

Problem #1:

Given:

The beam shown

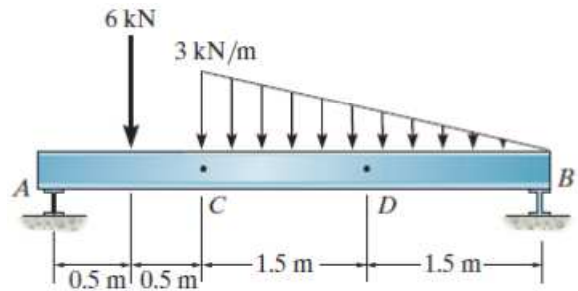
Required:

The internal forces at point E, located at 2 m from A

Solution:

First, we need to calculate the reaction at B. (Why only B?!)

FBD (1) is drawn below.

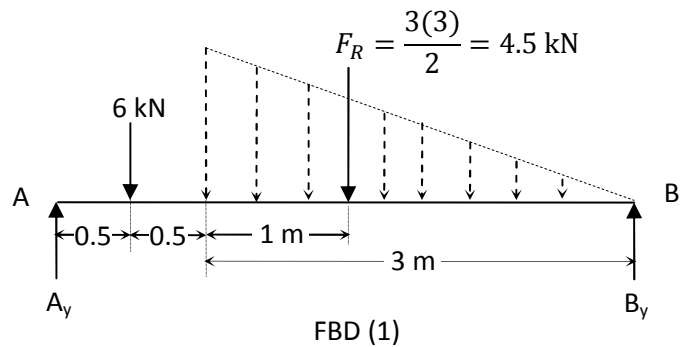


$$\curvearrowright \Sigma M_A = 0$$

(Why ΣM , not ΣF , and why point A?!)

$$\Rightarrow 4B_y - 4.5(2) - 6(0.5) = 0$$

$$\Rightarrow B_y = 3 \text{ kN (As shown)}$$

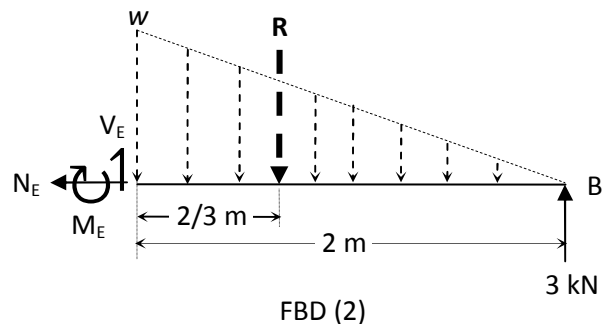


Now, we make a section through E and take the **right** part (Why?!). FBD (2) is drawn.

Note the assumed directions of the internal forces (Why?!)

$$\frac{w}{3} = \frac{2}{3} \Rightarrow w = 2 \text{ kN/m}$$

$$R = \frac{2(2)}{2} = 2 \text{ kN}$$



Note that the equivalent load R was calculated **after** cutting. (Why?!)

$$\rightarrow \Sigma F_x = 0 \Rightarrow \boxed{N_E = 0}$$

$$+\uparrow \Sigma F_y = 0 \Rightarrow V_E - 2 + 3 = 0 \Rightarrow \boxed{V_E = -1 \text{ kN} = 1 \text{ kN} \downarrow}$$

$$\curvearrowright \Sigma M_E = 0$$

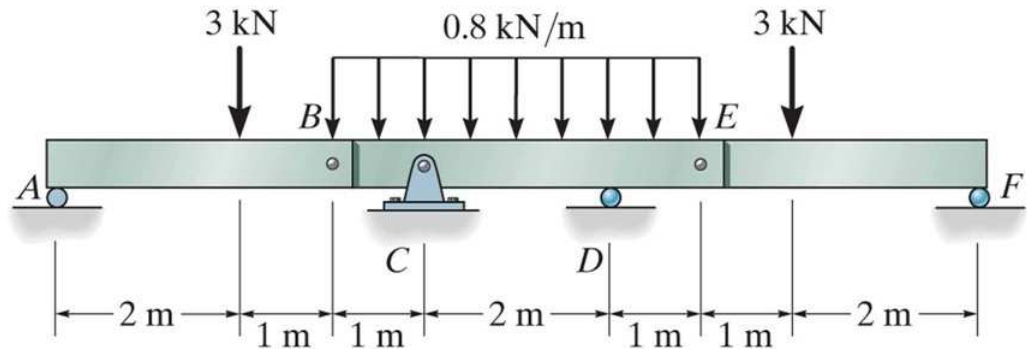
$$\Rightarrow -M_E - 2\left(\frac{2}{3}\right) + 3(2) = 0$$

$$\Rightarrow M_E = 4\frac{2}{3} \text{ kN.m (as shown)}$$

Problem #2:

Given:

The beam shown



Required:

- The reactions
- The internal forces at the center of the beam

Solution:

To be able to find the reactions, we need to separate the beam at B or E. (Why?!)

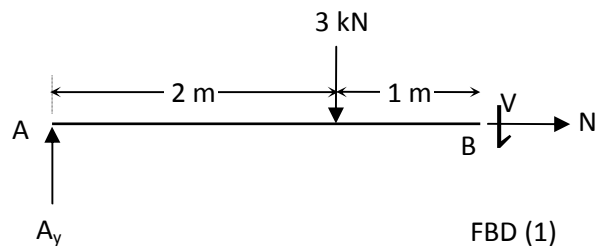
Taking AB, FBD (1) is drawn. Note that there's no moment at B. (Why?!).

$$\sum M_B = 0$$

(Why not taking $\sum F_y = 0$?!)

$$\Rightarrow 3(1) - A_y(3) = 0$$

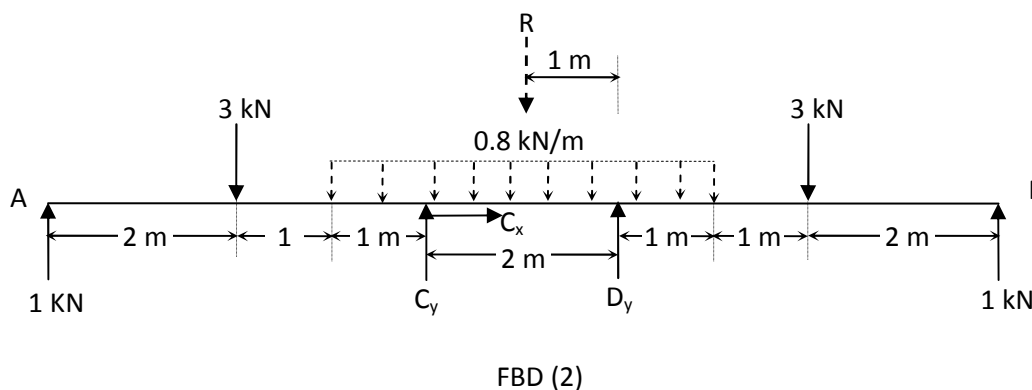
$$\Rightarrow \boxed{A_y = 1 \text{ kN } \uparrow \text{ (as shown)}}$$



Due to symmetry (in geometry, loads, supports, etc),

$$\boxed{F_y = A_y = 1 \text{ kN } \uparrow}$$

Now, we take FBD (2) for the whole beam.



$$\rightarrow \Sigma F_x = 0 \Rightarrow \boxed{C_x = 0}$$

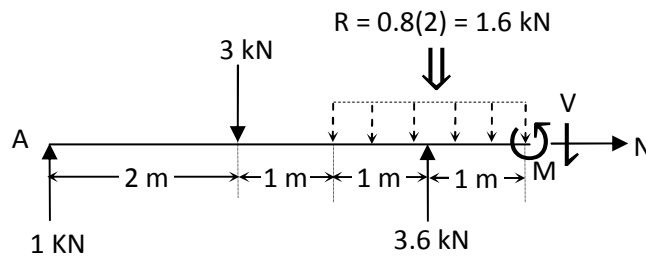
$$\curvearrowright \Sigma M_D = 0$$

$$\Rightarrow -1(6) + 3(4) - 2C_y + 0.8(4)(1) - 3(2) + 1(4) = 0$$

$$\Rightarrow \boxed{C_y = 3.6 \text{ kN } \uparrow \text{ (as shown)}}$$

By symmetry, $\boxed{D_y = C_y = 3.6 \text{ kN } \uparrow}$

To determine the internal forces at the beam center, a cut (section) is made through that point, and FBD (3) (the left part) is drawn.



FBD (3)

$$\rightarrow \Sigma F_x = 0 \Rightarrow \boxed{N = 0}$$

$$+\uparrow \Sigma F_y = 0 \Rightarrow 1 - 3 - 1.6 + 3.6 - V = 0 \Rightarrow \boxed{V = 0 \text{ kN}}$$

This is expected, as it is the value of the shear in the middle/center of a “symmetrical beam”. (Why and how?!)

$$\curvearrowright \Sigma M = 0$$

$$\Rightarrow M - 1(5) + 3(3) + 1.6(1) - 3.6(1) = 0$$

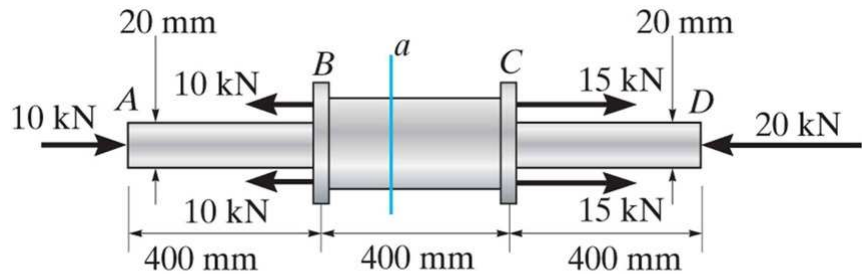
$$\Rightarrow \boxed{M = -2 \text{ kN}\cdot\text{m} = 2 \text{ kN}\cdot\text{m } \curvearrowright}$$

Problem #3:

Given:

The figure shown

$$D_{AB} = D_{CD} = 20 \text{ mm}; D_{BC} = 40 \text{ mm}$$



Required:

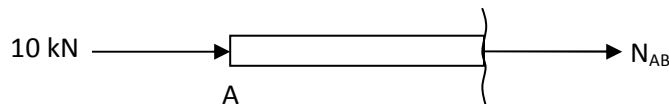
σ in AB, BC, and CD

Solution:

$$\sigma = \frac{P}{A} = \frac{N}{A}$$

N is the **internal** normal force, obtained by passing a cut (section) through the part of interest.

A section through each of AB, BC, and CD is made, and the "easier" part (left or right) is taken and an FBD is drawn, as shown below.



(N_A may be assumed in the other direction \leftarrow , i.e., compression.)

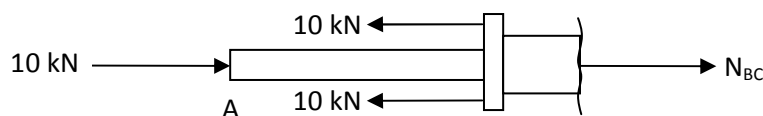
$$\overset{+}{\rightarrow} \Sigma F_x = 0 \Rightarrow 10 + N_{AB} = 0 \Rightarrow$$

$$N_{AB} = -10 \text{ kN} = 10 \text{ kN} \leftarrow = 10 \text{ kN "C"}$$

$$\sigma_{AB} = \left(\frac{N}{A} \right)_{AB} = \frac{-10(10)^3}{\pi/4 (20)^2} \Rightarrow$$

$$\sigma_{AB} = -31.83 \text{ MPa} = 31.83 \text{ MPa "C"}$$

*Be careful about the units (N , m , mm , Pa , MPa , ...)



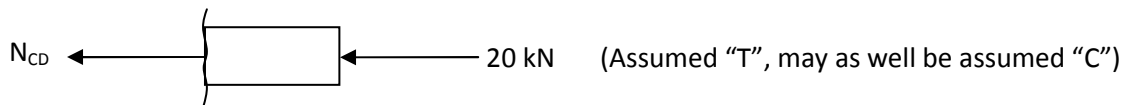
Solution of HW # 1

$$\xrightarrow{+} \Sigma F_x = 0 \Rightarrow 10 - 10 - 10 + N_{BC} = 0 \Rightarrow$$

$$N_{BC} = 10 \text{ kN "T"}$$

$$\sigma_{BC} = \left(\frac{N}{A}\right)_{BC} = \frac{10(10)^3}{\pi/4 (40)^2} \Rightarrow$$

$$\boxed{\sigma_{BC} = 7.958 \text{ MPa "T"}}$$



$$\xrightarrow{+} \Sigma F_x = 0 \Rightarrow -20 - N_{CD} = 0 \Rightarrow$$

$$N_{CD} = -20 \text{ kN} = 20 \text{ kN "C"}$$

$$\sigma_{CD} = \left(\frac{N}{A}\right)_{CD} = \frac{-20(10)^3}{\pi/4 (20)^2} \Rightarrow$$

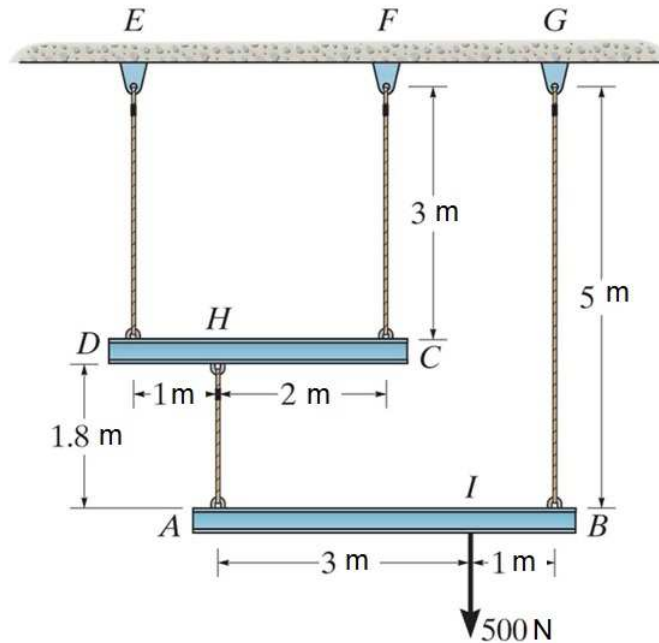
$$\boxed{\sigma_{CD} = -63.66 \text{ MPa} = 63.66 \text{ MPa "C"}}$$

Problem #4:

Given:

The figure shown

$$A_{cable} = 20 \text{ mm}^2$$



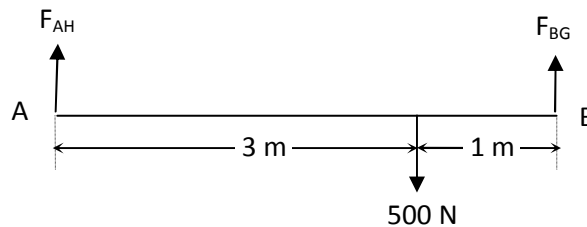
Required:

σ in the cables.

Solution:

In order to determine the stresses in the cables, we need to find the internal normal forces in these cables as $\sigma = \frac{N}{A}$ where N is the **internal** force.

FBD (1) is drawn for AB. (Why?!)



FBD (1)

$$\sum M_B = 0 \text{ (Why?!)}$$

$$\Rightarrow 500(1) - F_{AH}(4) = 0$$

$$\Rightarrow F_{AH} = 125 \text{ N "T"}$$

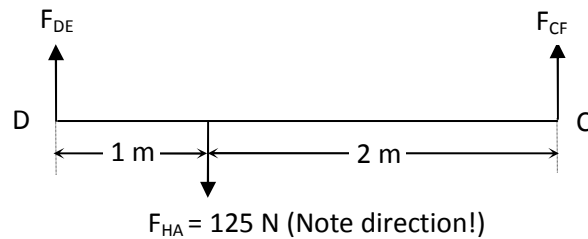
Solution of HW # 1

$$+\uparrow \Sigma F_y = 0 \Rightarrow 125 - 500 + F_{BG} = 0$$

$$\Rightarrow F_{BG} = 375 \text{ N "T"}$$

Now, FBD (2) for DC is drawn.

(Another FBD for both AB and DC can be drawn. Why and how?!)



FBD (2)

$$\curvearrowright \Sigma M_C = 0$$

$$\Rightarrow 125(2) - F_{DE}(3) = 0 \Rightarrow F_{DE} = 83.333 \text{ N "T"}$$

$$+\uparrow \Sigma F_y = 0 \Rightarrow 83.3333 - 125 + F_{CF} = 0$$

$$\Rightarrow F_{CF} = 41.667 \text{ N "T"}$$

$$\sigma_{AH} = \frac{F_{AH}}{A} = \frac{125}{20} \Rightarrow \boxed{\sigma_{AH} = 6.25 \text{ MPa "T"}}$$

$$\sigma_{BG} = \frac{F_{BG}}{A} = \frac{375}{20} \Rightarrow \boxed{\sigma_{BG} = 18.75 \text{ MPa "T"}}$$

$$\sigma_{DE} = \frac{F_{DE}}{A} = \frac{83.3333}{20} \Rightarrow \boxed{\sigma_{DE} = 4.167 \text{ MPa "T"}}$$

$$\sigma_{CF} = \frac{F_{CF}}{A} = \frac{41.667}{20} \Rightarrow \boxed{\sigma_{CF} = 2.083 \text{ MPa "T"}}$$