

CE 203-3 [072]

H. W. #12 solution

Problem 1:-

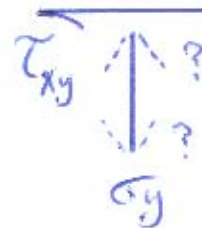
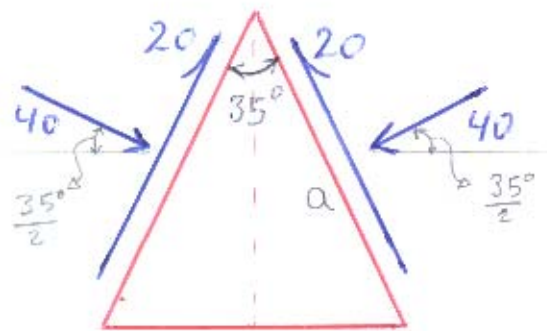
Given:- For the state of stress shown in Fig.

Req:- σ_y , τ_{xy} , σ_{max} , σ_{min} , and τ_{max}

Solution:-

Considering the equilibrium of the element, the inclined side of the triangle has a value $= \alpha$.

Note that equilibrium of forces not strains is taken



all in mpa

$$\uparrow \sum F_v = 0$$

$$-\sigma_y (2\alpha \sin \frac{35}{2}) - 2 \times 40 \times \alpha \sin \frac{35}{2} + 2 \times 20 \times \alpha \cos \frac{35}{2} = 0$$

$$\Rightarrow \sigma_y = -40 + 20 \frac{\cos 17.5}{\sin 17.5} \Rightarrow \sigma_y = 23.43 \text{ Mpa (tension)}$$

$$\sum F_H = 0, \Rightarrow \tau_{xy} = 0$$

Since $\tau_{xy} = 0$, $\Rightarrow \sigma_y$ is a principal stress.

We get σ_x which is the other principal stress by considering the equilibrium of the following element

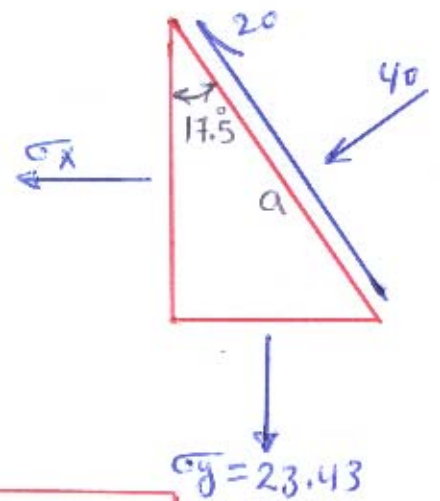
$$\leftarrow \sum F_H = 0 ;$$

$$\sigma_x (a \cos 17.5) + 20 a \sin 17.5 + 40 a \cos 17.5 = 0$$

$$\Rightarrow \sigma_x = -40 - 20 \frac{\sin 17.5}{\cos 17.5}$$

$$\therefore \sigma_x = -46.31 \text{ Mpa}$$

$$\therefore \sigma_{\max} = 23.43 \text{ Mpa} , \sigma_{\min} = -46.31 \text{ Mpa}$$



$$\tau_{\max/\min} = \pm \left(\frac{\sigma_{\max} - \sigma_{\min}}{2} \right) = \pm \left(\frac{23.43 + 46.31}{2} \right)$$

$$\tau_{\max/\min} = \pm 34.87 \text{ Mpa.}$$

$$\therefore \tau_{\max} = 34.87 \text{ Mpa.}$$

$$\tau_{\min} = -34.87 \text{ Mpa.}$$

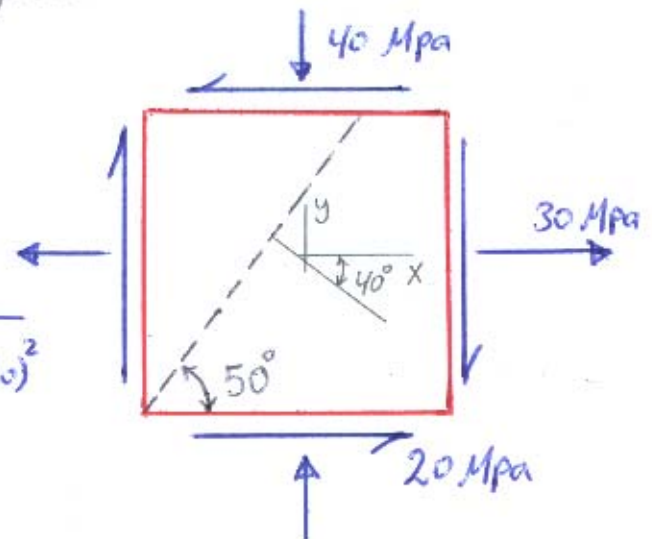
Problem 2:-

Given:- For the state of stress shown in fig.
Use the equations

- Req :-
- The magnitude and direction of the principal normal stresses; show them on a properly oriented element.
 - The magnitude and direction of the maximum (principal) shear stresses; show them on a properly oriented element.
 - The normal and shear stresses on the plane indicated by the dashed line; show them on a properly oriented element.

Solution :-

$$\begin{aligned} \text{a) } \sigma_{\max/\min} &= \frac{\sigma_x + \sigma_y}{2} \pm \sqrt{\left(\frac{\sigma_x - \sigma_y}{2}\right)^2 + \tau_{xy}^2} \\ &= \frac{30 - 40}{2} \pm \sqrt{\left(\frac{30 + 40}{2}\right)^2 + (-20)^2} \\ &= -5 \pm 40.3 \end{aligned}$$



$$\Rightarrow \boxed{\sigma_{\max} = 35.3 \text{ MPa}} \quad ; \quad \boxed{\sigma_{\min} = -45.3 \text{ MPa}}$$

$$\tan 2\theta_p = \frac{2\tau_{xy}}{\sigma_x - \sigma_y} = \frac{2(-20)}{30 + 40} = -0.5714$$

$$2\theta_{p1} = 330.26^\circ \quad (-29.74^\circ)$$

$$2\theta_{p2} = 150.25^\circ$$

$$\therefore \boxed{\theta_{p1} = 165^\circ} \quad ; \quad \boxed{\theta_{p2} = 75.1^\circ}$$

* To check θ_p , corresponds to which of the σ_{max} or σ_{min}

$$\begin{aligned}\sigma_{\xi} &= \frac{\sigma_x + \sigma_y}{2} + \frac{\sigma_x - \sigma_y}{2} \cos 2\theta_p + \tau_{xy} \sin 2\theta_p \\ &= \frac{30 - 40}{2} + \frac{30 + 40}{2} \cos(330.26) + (-20) \sin(330.26) \\ &= 35.3 = \sigma_{max} \quad \text{ok.}\end{aligned}$$

$$\begin{aligned}b) \tau_{\max/\min} &= \pm \sqrt{\left(\frac{\sigma_x - \sigma_y}{2}\right)^2 + \tau_{xy}^2} = \pm \sqrt{\left(\frac{30 + 40}{2}\right)^2 + (-20)^2} \\ &= \pm 40.3 \text{ Mpa}\end{aligned}$$

$$\therefore \tau_{\max} = 40.3 \text{ Mpa} \quad ; \quad \tau_{\min} = -40.3 \text{ Mpa.}$$

For this case of τ_{\max} and τ_{\min}

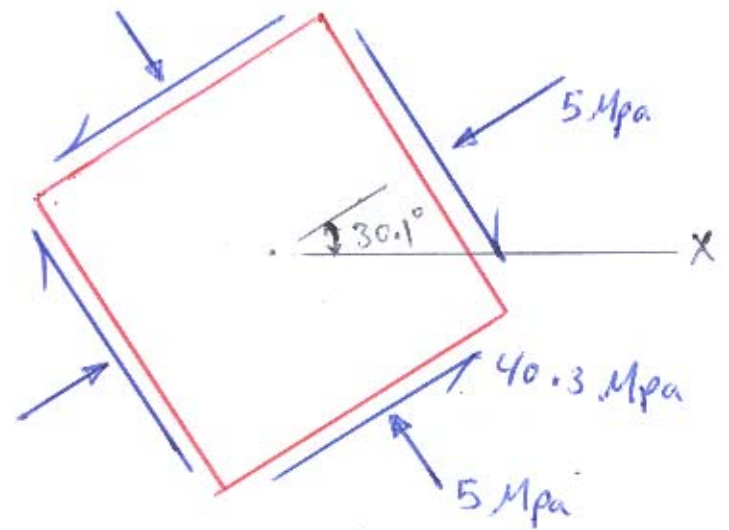
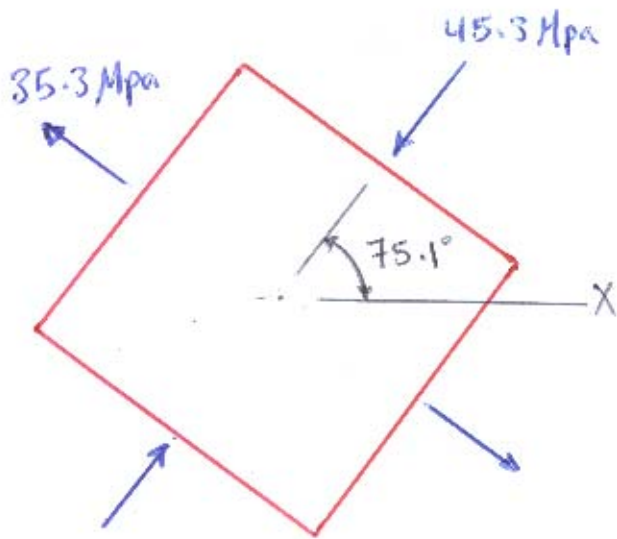
$$\sigma_{\xi} = \sigma_{\eta} = \frac{\sigma_x + \sigma_y}{2} = \frac{30 - 40}{2} = -5 \text{ Mpa.}$$

$$\tan 2\theta_s = -\frac{1}{\tan 2\theta_p} = -\frac{1}{(-0.5714)} = 1.75$$

$$\therefore \theta_{s1} = 30.13^\circ \quad ; \quad \theta_{s2} = 120.1^\circ$$

* To check θ_{s1} corresponds to τ_{\max} or τ_{\min}

$$\begin{aligned}\tau_{\xi\eta} &= -\frac{\sigma_x - \sigma_y}{2} \sin 2\theta_{s1} + \tau_{xy} \cos \theta_{s1} \\ &= -\frac{30 + 40}{2} \sin(30.13) + (-20) \cos(30.13) \\ &= -40.3 \text{ Mpa} \quad (\tau_{\min})\end{aligned}$$



a) Principal normal stresses

b) Principal shear stresses

c) At 50° plane (-40° from x-axis)

$$\sigma_x(-40^\circ) = \frac{30-40}{2} + \frac{30+40}{2} \cos(2\alpha-40^\circ) + (-20) \sin(2\alpha-40^\circ)$$

$$\therefore \sigma_x = 20.77 \text{ Mpa.}$$

$$\sigma_y(-40^\circ) = \frac{30-40}{2} - \frac{30+40}{2} \cos(2\alpha-40^\circ) - (-20) \sin(2\alpha-40^\circ)$$

$$\therefore \sigma_y = -30.77 \text{ Mpa.}$$

$$\tau_{xy}(-40^\circ) = -\frac{30+40}{2} \sin(2\alpha-40^\circ) + (-20) \cos(2\alpha-40^\circ)$$

$$\therefore \tau_{xy} = 31.00 \text{ Mpa}$$

$$\therefore \sigma_x = 20.8 \text{ Mpa} ; \sigma_y = 30.8 \text{ Mpa} ; \tau_{xy} = 31.0 \text{ Mpa}$$

$$\therefore \theta_{p1} = 16.5^\circ \text{ ccw } (\uparrow) \text{ measured from } x\text{-axis} \\ \text{to axis of } \sigma_{\max}$$

$$2\theta_{p2} = 180^\circ - 2\alpha \Rightarrow \theta_{p2} = 75.1^\circ \text{ ccw } (\uparrow) \text{ from } x\text{-axis}$$

$$R = Cx = \frac{KE}{\sin 2\alpha} = \frac{20}{\sin(29.74)} = 40.3 \text{ Mpa.}$$

$$\sigma_{\max} = R - oc = 40.3 - 5 \Rightarrow \sigma_{\max} = 35.3 \text{ Mpa.}$$

$$\sigma_{\min} = -(R + oc) = -(40.3 + 5) \Rightarrow \sigma_{\min} = -45.3 \text{ Mpa}$$

$$b) \tau_{\max/\min} = \pm R \Rightarrow \tau_{\max} = 40.3 \text{ Mpa} \\ \tau_{\min} = -40.3 \text{ Mpa}$$

$$2\theta_{s1} = 2\theta_{p1} - 90^\circ \Rightarrow \theta_{s1} = 120.1^\circ \text{ ccw from } x\text{-axis} \\ 2\theta_{s2} = 2\theta_{p2} - 90^\circ \Rightarrow \theta_{s2} = 30.13^\circ \text{ ccw from } x\text{-axis}$$

$$c) \sigma_{xy} = cG - oc = 40.3 \cos(80 - 20c) - 5$$

$$\sigma_{xy} = 20.77 \text{ Mpa}$$

$$\sigma_{\eta} = -(oc + cd) = -\{5 + 40.3 \cos(80 - 2\alpha)\}$$

$$\sigma_{\eta} = -30.77 \text{ Mpa}$$

$$\tau_{xy\eta} = \pm R \sin(80 - 2\alpha) = \tau_{xy\eta} = \pm 40.3 \sin(80 - 2\alpha)$$

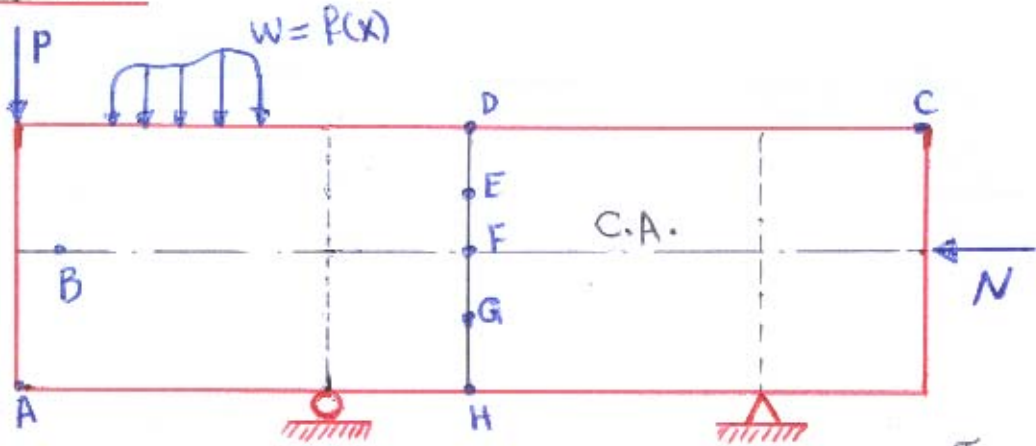
$$\tau_{xy\eta} = 31.0 \text{ Mpa.}$$

Problem 4:-

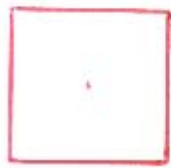
Given:- The beam shown in Fig.

Req:- Qualitatively, sketch the state of stress and Mohr's circle for points A to H

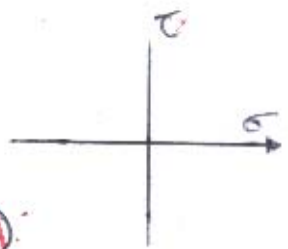
Solution:-



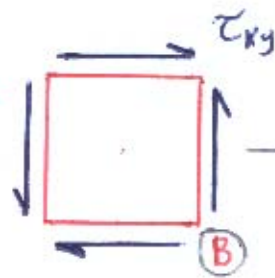
2.5



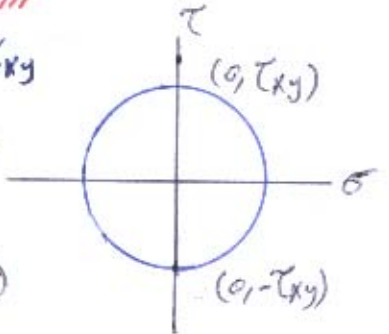
A



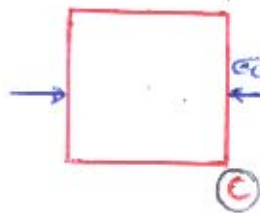
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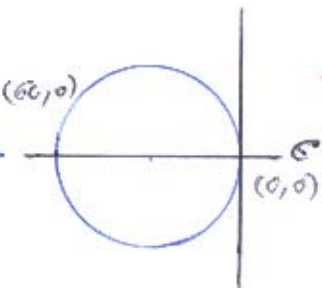
B



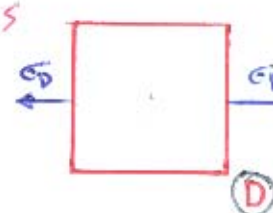
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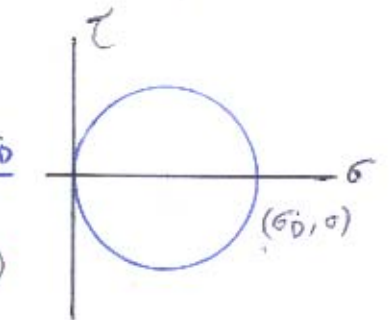
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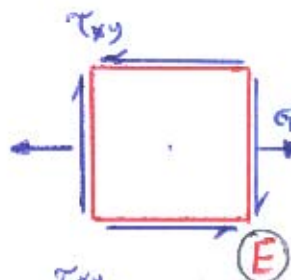
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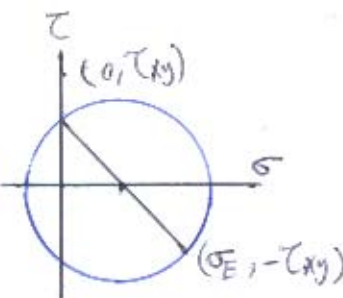
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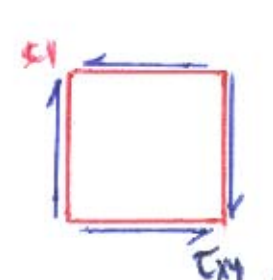
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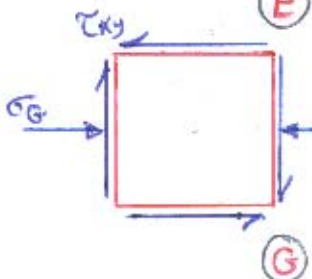
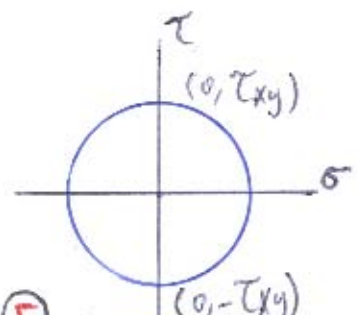
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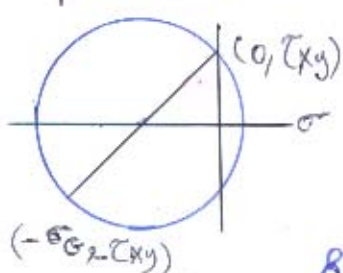
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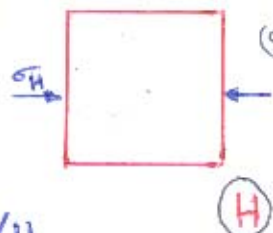
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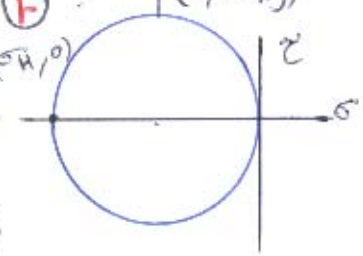
G



8/11



H



$$\tau_{c.a.} = \frac{VQ}{Ib} = \frac{26 \times 10^3 (100 \times \frac{125^2}{2} + 2 \times 50 \times \frac{25^2}{2})}{131.3 \times 10^6 (200)} = 0.804 \text{ Mpa}$$

$$\tau_i = \frac{26 \times 10^3 (100 \times 100 \times (\frac{100}{2} + 25))}{131.3 \times 10^6 (100)} = 1.485 \text{ Mpa.}$$

At point **C** maximum normal stress is at top and bottom

Top:- $\sigma_x = 53.18 \text{ Mpa}$

$$\sigma_y = 0$$

$$\tau_{xy} = 0$$

For this using Mohr's circle

$$\therefore \sigma_{max} = 53.18 \text{ Mpa} \quad ; \quad \sigma_{min} = 0$$

$$\tau_{max} = 26.59 \text{ Mpa} \quad ; \quad \tau_{min} = -26.59 \text{ Mpa.}$$

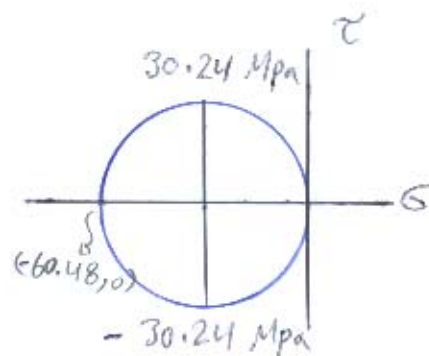
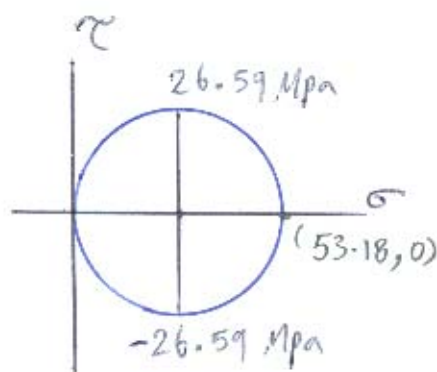
Bottom:- $\sigma_x = -60.48 \text{ Mpa}$

$$\sigma_y = 0$$

$$\tau_{xy} = 0$$

$$\therefore \sigma_{max} = 0 \quad ; \quad \sigma_{min} = -60.48 \text{ Mpa.}$$

$$\tau_{max} = 30.24 \text{ Mpa} \quad ; \quad \tau_{min} = -30.24 \text{ Mpa.}$$



$$V_{max} = 32 \text{ kN @ B.}$$

$$M = 32 \text{ kN}\cdot\text{m} = 32 \times 10^6 \text{ N}\cdot\text{mm} \quad \left(\overset{T}{\curvearrowright} \right) \text{ @ B.}$$

$$\sigma = \sigma_N \pm \sigma_M = -\frac{100 \times 10^3}{30 \times 10^3} \pm \frac{32 \times 10^6 y}{131.3 \times 10^6}$$

$$\Rightarrow \sigma = -3.333 \pm 0.2414 y$$

$$= -3.333 \pm 30.46$$

$$\sigma_{top} = -3.333 + 30.46 = 27.13 \text{ Mpa}$$

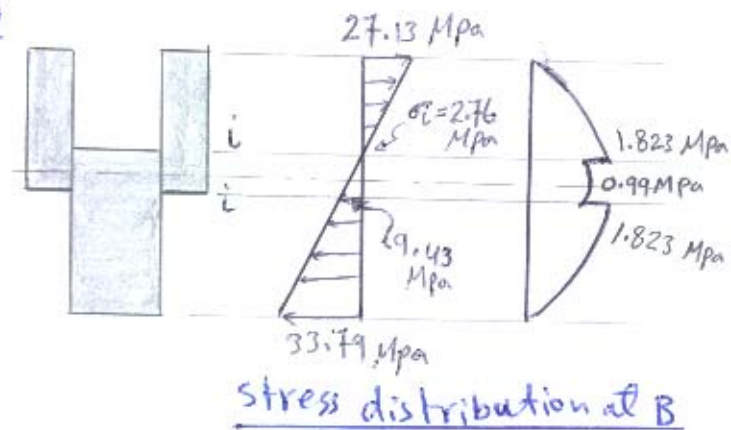
$$\sigma_{bottom} = -3.333 - 30.46 = -33.79 \text{ Mpa.}$$

$$\tau_{c,A} = \frac{32 \times 10^3 (100 \times 125^2/2 + 2 \times 50 \times 25^2/2)}{131.3 \times 10^6 (200)}$$

$$= 0.99 \text{ Mpa.}$$

$$\tau_i = \frac{32 \times 10^3 (100 \times 100 \times (100/2 + 25))}{131.3 \times 10^6 (100)}$$

$$= 1.823 \text{ Mpa}$$



τ is maximum at lower i-face at B.

$$\sigma_x = -9.43 \text{ Mpa}, \quad \sigma_y = 0; \quad \tau_{xy} = 1.823 \text{ Mpa.}$$

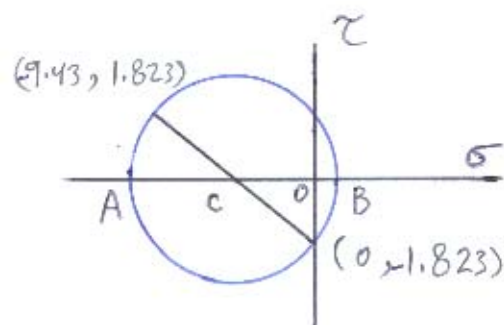
$$\Rightarrow \sigma_{max} = 0.340 \text{ Mpa}, \quad \sigma_{min} = -9.77 \text{ Mpa.}$$

$$\tau_{max} = 5.06 \text{ Mpa}, \quad \tau_{min} = -5.06 \text{ Mpa.}$$

$$\sigma_{max}^T = 53.18 \text{ Mpa}$$

$$\sigma_{max}^c = 60.48 \text{ Mpa}$$

$$\tau_{max} = 30.24 \text{ Mpa}$$



$$10c = \frac{9.43 + 0}{2} = 4.72 \text{ Mpa}$$

$$R = \sqrt{(9.43 - 4.72)^2 + 1.823^2}$$

$$= 5.06 \text{ Mpa.}$$