

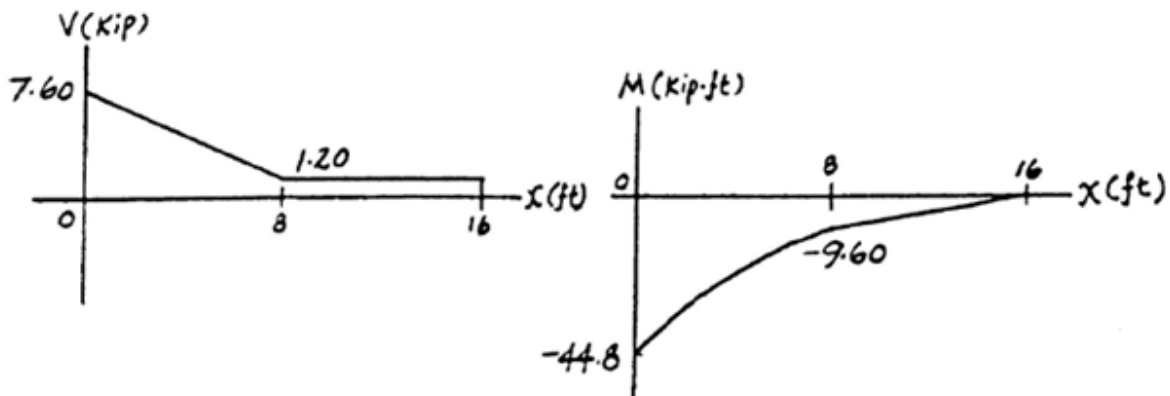
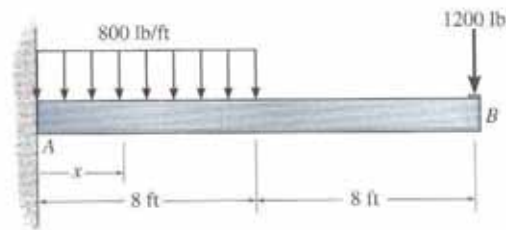
CIVIL ENGINEERING DEPARTEMENT - KFUPM  
CE 202 Statics and Strength of Materials

First Semester: '101  
Instructor: Dr. Salah Al-Dulaijan

**HOME WORK ASSIGNMENT NO. 10.**

- **Subject material covered:** Flexural Formula, shear formula, and combined loadings
- **DUE DATE:** Wednesday, January 19th, 2011.

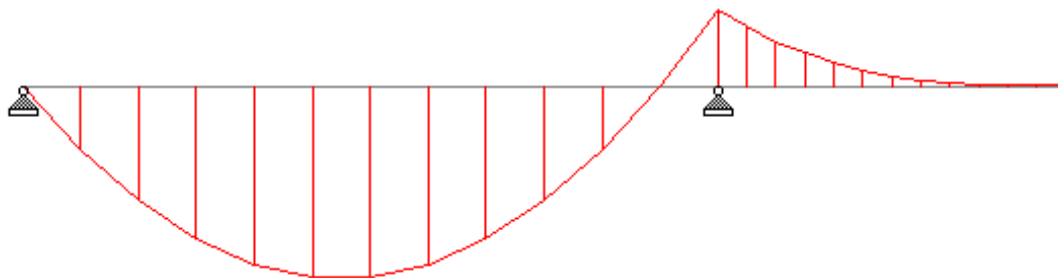
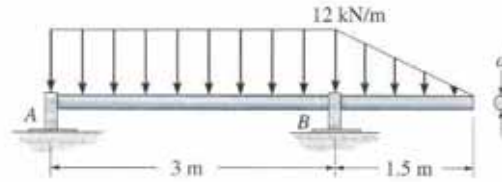
1. If the beam has a square cross-section of 9 in. on each side, determine the absolute maximum bending stress in the beam.



*Absolute Maximum bending Stress:* The maximum moment is  $M_{max} = 44.8$  kip.ft as indicated on the bending moment diagram. Applying the flexural formula

$$\sigma_{max} = \frac{M_{max} c}{I} = \frac{44.8 \times 12 \times 4.5}{1/12 \times 9 \times 9^3} = 4.42 \text{ ksi}$$

2. The rod is supported by smooth journal bearings at A and B that only exert vertical reactions on the shaft. Determine its smallest diameter  $d$  if the allowable bending stress is  $\sigma_{allow} = 180 \text{ MPa}$ .

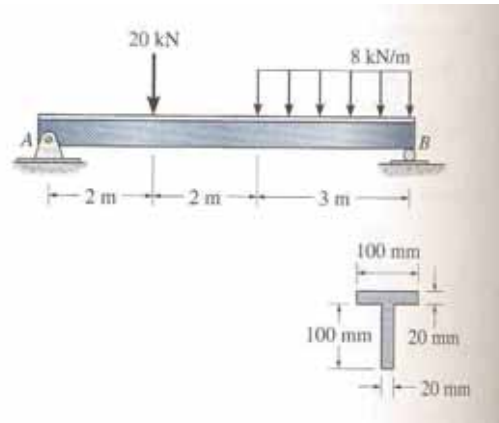


$$M_{max} = 8.37 \text{ kNm @ } 1.38\text{m}$$

$$\sigma_{allow} = \frac{M_{max}c}{I} \quad ; \quad c = r_{min} = \frac{\sigma_{allow}\pi r^4}{4M_{max}} \quad , \quad \text{or} \quad r_{min} = \left( \frac{4M_{max}}{\pi\sigma_{allow}} \right)^{\frac{1}{3}}$$

$$d_{min} = 2r_{min} = 2 \left( \frac{4 \times 8.37 \times 10^3}{\pi \times 180 \times 10^6} \right)^{\frac{1}{3}} = 2 \times 0.03898 \text{ m} \approx 78 \text{ mm}$$

3. The T-beam is subjected to the loading shown. Determine the maximum transverse shear stress in the beam at the critical section.



**Internal Shear Force:** As shown on Shear diagram,  
 $V_{max} = 24.57 \text{ kN}$ .

**Section Properties:**

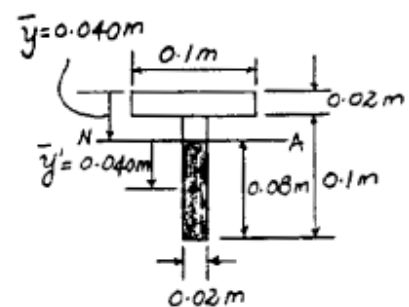
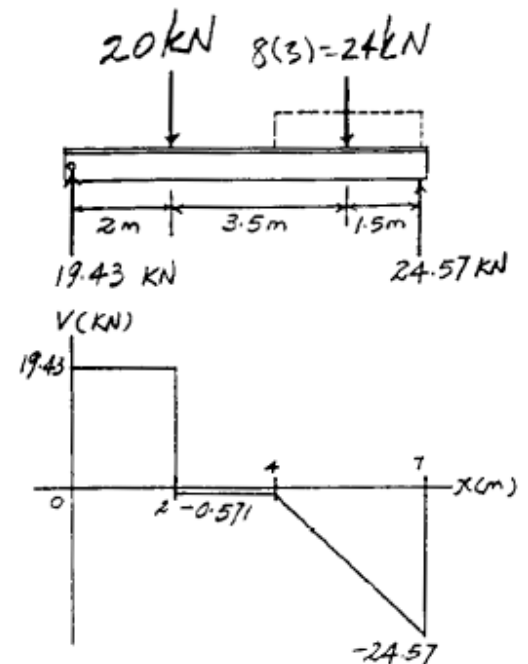
$$\bar{y} = \frac{\sum \bar{y}A}{\sum A} = \frac{0.01(0.1)(0.02) + 0.07(0.1)(0.02)}{0.1(0.02) + 0.1(0.02)} = 0.0400 \text{ m}$$

$$I_{NA} = \frac{1}{12}(0.1)(0.02^3) + 0.1(0.02)(0.0400 - 0.01)^2 + \frac{1}{12}(0.02)(0.1^3) + (0.02)(0.1)(0.07 - 0.0400)^2 = 5.3333(10^{-6}) \text{ m}^4$$

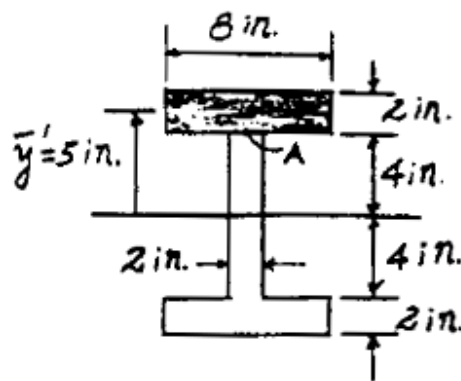
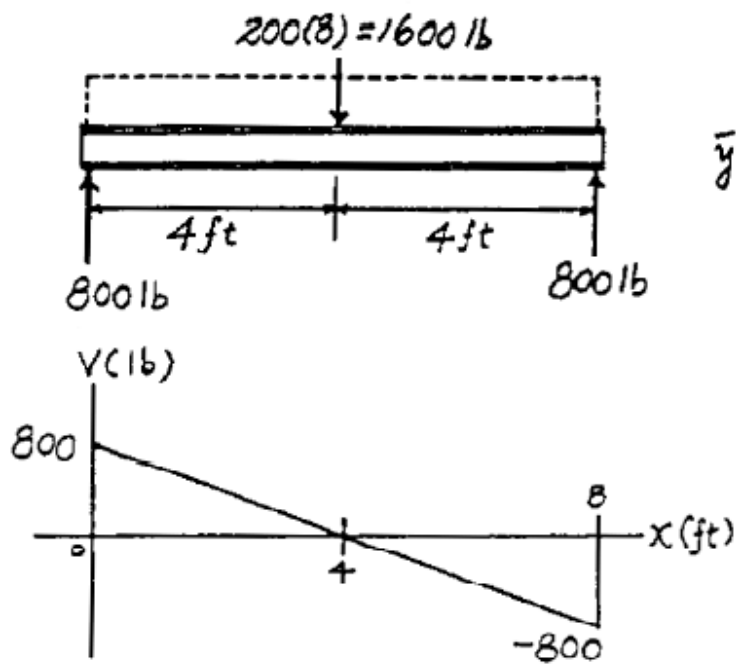
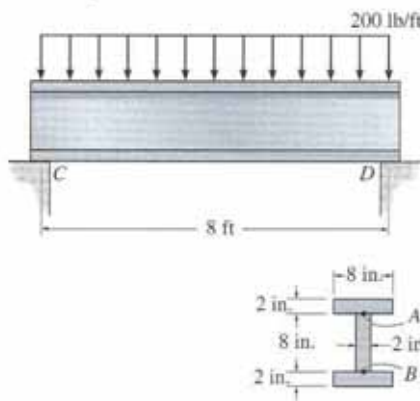
$$Q_{max} = \bar{y}'A' = 0.04(0.02)(0.08) = 64.0(10^{-6}) \text{ m}^3$$

**Maximum Shear Stress:** Maximum shear stress occurs at the point where the neutral axis passes through the section. Applying the shear formula

$$\tau_{max} = \frac{VQ_{max}}{It} = \frac{24.57(10^3) 64.0(10^{-6})}{5.3333(10^{-6})(0.02)} = 14.7 \text{ MPa}$$



4. The beam is made from three plastic pieces glued together at the seams A and B. If it is subjected to the loading shown, determine the shear stress developed in the glued joints at the critical section. The supports at C and D exert only vertical reactions on the beam.



**Support Reactions:** As shown on FBD.

**Internal Shear Force:** As shown on shear diagram,  
 $V_{\max} = 800 \text{ lb.}$

**Section Properties:**

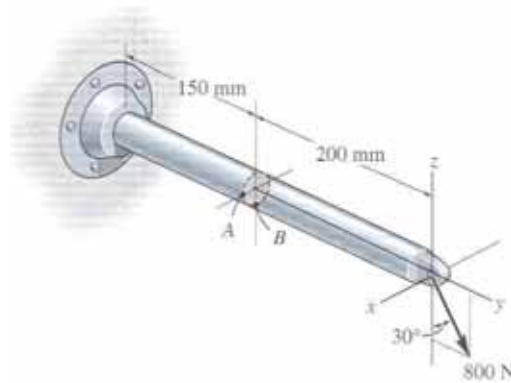
$$I_{NA} = \frac{1}{12}(8)(12^3) - \frac{1}{12}(6)(8^3) = 896 \text{ in}^4$$

$$Q_A = \bar{y}'A' = 5(8)(2) = 80.0 \text{ in}^3$$

**Shear Stress:** Applying the shear formula

$$\tau_A = \frac{VQ_A}{It} = \frac{800(80.0)}{896(2)} = 35.7 \text{ psi}$$

5. The bar has a diameter of 40 mm. If it is subjected to a force of 800 N as shown, determine the stress components that act at points A and B.



$$I = \frac{1}{4} \pi r^4 = \frac{1}{4} (\pi)(0.02^4) = 0.1256637 (10^{-6}) \text{ m}^4$$

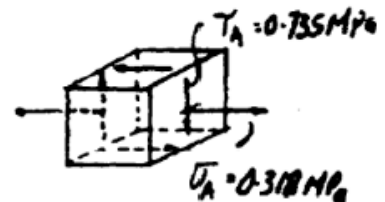
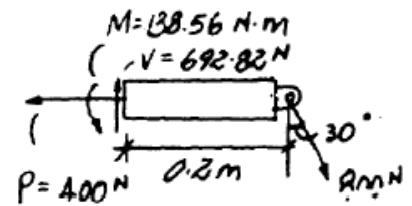
$$A = \pi r^2 = \pi (0.02^2) = 1.256637 (10^{-3}) \text{ m}^2$$

$$Q_A = \bar{y}' A' = \left( \frac{4(0.02)}{3\pi} \right) \left( \frac{\pi(0.02)^2}{2} \right) = 5.3333 (10^{-6}) \text{ m}^3$$

$$\sigma_A = \frac{P}{A} + \frac{Mz}{I}$$

$$= \frac{400}{1.256637 (10^{-3})} + 0 = 0.318 \text{ MPa}$$

$$\tau_A = \frac{VQ_A}{I t} = \frac{692.82 (5.3333) (10^{-6})}{0.1256637 (10^{-6})(0.04)} = 0.735 \text{ MPa}$$



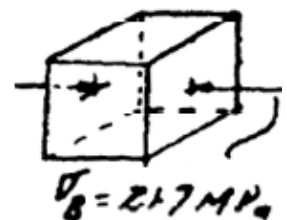
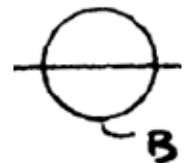
$$I = \frac{1}{4} \pi r^4 = \frac{1}{4} (\pi)(0.02^4) = 0.1256637 (10^{-6}) \text{ m}^4$$

$$A = \pi r^2 = \pi (0.02^2) = 1.256637 (10^{-3}) \text{ m}^2$$

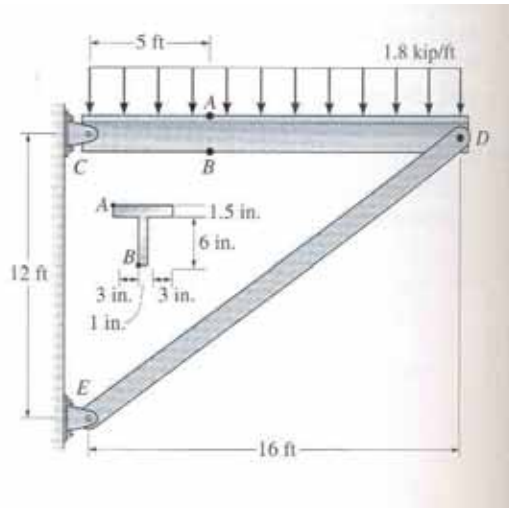
$$Q_B = 0 ; \tau_B = 0$$

$$\sigma_B = \frac{P}{A} - \frac{M c}{I} = \frac{400}{1.256637 (10^{-3})} - \frac{138.56 (0.02)}{0.1256637 (10^{-6})}$$

$$= - 21.7 \text{ MPa}$$



6. The frame supports a centrally applied distributed load of 1.8 kip/ft. Determine state of stress developed at points A and B on member CD. The pins at C and D are at the same location as the neutral axis for the cross-section.



Part	$A_i$	$y_i$	$A_i y_i$	$A_i (y_i - \bar{y})^2$	$I_{xi}$
1	10.5	0.75	7.875	19.5247934	1.96875
2	6	4.5	27	34.1683884	18
$\Sigma$	$16.5 \text{ in}^2$		34.875	53.6931818	19.96875

$$\bar{y} = 2.113636 \text{ in} \quad I_x = 73.66 \text{ in}^4$$

Neutral axis = 2.11 in from the top (A)

So  $y_A = 2.11 \text{ in}$  and  $y_B = (6 + 1.5 - 2.11) = 5.39 \text{ in}$

The stresses at any section of member CD consist of tensile stress ( $\sigma^a$ ) from the truss action and bending stresses ( $\sigma^b$ ) from the bending action, both resulting from the distributed load on this member.

Points A and B are both at points where  $Q = 0$ , so  $\tau_A = \tau_B = VQ/It = 0$

*Bending Stresses,  $\sigma^b$ :*

$$R_C = R_D = 1.8 \times 16 \times 0.5 = 14.4 \text{ Kips}$$

$$M_{A-B} = 14.4 \times 5 - 1.8 \times 5 \times 2.5 = 49.5 \text{ Kip-ft}$$

Top fibres of member CD are in compression (so,  $\sigma_A^b = -ve$ ), while the bottom fibres of are in tension (so,  $\sigma_B^b = -ve$ ).

$$\sigma_A^b = -\frac{My_A}{I} = -\frac{49.5 \times 12 \times 2.11}{73.66} = -17.0152 \text{ Ksi},$$

$$\sigma_B^b = +\frac{My_B}{I} = \frac{49.5 \times 12 \times 5.39}{73.66} = +43.4654 \text{ Ksi}$$

*Axial Stresses,  $\sigma^a$ :*

Let  $\angle CDE = \beta$ , then  $\sin \beta = 0.6$  and  $\cos \beta = 0.8$

Equilibrium of joint D gives

$$F_{DE} = \frac{R_D}{\sin \beta} = \frac{14.4}{0.6} = 24 \text{ Kips}, \quad \text{and}$$

$$F_{CD} = F_{DE} \cos \beta = 24 \times 0.6 = 19.2 \text{ Kips}.$$

$$\sigma_A^a = \sigma_B^a = \sigma^a = \frac{F_{CD}}{A_{CD}} = \frac{19.2 \text{ Kips}}{16.5 \text{ in}^2} = +1.1636 \text{ Ksi (tension)}$$

$$\sigma_A = \sigma_A^b + \sigma_A^a = -17.0152 + 1.1636 = -15.8516 \text{ Ksi} \approx -15.85 \text{ Ksi}$$

$$\sigma_B = \sigma_B^b + \sigma_B^a = 43.4654 + 1.1636 = +44.629 \text{ Ksi} \approx +44.63 \text{ Ksi}$$

Therefore The states of stress

$$\text{at A, } (\sigma_A, \tau_A) = (-15.85 \text{ Ksi}, 0.00 \text{ Ksi}), \quad \text{and}$$

$$\text{at B, } (\sigma_B, \tau_B) = (+44.63 \text{ Ksi}, 0.00 \text{ Ksi})$$

Sketch each one on an element