
Chapter 4

Unsteady State Conduction



Steady State Conduction



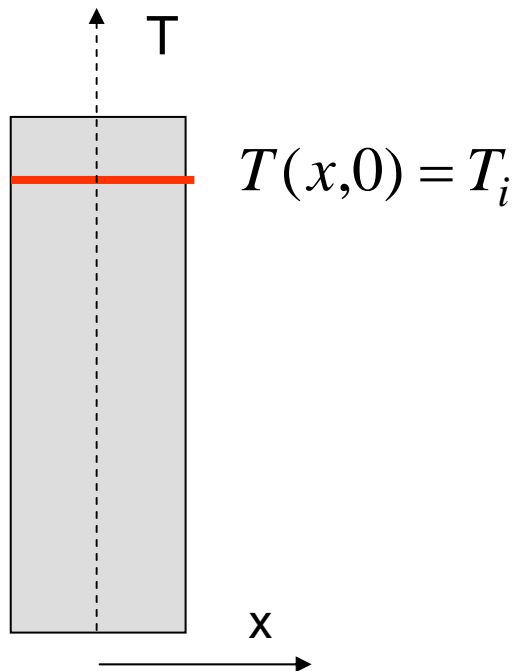
4-1 Introduction

Transient Conduction

- Many heat transfer problems are time dependent
- Changes in operating conditions in a system cause temperature variation with time, as well as location within a solid, until a new steady state (thermal equilibrium) is obtained.
- In this chapter we will develop procedures for determining the time dependence of the temperature distribution
- Real problems may include finite and semi-infinite solids, or complex geometries, as well as two and three dimensional conduction
- Solution techniques involve the lumped capacitance method, exact and approximate solutions, and finite difference methods.
- We will focus on the Lumped Capacitance Method, which can be used for solids within which *temperature gradients are negligible*.

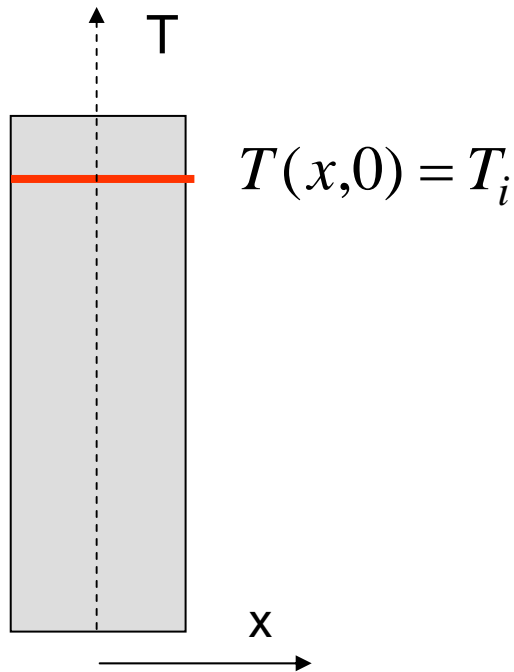
4-2 Lumped Capacitance Method

- Consider a hot metal that is initially at a uniform temperature, T_i , and at $t=0$ is quenched by immersion in a cool liquid, of lower temperature T_∞
- The temperature of the solid will decrease for time $t>0$, due to convection heat transfer at the solid-liquid interface, until it reaches T_∞



Lumped Capacitance Method

- If the thermal conductivity of the solid is very high, resistance to conduction within the solid will be small compared to resistance to heat transfer between solid and surroundings.
- Temperature gradients within the solid will be negligible, i.e.. ***the temperature of the solid is spatially uniform at any instant.***



Lumped Capacitance Method

Starting from an overall energy balance on the solid: $-\dot{E}_{out} = \dot{E}_{st}$

$$-hA_s(T - T_\infty) = \rho Vc \frac{dT}{dt}$$

The time required for the solid to reach a temperature T is:

$$t = \frac{\rho Vc}{hA_s} \ln \frac{\theta_i}{\theta}$$

where $\theta = T - T_\infty$
 $\theta_i = T_i - T_\infty$

The temperature of the solid at a specified time t is:

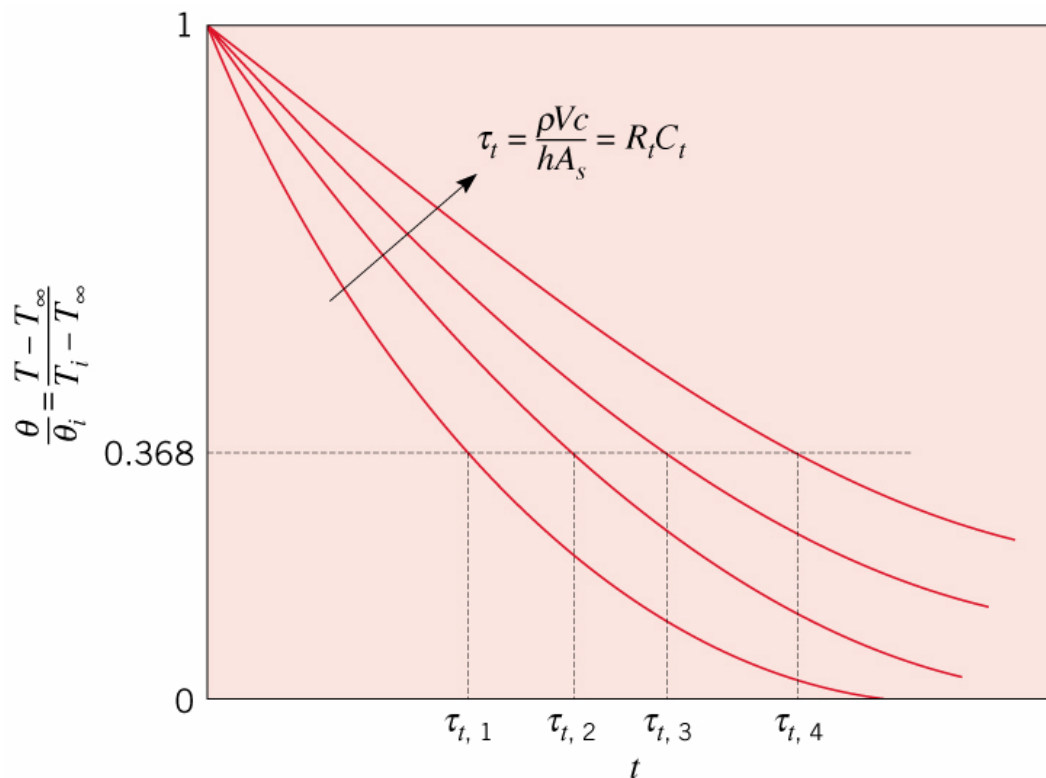
$$\frac{\theta}{\theta_i} = \frac{T - T_\infty}{T_i - T_\infty} = \exp \left[- \left(\frac{hA_s}{\rho Vc} \right) t \right] \quad (4.5)$$

The total energy transfer, Q, occurring up to some time t is:

$$Q = \int_0^t q dt = hA_s \int_0^t \theta dt = (\rho Vc) \theta_i [1 - \exp(-t / \tau_t)]$$

Transient Temperature Response

Based on eq. (4.5), the temperature difference between solid and fluid decays exponentially.



- Let's define a thermal time constant

$$\tau_t = \left(\frac{1}{h A_s} \right) (\rho V c) = R_t C_t$$

R_t is the resistance to convection heat transfer, C_t is the lumped thermal capacitance of the solid

- Increase in R_t or C_t causes solid to respond more slowly and more time will be required to reach thermal equilibrium.

Validity of Lumped Capacitance Method

- Need a suitable criterion to determine validity of method. Must relate relative magnitudes of temperature drop in the solid to the temperature difference between surface and fluid.

$$\frac{\Delta T_{solid(\text{due to conduction})}}{\Delta T_{solid/liquid(\text{due to convection})}} = \frac{(L/kA)}{(1/hA)} = \frac{R_{cond}}{R_{conv}} = \frac{hL}{k} \equiv Bi$$

- ? What should be the relative magnitude of ΔT solid versus ΔT solid/liquid for the lumped capacitance method to be valid?

Example 4-1

- A steel ball ($c=0.46$ kJ/kg. C, $k= 35$ W/m.C) 5 cm in diameter and initially at a uniform temp. of 450 C is suddenly placed in a controlled environment in which the temperature is maintained at 100 C. The convection heat transfer coefficient is 10 W/m² . C.
- Calculate the time required for the ball to attain a temperature of 150 C.

Example 4-1

We anticipate that the lumped-capacity method will apply because of the low value of h and high value of k . We can check by using Equation (4-6):

$$\frac{h(V/A)}{k} = \frac{(10)[(4/3)\pi(0.025)^3]}{4\pi(0.025)^2(35)} = 0.0023 < 0.1$$

so we may use Equation (4-5). We have

$$\begin{array}{lll} T = 150^\circ C & \rho = 7800 \text{ kg} / \text{m}^3 & [486 \text{ lb}_m / \text{ft}^3] \\ T_\infty = 100^\circ C & h = 10 \text{ W} / \text{m}^2 \cdot ^\circ C & [1.76 \text{ Btu} / \text{h} \cdot \text{ft}^2 \cdot ^\circ F] \\ T_0 = 450^\circ C & c = 460 \text{ J} / \text{kg} \cdot ^\circ C & [0.11 \text{ Btu} / \text{lb}_m \cdot ^\circ F] \end{array}$$

$$\frac{hA}{\rho c V} = \frac{(10)4\pi(0.025)2}{(7800)(460)(4\pi/3)(0.025)^3} = 3.44 \times 10^{-4} \text{ s}^{-1}$$

$$\frac{T - T_\infty}{T_0 - T_\infty} = e^{-[hA/\rho c V]\tau}$$

$$\frac{150 - 100}{450 - 100} = e^{-3.344 \times 10^{-4} \tau}$$

$$\tau = 5819 \text{ s} = 1.62 \text{ h}$$

Example 4-2

- **Semi-Infinite Solid with Sudden Change in Surface Conditions**
- A large block of steel is initially at a uniform temperature of 35°C. The surface is exposed to a heat flux (a) by suddenly raising the surface temperature to 250°C and (b) through a constant surface heat flux of W/m².
- Calculate the temperature at a depth of 2.5 cm after a time of 0.5 min for both these cases.

Example 4-2

We can make use of the solutions for the semi-infinite solid given as Equations (4-8) and (4-13a). For case a.

$$\frac{x}{2\sqrt{\varepsilon\tau}} = \frac{0.025}{(2)(1.4 \times 10^{-5})(30)^{1/2}} = 0.61$$

The error function is determined from Appendix A as

$$\operatorname{erf} \frac{x}{2\sqrt{\alpha\tau}} = \operatorname{erf} 0.61 = 0.61164$$

We have $T_i = 35^\circ C$ and $T_o = 250^\circ C$, so the temperature at $x=2.5\text{cm}$ is determined from Equation (4-8) as

$$\begin{aligned} T(x, \tau) &= T_o + (T_i - T_o) \operatorname{erf} \frac{x}{2\sqrt{\alpha\tau}} \\ &= 250 + (35 - 250)(0.61164) = 118.5^\circ C \end{aligned}$$

Example 4-2

For the constant-heat-flux case b, we make use of Equation (4-13a). Since q_o / A is given as $3.2 \times 10^5 \text{ W / m}^2$, we can insert the numerical Values to give

$$\begin{aligned} T(x, \tau) &= 35 + \frac{(2)(3.2 \times 10^5) [(1.4 \times 10^{-5})(30) / \pi]^{1/2}}{45} e^{-(0.61)} \\ &\quad - \frac{(0.025)(3.2 \times 10^5)}{45} (1 - 0.61164) \\ &= 79.3^\circ\text{C} \quad x = 2.5\text{cm}, \tau = 30\text{s} \end{aligned}$$

For the constant-heat-flux case the surface temperature after 30 s would be evaluated with $x=0$ in Equation (4-13a). Thus,

$$T(x = 0) = 35 + \frac{(2)(3.2 \times 10^5) [(1.4 \times 10^{-5})(30) / \pi]^{1/2}}{45} = 199.4^\circ\text{C}$$

Biot and Fourier Numbers

- The lumped capacitance method is valid when

$$Bi = \frac{hL_c}{k} < 0.1$$

where the characteristic length:
 $L_c = V/A_s = \text{Volume of solid/surface area}$

We can also define a “dimensionless time”, the Fourier number:

$$Fo = \frac{\alpha t}{L_c^2}$$

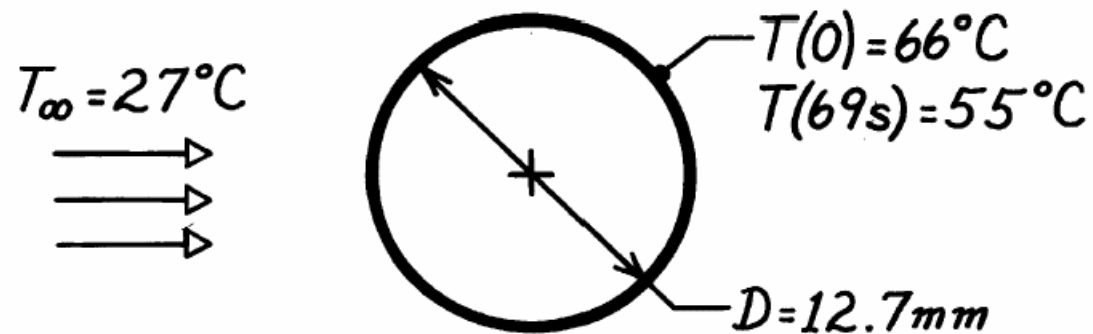
Eq. (5.2) becomes:

$$\frac{\theta}{\theta_i} = \frac{T - T_\infty}{T_i - T_\infty} = \exp[-Bi \cdot Fo] \quad (5.4)$$

Example (Problem 5.6 Textbook)

The heat transfer coefficient for air flowing over a sphere is to be determined by observing the temperature-time history of a sphere fabricated from pure copper. The sphere, which is 12.7 mm in diameter, is at 66°C before it is inserted into an air stream having a temperature of 27°C. A thermocouple on the outer surface of the sphere indicates 55°C, 69 s after the sphere is inserted in the air stream.

- Calculate the heat transfer coefficient, assuming that the sphere behaves as a spacewise isothermal object. Is your assumption reasonable?



Other transient problems

- When the lumped capacitance analysis is not valid, we must solve the partial differential equations analytically or numerically
- Exact and approximate solutions may be used
- Tabulated values of coefficients used in the solutions of these equations are available
- Transient temperature distributions for commonly encountered problems involving semi-infinite solids can be found in the literature