V}{\ln(r_1/r_2)} (\ln r - \ln r_2)$$

Note: $\int r \ln r dr$

$$= \frac{r^2}{2} \ln r - \frac{r^2}{4}$$

$$= \frac{2\pi V}{\ln(r_1/r_2)} \left[\frac{r^2}{2} \ln r - \frac{r^2}{4} - \frac{r^2}{2} \ln r_2 \right]_{r_1}^{r_2}$$

$$Q = \frac{2\pi V}{\ln(r_2/r_1)} \left[\frac{1}{4} (r_2^2 - r_1^2) - \frac{r_1^2}{2} \ln \frac{r_2}{r_1} \right]$$

Coeating Thickness

$$\begin{aligned} Q &= V [\pi (r_1 + \delta)^2 - \pi r_1^2] \\ &= V \pi (\delta^2 + 2r_1 \delta) \end{aligned}$$

Force to Pull Rod

$$F = 2\pi r_1 L (-\tau_{rz})_{r=r_1}$$

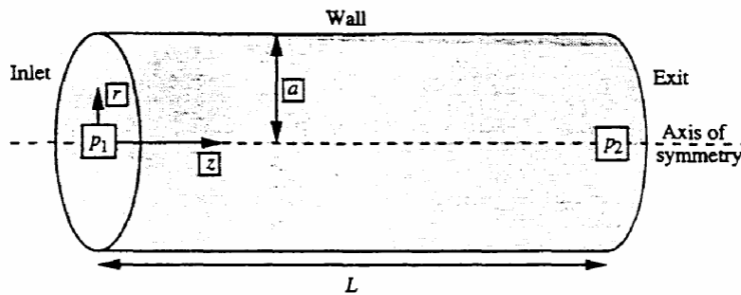
$$\tau_{rz} = \mu \left(\frac{\partial v_z}{\partial r} + \frac{\partial v_r}{\partial z} \right)$$

$$F = \frac{2\pi \mu L V}{\ln(r_2/r_1)}$$

$$= -\frac{1}{r} \frac{\mu V}{\ln(r_2/r_1)}$$

6.7-1

Details of Pipe Flow



Assumptions

1. $v_z \neq 0$; $v_r = v_\theta = 0$
2. Steady, constant ρ and μ .
3. Symmetry about z axis: $\frac{\partial v_z}{\partial \theta} = 0$

Continuity Equation

$$\frac{\partial \rho}{\partial t} + \frac{1}{r} \frac{\partial (\rho v_r)}{\partial r} + \frac{1}{r} \frac{\partial (\rho v_\theta)}{\partial \theta} + \frac{\partial (\rho v_z)}{\partial z} = 0$$

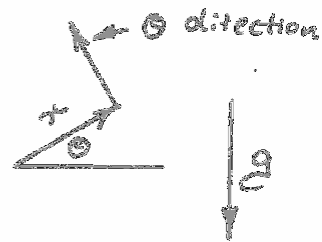
Simplifies to $\frac{\partial v_z}{\partial z} = 0$ Hence $v_z = v_z(t)$

Three Momentum Balances (after simplification)

$$\frac{\partial p}{\partial t} = \rho g_t = \rho (-g \sin \theta)$$

$$\frac{1}{r} \frac{\partial p}{\partial \theta} = \rho g_\theta = \rho (-g \cos \theta)$$

$$\frac{\partial p}{\partial z} = \mu \frac{1}{r} \frac{d}{dt} \left(r \frac{dv_z}{dr} \right)$$



6.7-2

Integrate r and θ Momentum Balances

$$\left. \begin{aligned} p &= -\rho g r \sin \theta + f_1(z, \theta) \\ p &= -\rho g r \sin \theta + f_2(z, t) \end{aligned} \right\} \text{Hence } f_1(z, \theta) = f_2(z, t) = f(z) \text{ only}$$

Pressure Variation in z direction

Assume (to be verified) a linear decline from p_1 to p_2

Hence

$$p = p_1 - \frac{z}{L}(p_1 - p_2) - \rho g r \sin \theta$$

Integrate z -Momentum Balance Once

$$r \frac{dv_z}{dt} = -\frac{1}{2\mu} \left(\frac{-\partial p}{\partial z} \right) r^2 + c_1$$

Integrate Again

$$v_z = -\frac{1}{4\mu} \left(\frac{-\partial p}{\partial z} \right) r^2 + c_1 \ln r + c_2$$

Determination of Constants of Integration

c_1 must be zero, to avoid physically impossible infinite velocity at center from $c_1 \ln r$ term

or Use symmetry that $\frac{dv_z}{dt} = 0$ at $r = 0$

Will give the same result

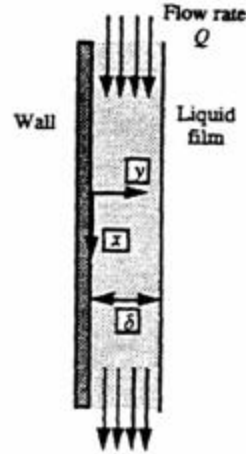
At the wall, $r = a$:

$$0 = -\frac{1}{4\mu} \left(\frac{-\partial p}{\partial z} \right) a^2 + c_2 \quad \text{or } c_2 = \frac{1}{4\mu} \left(\frac{-\partial p}{\partial z} \right) a^2$$

6.14-1
Wetted-Wall Column

Assume

1. Steady, constant ρ, μ .
2. $v_x \neq 0; v_y = v_z = 0$
3. $\frac{\partial v_x}{\partial z} = 0$ 4. $g_x = g$
 $g_y = 0$



Continuity (ρ constant)

$$\nabla \cdot \vec{v} = \frac{\partial v_x}{\partial x} + \frac{\partial v_y}{\partial y} + \frac{\partial v_z}{\partial z} = 0$$

Hence $\frac{\partial v_x}{\partial x} = 0$ and so $v_x = v_x(y)$

Simplified Momentum Balances

$$x: \frac{\partial p}{\partial x} = \mu \frac{d^2 v_x}{dy^2} + \rho g_x$$

y: $\frac{\partial p}{\partial y} = 0$ That is, there is no variation of pressure horizontally across the film. Since the pressure outside the film

is everywhere atmospheric ($p=0$, say), the pressure everywhere inside the film must be $p=0$ also

Hence $\partial p / \partial x = 0$ and the x-momentum balance is

$$\frac{d^2 v_x}{dy^2} = - \frac{\rho g}{\mu}$$